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THESIS

AN INTERACTIVE COMPUTER PROGRAM
FOR THE PRELIMINARY DESIGN AND
ANALYSIS OF MARINE REDUCTION GEARS

by

Joseph Louis Paquette

March, 1982

Thesis Advisor:

G. Cantin

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An Interactive Computer Program
for the Preliminary Design and
Analysis of Marine Reduction Gears

by

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Lieutenant, United States Navy
B.S.S.E., United States Naval Academy, 1976

Submitted in partial fulfillment of the
requirements for the degree of

MASTER OF SCIENCE IN MECHANICAL ENGINEERING

from the

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March, 1982

ABSTRACT

The objective of this project was to develop an interactive computer program providing flexibility in the design and analysis of marine propulsion gears. The program, Reduction Gear Analysis and Design (REGAD), will handle conventional parallel axis and simple epicyclic reduction gears. It is capable of generating preliminary designs of new gear sets or providing analyses of existing or proposed gear sets. Program development, organization, and operation are discussed.

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I. INTRODUCTION

In the conceptual stage of ship design, many parameters and options are considered. This is especially true with respect to the propulsion plant. Changes in hull design, displacement, and numbers of propellers all affect changes in the requirements for the propulsion plant. There are also various options under consideration in the propulsion plant: turbines or internal combustion engines, the number of engines, the auxiliaries required for support, etc. All of these will affect the initial design of the reduction gears. It is, therefore, important to be able to produce preliminary designs of reduction gears for the options under consideration.

Preliminary designs provide useful information on feasible arrangements and size without going into the specific design details dependent upon manufacturing. Since any number of preliminary designs may be required due to perturbations discussed above, it is important to be able to automate the design process. An interactive computer program providing various options would free the engineer from tedious, time consuming, and often error prone number manipulation and allow him to produce multiple designs for consideration. It would also provide a quick means of checking

the effects of various parameters in addition to the ability to analyze proposed designs or configurations.

Reduction Gear Analysis and Design (REGAD) was developed to fill this need. It is an interactive computer program offering close user control through numerous options. Being interactive, it provides a rapid means of designing or analyzing a gear set, thereby reducing the turn-around time inherent in the use of batch systems. The program was kept modularized and well documented for ease of maintenance and modification. The modularized construction also provides an additional benefit of being able to use this program on smaller computers by using an overlay scheme.

II. PROGRAM CAPABILITIES

A. SCOPE

REGAD was written to provide preliminary designs or analyses of marine propulsion reduction gears. It is incapable of providing detailed designs or performing detailed analyses since specifics of manufacturing are not required for input. The program does not consider shafting, bearings, lubrication, couplings, casings, or other auxiliaries. It will provide sizing information in the form of pitch diameters, effective facewidths, gear ratios, and numbers of teeth per gear. In addition, the program will provide estimates of loadings and stress levels. Estimated weight and dimensions of the gear set are also provided.

All computations are based primarily on the American Gear Manufacturers Association's standards [Ref. 1, 2, 3] using appropriate constants for marine propulsion gears [Ref. 4, 5]. As an option in the program, these constants can be replaced by the user to enable him to investigate other applications such as reduction gears for ships service or emergency generators.

B. LIMITATIONS AND OPTIONS

Program application is limited to marine reduction gears with a maximum of three reduction stages. Conventional parallel axis and simple epicyclic arrangements with helical gears are possible. When dealing with epicyclics, it is assumed that load sharing of the planets is achieved and that the ring gear is suitably flexible. Efficiencies of the gear sets are not provided since power losses are not computed. While estimates of bending and contact stresses are provided, scoring can not be estimated since lubrication is not considered. REGAD does not require the K-factor as input as in previous programs since hardness ranges for the pinions and gears are required. However, the K-factors are computed and displayed for reference purposes. The weight estimates are based on actual designs and do not include turning gears, attached lubrication oil pumps, or other auxiliaries.

The following is a list of major options provided by the program:

- (1) brief, on-line program description
- (2) choice of design or analysis
- (3) listing of preprogrammed constants and an ability to change selected constants of the user's choice
- (4) choice of single, double, or triple reductions
- (5) choice of single or double helical gears
- (6) choice of six hardness ranges for gears and pinions

- (7) conventional parallel axis arrangements (see Figures 1, 2, and 3)
 - (a) one or two power inputs
 - (b) single power path (articulated) or dual power paths (locked train)
- (8) simple epicyclic arrangements (see Figure 4)
 - (a) choice of planetary or star arrangements
 - (b) single power input
 - (c) choice of three, four, or five planet/star gears.

III. PROGRAM ORGANIZATION AND OPERATION

A. REGAD FLOWPATHS

As stated previously, the program was designed in modular form with each module consisting of a number of subprograms. These modules are just conceptual groupings of associated subprograms, and are not related to actual program implementation on any specific computer. Figure 5 shows the basic flow paths of the program. Module One is for program initialization and problem set up. Module Two performs calculations for conventional parallel axis gear sets, while Module Three handles epicyclic gear sets. Module Four is a grouping of all the computational subprograms required by the other modules.

B. MODULE IDENTIFICATION AND DESCRIPTION

This section provides a brief description of each subprogram in each module.

1. Module One : Initialization and Set-up

Module One contains the subprograms necessary for initialization, execution, and initial data entry. It is, basically, the control module for the program. The following is a grouping of the subprograms in Module One.

a. REGAD

REGAD is the main program. It provides the options for either design or analysis and either parallel axis or epicyclic arrangements and controls the flow to the proper module. It then calls the required subprograms for execution of Modules One, and Two or Three.

b. BLOCK DATA

The BLOCK DATA subprogram initializes variables in each of the common blocks.

c. SUBROUTINE DSCRPT

This subroutine is called by REGAD after an affirmative response to a user option to provide a brief description of the REGAD package. It contains an option to stop the program if only a program description is desired.

d. SUBROUTINE INPUT

All options and initial design parameters are entered via this subroutine which is called by REGAD.

e. SUBROUTINE AGMA

The constants for marine propulsion gears required by various AGMA formulations are initialized in the BLOCK DATA subprogram, and can be listed as an option in REGAD. REGAD calls this subroutine after an affirmative user response to display the preprogrammed values. This subroutine then allows the user to selectively change any constant desired.

2. Module Two : Parallel Axis

This module contains all the major subprograms called by REGAD to provide an initial design or to perform an analysis of conventional parallel axis reduction gears. The following is a grouping of the subprograms in Module Two.

a. SUBROUTINE PRLDES

This subroutine will produce a design of a parallel axis gear set. All pinion and gear diameters, effective facewidths, and gear ratios are computed using a basic random search optimization technique to find a feasible design by attempting to minimize a function of gear pair volume. It should be noted that, while attempting to minimize gear volume, the design is not necessarily optimized for minimum weight. The optimization technique is used here only to produce a feasible design in terms of dimension and power constraints by minimizing a function of gear pair volume. To produce a truly optimized design for minimum weight, a full optimization must include many more design variables such as helix and pressure angles, pitches, and hardnesses in addition to the dimensions. Additional constraints such as stress and unit load levels would need to be incorporated. All of this would require a more sophisticated and efficient optimization technique than is used here.

b. SUBROUTINE PRLANL

To analyze a proposed or existing design, REGAD will call this subroutine. It will request, as user-supplied input, the basic information calculated in PRLDES, i.e., pitch diameters and effective facewidths. Using this information, PRLANL will compute other parameters such as gear ratios, power and speed splits, and numbers of teeth per gear.

c. SUBROUTINE PRLRES

Immediately following a call to PRLDES or PRLANL, REGAD will call PRLRES to compute all remaining information such as expected loadings and stress levels. The user should be aware that the stress levels are computed according to AGMA formulations [Ref. 2 and 3] and take into account load distribution and overloads. This will produce levels that may seem high but are actually closer to actual levels to be expected in service.

d. SUBROUTINE PRLSIZ

REGAD calls this subroutine after PRLRES to compute estimates of gear set weight and gearbox dimensions. These estimates are determined by empirical relationships obtained from a rather limited data base of actual designs.

e. SUBROUTINE PRLOUT

This is the last subroutine called by REGAD in the parallel axis path. It provides a detailed output of the results obtained from the design or analysis including design parameters entered by the user, the dimensions of each component, expected loadings and stress levels, and configuration information.

3. Module Three : Epicyclic

Module Three contains all the major subprograms called by REGAD to design or analyze simple epicyclic reduction gears. Subroutines EPCDES, EPCANL, EPCRES, EPCSIZ, and EPCOUT are all analogous to those in Module Two. They perform the same functions, but for simple epicyclic gears. Therefore, individual descriptions will not be repeated here.

4. Module Four : Computational Subprogram Library

This module is an organizational grouping of all the subprograms called by those in Modules One, Two, and Three.

a. Subroutine Subprograms

The following are the subroutines used:

- (1) GFI - subroutine to compute the AGMA durability geometry factor, I
- (2) GFJ - subroutine to compute the AGMA strength geometry factor, J.

b. Real Function Subprograms

The following are the real function subprograms used:

- (1) ARCCOS - computes the arc cosine of two arguments
- (2) ARCSIN - computes the arc sine of two arguments
- (3) AGMAE1 - uses LaGrangian interpolation of Table E-1 [Ref. 1] to compute the constants required for the stress concentration factor formulation
- (4) CKDATA - called by SUBROUTINE AGMA to allow the user to change the preprogrammed constants
- (5) POWERB - computes allowable service power based on AGMA strength rating [Ref. 3]
- (6) POWERH - computes allowable service power based on AGMA durability rating [Ref. 2]
- (7) RTFNDR - a modified version of FUNCTION ZEROIN [Ref. 6] used to find a zero of a function in a specified interval
- (8) FALFA - the function required by SUBROUTINE GFJ and the zero of which is computed in FUNCTION RTFNDR
- (9) SHRLD - computes the load sharing ratio, m_N

- (10) THICK - computes tooth thickness at any diameter given a known thickness at a different diameter.

C. DATA TRANSFER

All data transfer between subprograms in Modules One, Two, and Three is via combinations of seven common blocks. Data transfer to and from subprograms in Module Four is via argument lists and common blocks as required. The following is a list of the common blocks used:

- (1) /AGMAB/ : constants for AGMA strength formulations
- (2) /AGMAH/ : constants for AGMA durability formulations
- (3) /DESDAT/ : design parameters and options
- (4) /DESPRL/ : parallel axis design information
- (5) /RESPRL/ : parallel axis computational results
- (6) /DESEPC/ : epicyclic design information
- (7) /RESEPC/ : epicyclic computational results.

The variables in each common block along with their definitions can be found in Appendix B.

D. PROGRAM OPERATION

REGAD is an interactive program designed to allow the user to solve his problem at a terminal. Being interactive, the program has many options that control program execution, in addition to requests for data necessary for the execution

of the program. Each request for information will contain the necessary guidelines needed by the the user to respond. This may take the form of a mini-table containing information on each option choice, the range of values when a specific quantity is requested, or units, where applicable, of the requested data.

All option parameters are integer values and should not be entered with a decimal. Option codes entered by the user are checked for validity to ensure they fall within the allowed range. If two options are offered, enter a 1 or a 2. Any value entered less than one will automatically default to one, and any value greater than two will automatically default to two. In those cases where there are more than two options, the response is checked to see if it falls within the allowed range. If it does not, a message alerts the user to this fact and allows him to re-enter the correct code. Some questions require affirmative or negative responses. To reply, use a Y for yes or an N for no. Use of other values may give undesirable results.

Every attempt has been made to anticipate possible error conditions. If one of these is encountered, a message is generated to inform the user. If the error encountered is a terminal error, the message will also indicate that the program run was aborted under program control.

A detailed development of this package is provided in Appendix A where specifics can be found. Appendix B provides a cross-reference of the variables used in Appendix A with those used in the program. It also contains detailed information on the common blocks. Sample runs of the program can be found in Appendix C, and a complete listing of the program is in Appendix D.

IV. CONCLUSIONS AND RECOMMENDATIONS

Computer aided design (CAD) is an important and useful tool for engineers. As computer technology continues to expand, CAD will become increasingly available for the practicing engineer, allowing him to use his initiative in design instead of being a slave to the numbers involved. REGAD is such a tool for use in the preliminary design of marine reduction gears during the conceptual stages of propulsion plant design.

REGAD could become even more useful if additional options are provided. A module to perform sensitivity analyses of a given design would greatly enhance the use of this program. This option would allow the user to start with any design and vary a selected variable over a specified range to determine its impact on the design. It could also be used to "fine tune" a design by modifying selected parameters to produce the results desired without having to rerun the program for each modification. Graphics would add another dimension by providing graphical displays of the gear arrangements and of certain data such as the results of a sensitivity analysis. A module to handle various composite designs of parallel axis and epicyclic gears would be an important addition. Also, it is recommended that a larger

data base be collected to provide more accurate empirical constants for the weight and gearbox size estimates.

APPENDIX A

PROGRAM DEVELOPMENT

With the exception of several general conversion relationships, all computations are accomplished in Modules Two and Three with calls to subprograms in Module Four. The analytical relationships used in the program will be examined, however, most of the relations used can be easily found in the literature and in various texts, so background developments will not be given.

I. GENERAL RELATIONSHIPS

The following relationships are used in Module One and in various other subprograms. The transverse diametral pitch of any gear is the ratio of its number of teeth to its pitch diameter;

$$P_d = \frac{N}{d} \quad (1)$$

The normal and transverse diametral pitches are related by;

$$P_d = P_{nd} \cos \psi \quad (2)$$

and the pressure angles by;

$$\tan \Phi_n = \tan \Phi_t \cos \psi \quad (3)$$

Axial pitch is defined as;

$$p_x = \frac{\pi}{P_{nd} \sin \psi} = \frac{\pi}{P_d \tan \psi} \quad (4)$$

II. CONVENTIONAL PARALLEL AXIS FORMULATIONS

Subroutines PRLANL and PRLDES each provide the pitch diameters of the pinions and gears, the effective face-widths, the stage reduction ratios, the numbers of teeth per gear, speed and power splits, and the geometry factors to subroutines PRLRES and PRLSIZ to compute all further information. The speed splits are the actual speeds of the individual gears and a power split is the actual power transferred by a gear. The strength and durability geometry factors are computed in separate subroutines in Module Four and will be discussed later.

A. COMMON RELATIONSHIPS

Power splits are determined from the configuration. For a single power path configuration, the power is transferred equally from the pinion to the gear, where in a dual power path configuration, the pinion transfers one half its power

to each of two gears. These splits are computed exactly since losses are neglected.

Speed splits and stage reduction ratios are based on;

$$m_G = \frac{D}{d} = \frac{n_G}{n_p} \quad (5)$$

Numbers of teeth on each gear are computed from the equation below and are rounded to the nearest integer.

$$N = d \times P_d \quad (6)$$

B. DETERMINATION OF DIAMETERS AND FACEWIDTHS

All diameters and facewidths are entered by the user in subroutine PRLANL. Stage gear ratios, power and speed splits and numbers of teeth per gear are computed as discussed in the previous section.

In subroutine PRLDES, the diameters, facewidths, and stage gear ratios are determined by using a basic local random search optimization technique to produce a feasible design. This algorithm requires an initial design to start.

The initial design is based on Dudley's [Ref. 7] formulation for preliminary estimates of gear size;

$$C^2F = \frac{31500}{K} \frac{PWR}{n_p} \frac{(m_G + 1)^3}{m_G} \quad (7)$$

$$C = \frac{d}{2} (m_G + 1) \quad (8)$$

By substituting equation 8 into equation 7, a formula for estimating pinion diameter is obtained;

$$d^3 = \frac{126000}{n_p K} \frac{PWR}{(F/d)} \frac{(m_G + 1)}{m_G} \quad (9)$$

where $F/d = 1.0$ for single helical gears and $F/d = 2.25$ for double helical gears. The term K is the K -factor which is an indication of durability. An expression for estimating K is provided by Thoma [Ref. 4];

$$K \leq \left(\frac{S_{ac} \times 10^{-4}}{C_R} \right)^2 \times \left(\frac{2.80}{C_o C_m} \right) \quad (10)$$

where the constants used are the AGMA durability constants. The K -factor in equation 10 is for the second reduction. For the first reduction, multiply K from equation 10 by 1.20. The initial estimates for the stage gear ratios are:

- (1) single reduction $m_G = M_0$
- (2) double reduction $m_{G_2} = \sqrt{M_0} + 3$ dual power path
 $m_{G_2} = \sqrt{M_0} - 1$ single power path
 $m_{G_1} = \sqrt{M_0} / m_{G_2}$
- (3) triple reduction $m_{G_2} = \sqrt[3]{M_0}$
 $m_{G_3} = \sqrt[3]{M_0} + 3$
 $m_{G_1} = \sqrt[3]{M_0} / m_{G_2} m_{G_3} .$

The initial facewidths used are:

- (1) single helical gears $F = d$
- (2) double helical gears $F = 2.25 d$.

With this initial design as a starting point for the random search algorithm, successive designs are determined by randomly adding small amounts of between +1.0 and -1.0 to the diameters, facewidths, and stage gear ratios. These small amounts are scaled to take into account the difference in range of values for each variable. This process will attempt to find a feasible design in which all specified constraints are satisfied. If the initial design violates one or more constraints, the design that violates them the least in succeeding iterations will be kept until a design satisfying all constraints is found. Once a feasible design is found, an attempt to improve this design is made by trying to reduce the size of the gears by minimizing a function of gear pair volume;

$$\text{Volume} = \sum \sum C^2 F = \sum \sum \left[\frac{1}{4} (m_G + 1)^2 d^2 F \right] \quad (11)$$

The interior summation is over the number of reduction stages, and the exterior summation is over the number of power inputs. The constraints imposed which determine the limits on each of the designs are:

- (1) actual transmitted power is less than or equal to the allowable service power in accordance with references 2 and 3
- (2) maximum gear diameter of 200 inches due to manufacturing limitations
- (3) minimum facewidth greater than four axial pitches to ensure proper helical action
- (4) maximum facewidth less than the pinion pitch diameter for a single helical gear or 2.25 times the pinion pitch diameter for a double helical gear
- (5) pinions and gears in succeeding reduction stages are to be larger than those in the previous stage due to the greater amounts of torque carried
- (6) in dual power path arrangements, the gear ratio for each reduction stage is greater than the preceding stage due to the torque carried.

The design obtained can then be adjusted by the user as desired by changing parameters with the analysis option.

III. EPICYCLIC FORMULATIONS

As in Module Two, the pitch diameters, effective face-widths, stage reduction ratios, numbers of teeth, speed and power splits, and the geometry factors are all entered or computed in the analysis or design subroutines (EPCANL or EPCDES) for use in the final computations subroutines (EPCRES and EPCSIZ). Here, the speed splits are the rotational speeds of the sun and planet gears and of either the ring gear or the carrier, depending on the configuration. Planetary arrangements have fixed ring gears while star arrangements have fixed carriers. Also, the direction of rotation must be considered. Star arrangements reverse the direction of rotation of the input and the planetary arrangements will maintain direction of rotation. Assuming equal load sharing of the planets and neglecting losses, power splits are straightforward. The input and output powers are equal while each planet carries an equal share of the total power. Load sharing is an important consideration in the design of epicyclic gears, and must be assured in marine reduction gears due to the high power levels experienced. Equal load sharing of the planets can be reasonably achieved in several different ways. One method requires the sun gear to float, supported only by the planet gears, with

a relatively flexible ring gear to allow for inaccuracies in the teeth. There are also mechanical devices available to assist in achieving an equal division of the load. Experience has shown, for marine applications, that three to five planets with stage ratios in the range of two to eight work best.

A. COMMON RELATIONSHIPS

Unlike conventional parallel axis arrangements, there are specific numerical rules governing the proper assembly and operation of an epicyclic gear set. These involve the selection of the numbers of teeth and planets along with computing the various speed ratios. Mesh frequencies are also configuration dependent as seen in a following section.

There are basically three relationships that must be satisfied to ensure proper assembly and operation. The first is a relationship defining the speed ratio of the epicyclic stage since it is not merely the ratio of numbers of teeth or diameters as in a conventional parallel axis gear set. The second relationship requires the ring gear diameter to be equal to the sum of the sun gear diameter and twice a planet gear's diameter. This ensures the planets' ability to fit between the sun and ring gear. For the final relationship, it can be shown geometrically that the sum of the numbers of teeth on the sun gear and ring gear must be

an integral multiple of the number of planets in the gear set to ensure proper alignment and meshing of all teeth. It should be noted that these relationships are based on equally spaced planets around the sun gear. The above relationships are conveniently expressed in terms of numbers of teeth on each gear as seen in references 7 and 8. The speed ratio for a planetary arrangement is;

$$m_G = \frac{n_o}{n_i} = \frac{N_R}{N_S} + 1 \quad (12)$$

and for a star arrangement;

$$m_G = \frac{-n_o}{n_i} = \frac{-N_R}{N_S} \quad (13)$$

where the negative sign indicates the star arrangement's reversal of rotational direction of the input. The rotational speed of the planet gears is required for the design of their bearings and can be determined by;

$$n_{PLN} = \frac{N_R}{N_{PLN}} n_o \quad (14)$$

where n_o in each equation above is the speed of the carrier for a planetary arrangement or is the ring gear's speed for a star arrangement. The assembly and meshing relations in terms of tooth numbers are;

$$N_R = N_S + 2 N_{PLN} \quad (15)$$

and;

$$N_R + N_S = k NP \quad (16)$$

where k is an integer and NP is the number of planets.

B. DETERMINATION OF DIAMETERS AND FACEWIDTHS

The random search technique discussed for conventional parallel axis gears is used to provide the epicyclic diameters, facewidths, and stage gear ratios. Equation 9 is used to provide an initial estimate of sun gear diameters where m_G is replaced by the ratio of the planet's pitch diameter to the sun's pitch diameter. This value is usually in the range of 1.5 to 3; therefore, a random number in this range is used to start the problem. Once the sun gear diameter is estimated, the other diameters can be found using the relationships in equations 12 to 16. The initial estimates for the stage gear ratio are the roots of the overall ratio corresponding to the number of reduction stages. For example, m_{G_1} and m_{G_2} for a double reduction gear set would be the square root of the overall ratio. Initial facewidths are chosen as before. The initial estimates of the diameters, facewidths, and the gear ratios provide a starting point for

the random search algorithm discussed previously. Again, the method will attempt to improve feasible designs by minimizing a function of gear volume;

$$\text{Volume} = \sum (NP \cdot d_{PLN}^2 + d_S^2 + d_R^2) \cdot F \quad (17)$$

where the summation is over the number of reduction stages. The constraints imposed are similar to those for the parallel axis gears:

- (1) actual transmitted power is less than or equal to the allowable service power in accordance with references 2 and 3
- (2) maximum ring gear diameter of 150 inches due to manufacturing limitations
- (3) minimum facewidth greater than four axial pitches to ensure proper helical action
- (4) maximum facewidth less than the sun's pitch diameter for a single helical gear or 2.25 times the sun's pitch diameter for a double helical gear
- (5) planet gears are to be larger than sun gears due to the greater amounts of torque carried
- (6) stage gear ratios are to be between 2 and 8 for each reduction stage.

As before, once a design is obtained, the user can utilize the analysis option to obtain the desired results.

IV. COMPUTATIONAL RESULTS AND DESIGN INFORMATION

Once the geometry is determined in the analysis or design subroutines, the computational results subroutine (PRLRES or EPCRES) and the size estimates subroutine (PRLSIZ or EPCSIZ) are called to provide design information concerning tooth loads, stresses, and other configuration, geometry, and size information. This section describes the formulations used.

The facewidth to diameter ratio is computed using the effective facewidth and the pitch diameter of the pinion for parallel axis gears or the sun gear for epicyclics. Center distance is taken as the average of the pinion and gear pitch diameters. A center distance is computed for epicyclics by finding the average of the sun and a planet gears' pitch diameters.

Pitchline velocity, V , is determined by;

$$V = \frac{\pi d n_p}{12} \quad (18)$$

where V is in feet per minute, d is in inches, and n_p is in revolutions per minute. The tangential component of tooth load, W_t , is computed from;

$$W_t = \frac{126000 \text{ Pwr}}{n_p d} \quad (19)$$

where W_t is in pounds-force, Pwr is in horsepower, and d and n_p are as before. Tooth loading per inch of facewidth is computed from;

$$\text{Tooth Load per Inch} = W_t / F \quad (20)$$

and the unit load, a normalized value of the load per inch above, is;

$$\text{Unit Load} = \frac{W_t P_{nd}}{F} \quad (21)$$

where the unit load is expressed in pounds-force per square inch.

Mesh frequencies provide information on how often a tooth is loaded. Mesh frequencies for parallel axis gears are determined by;

$$f = \frac{N_p n_p}{60} \quad (22)$$

with f expressed in Hertz. For epicyclic gears, the following are used:

$$\begin{aligned}
 \text{(a)} \quad f_s &= \frac{NP \ N_R}{N_R + N_S} n_s & \text{(d)} \quad f_s &= NP \ n_s \\
 \text{(b)} \quad f_p &= \frac{N_R}{N_{PLN}} \frac{N_S}{N_R + N_S} n_s & \text{(e)} \quad f_p &= 2 \frac{N_S}{N_{PLN}} n_s \\
 \text{(c)} \quad f_R &= \frac{NP \ N_S}{N_R + N_S} n_s & \text{(f)} \quad f_R &= NP \frac{N_S}{N_R} n_s
 \end{aligned} \tag{23}$$

where (a) through (c) are for planetary arrangements and (d) through (f) are for star arrangements.

The K-factor is computed for reference purposes by;

$$K = \frac{W_t}{F \ d} \frac{(m_G + 1)}{m_G} \tag{24}$$

The contact stresses are computed according to reference 2 by;

$$s_c = C_p \sqrt{\frac{W_t \ C_o}{C_v} \frac{C_s}{d \ F} \frac{C_m \ C_f}{I}} \tag{25}$$

Bending stresses are computed according to reference 3 by;

$$s_t = \frac{W_t \ K_o}{K_v} \frac{P_d}{F} \frac{K_s \ K_m}{J} \tag{26}$$

Individual torques, T, are found by;

$$T = \frac{W_t \cdot d}{2000} \quad (27)$$

while the total output torque is computed by;

$$T = \frac{63 \text{ SHP}}{n_p} \quad (28)$$

where T has the units of thousands of inch-pounds-force in both cases. Shaft horsepower, SHP, is the total power transferred to the output shaft.

Weight and size estimates are based on empirical relations obtained from a limited number of actual designs. The relations used are;

$$\begin{aligned} \text{Weight} &= C1 \cdot [\sum (d^2 F)]^{c2} \\ \text{Length} &= C3 \cdot \sum F \\ \text{Width} &= C4 \cdot D \\ \text{Height} &= C5 \cdot D \end{aligned} \quad (29)$$

where the constants used are found in Table 1. All dimensions are in inches and the weight is in pounds-force rounded to three significant figures.

Table 1: Empirical Constants for Weight and
Size Formulations

<u>Constant</u>	<u>Parallel Axis</u>	<u>Epicyclic</u>
C1	1196.0	0.905
C2	0.34	0.89
C3	2.26	2.85
C4	1.20	1.30
	1.37	--
C5	1.28	1.20
D	Bull Gear Diameter	Ring Gear Diameter

first C4: for single power inputs
second C4: for double power inputs

V. COMPUTATIONAL SUBPROGRAMS LIBRARY FORMULATIONS

The formulations provided below are for the major computational subprograms in Module Four. Those that are self-explanatory or are not computational in nature are only described in general.

A. ARCCOS AND ARCSIN

These function subprograms find the arc cosine and arc sine, respectively, for any two arguments. They were added for convenience since not all compilers have them as internal functions.

B. AGMAE1

This function subprogram returns the value of the constants H, L, and M required for the determination of the stress concentration factor, K_t , according to reference 1, for use in computing the strength geometry factor, J. Table E-1 in reference 1 provides the tabulated data necessary to perform a LaGrangian interpolation for each constant for a specified normal pressure angle in degrees. The interpolation formula used is;

$$F(\phi_n) = \frac{(\phi_n - 20)(\phi_n - 14.5)}{57.75} F_1 + \frac{(\phi_n - 14.5)(\phi_n - 25)}{-27.50} F_2 + \frac{(\phi_n - 14.5)(\phi_n - 20)}{52.50} F_3 \quad (30)$$

where F represents the appropriate values of H, L, or M.

C. CKDATA

FUNCTION CKDATA is called by subroutine AGMA to allow the user to selectively change the preprogrammed constants by checking if the value entered is zero. If it is zero, the current value of the specified constant is not changed. This provides for flexibility in changing constants with multiple values, and it guards against inadvertently entering a value of zero.

D. POWERB AND POWERH

These function subprograms are used to compute the maximum allowed service power, in horsepower, that can be transmitted by a gear according to references 2 and 3. The formulation based on the strength rating is;

$$P = \frac{n \ d \ K_v}{126000 \ SF \ K_o} \frac{F}{K_m} \frac{J}{K_s \ P_d} \frac{S_{ac} \ K_L}{K_R \ K_T} \quad (31)$$

and the durability rating formulation is;

$$P = \frac{n \ d}{126000 \ SF} \frac{I \ C_v}{C_s \ C_f \ C_o \ C_m} \left[\frac{S_{at} \ d}{C_p} \frac{C_L \ C_H}{C_R \ C_T} \right]^2 \quad (32)$$

where J and I are the respective geometry factors, F is the effective facewidth, n is the speed of d in revolutions per minute, and d is the pinion pitch diameter for parallel axis or is the sun pitch diameter for epicyclics. All other values are the preprogrammed constants.

E. RTFNDR AND FALFA

The function subprogram RTFNDR, a slightly modified version of FUNCTION ZEROIN [Ref. 6], is used to find the value of the root of the equation programmed in function FALFA. This root is required by the subroutine GFJ for the computation of the strength geometry factor, J.

F. SHRLD

This function subprogram computes the load sharing ratio used in computing the geometry factors. The load sharing ratio, m_N , is determined by;

$$m_N = \frac{P_N}{.95 Z} = \frac{\pi \cos \phi_n}{.95 Z P_{nd}} \quad (33)$$

where Z is the length of action defined as;

$$Z = \frac{1}{2} \left(\sqrt{D_o^2 - D_b^2} + \sqrt{d_o^2 - d_b^2} - \sqrt{D^2 - D_b^2} - \sqrt{d^2 - d_b^2} \right) \quad (34)$$

The subscripts on the pitch diameters are:

(1) o : outside diameter; $d_o = d + (2/P_d)$

(2) b : base diameter; $d_b = d \cos \phi_t$

For epicyclics, replace the outside diameters in equation 34 with inside diameters : $d_i = d - (2/P_d)$.

G. THICK

FUNCTION THICK returns the value of the normal arc thickness of a tooth at a specified diameter given a thickness at another diameter. For external gears;

$$t_2 = d_2 \left((t_1/d_1) + \text{inv } \phi_1 - \text{inv } \phi_2 \right) \quad (35)$$

and for internal gears;

$$t_2 = d_2 \left((t_1/d_1) - \text{inv } \phi_1 + \text{inv } \phi_2 \right) \quad (36)$$

where the subscript 2 represents the desired point and subscript 1 represents the known point. The involute function is defined as;

$$\text{inv } x = \tan x - x \quad (37)$$

The arguments of the involute functions in equations 35 and 36 are the transverse pressure angles at the points under consideration. The pressure angle at the desired point is defined as;

$$\cos \phi_2 = \frac{d_1 \cos \phi_1}{d_2} \quad (38)$$

The known point is usually taken at the pitch circle where $d_1 = d$, $\phi_1 = \phi_n$, and t_1 is defined as

$$t = \frac{P_n}{2} = \frac{\pi}{2 P_d} \cos \psi \quad (39)$$

H. GFI

This subroutine is used to compute the AGMA durability geometry factor, I , in accordance with reference 2. The geometry factor is defined as;

$$I = \frac{\cos \phi_t \sin \phi_t}{2 m_N} \frac{m_G}{(m_G \pm 1)} \quad (40)$$

where m_N is computed by function SHRLD described above. The plus sign applies to external gears and the minus sign applies to internal gears.

I. GFJ

SUBROUTINE GFJ is used to compute the AGMA strength geometry factor, J , in accordance with reference 1 with one major difference: the values used are from analytical developments and are not scaled to a normal diametral pitch of one as are the values used in a graphical layout discussed in reference 1. The strength geometry factor is defined as;

$$J = \frac{Y_C \cos^2 \psi}{K_f m_N} \quad (41)$$

The load sharing ratio, m_N , is computed in FUNCTION SHRLD. The stress concentration factor, K_f , is determined from;

$$K_f = H + \left(\frac{t}{r_f} \right)^L \cdot \left(\frac{t}{h} \right)^M \quad (42)$$

where H , L , and M are determined in FUNCTION AGMAE1. The value of the root fillet radius, r_f , is;

$$r_f = r_r + \frac{(b - r_r)^2}{(d/2 \cos^2 \psi) + (b - r_r)} \quad (43)$$

with the dedendum, $b = 1.25/P_d$, and the root tip radius,

$r_r \cong 0.28/P_{nd}$. The values of t and h are determined from the analytical geometry of the tooth form layout described below.

The tooth form factor, Y , is defined as;

$$Y_c = P_{nd} \left[\frac{\cos \phi_{Ln}}{\cos \phi_n} \left(\frac{1.5}{x C_n} - \frac{\tan \phi_{Ln}}{t} \right) \right]^{-1} \quad (44)$$

where t and x are also from the tooth form layout mentioned previously. The helical factor, C_n , is defined as;

$$C_n = \left[1 - \frac{\nu}{100} \left(1 - \frac{\nu}{100} \right) \right]^{-1} \quad (45)$$

where $\tan \nu = \tan \psi \sin \phi_n$ for $\psi \leq 50^\circ$. The normal load pressure angle at the tip of the tooth, ϕ_{Ln} , can be seen in figures 6 and 7 and is given by;

$$\phi_{Ln} = \cos^{-1} \left(\frac{d_b}{d_o} \right) \pm \frac{t_o}{d_o} \quad (46)$$

where the subscript o pertains to the point on the outside diameter and subscript b pertains to the base circle. The plus sign applies to internal gears and the minus, to external. The thickness, t_o , at the outside diameter is determined by function THICK. For internal gears, replace the outside values with the inside values as before.

The graphical tooth form layout is a method by which the variables h , t , and x can be determined from actual

measurements of a tooth form drawn and scaled for a normal diametral pitch of one for the case where tooth loading is at the tip. Loading at the tip of the tooth is the general practice for considering loads on helical gears. Refer to Figure 7 for the meanings of h , x , and T where $t = 2T$. Before determining h , x , and T analytically, several reference parameters must be determined as suggested by McIntire and Lyon [Ref. 9]. The first is the radius from the center of the gear to the tip of the inscribed Lewis stress parabola which is point E in Figure 7. This point is the intersection of the line of action of the tip load, tangent to the base circle, with the tooth centerline. The radius to this point is;

$$r_v = \frac{d_v}{2} = \frac{d_b}{2 \cos \phi_{t_n}} \quad (47)$$

An additional reference point is required to fix the geometry. The center of of the root fillet is taken as this point which can be obtained by a very close approximation. To locate this point, the gear center is taken as the origin of a cartesian coordinate system with the tooth centerline as the vertical axis. Two possible cases exist for the location of this point with respect to the base circle. Figure 8 shows the case where the point is inside the base circle and Figure 9 shows the case where it is outside. The

coordinates of this point, (XC, YC), can be found from Figures 8 and 9. For both cases it can be seen in Figures 8 and 9 that;

$$\text{HYP} = d_R + r_f \quad (48)$$

where $d_R = d - 2b = d - (2.5/P_d)$. From Figure 6, the angle, ϵ , is;

$$\epsilon = \text{inv } \phi + \sin^{-1} \frac{t_c}{d} \quad (49)$$

where t_c is the chordal tooth thickness given by;

$$t_c = t - \frac{t^3 \cos^2 \psi}{6 d^3} \quad (50)$$

and t is the normal arc tooth thickness defined earlier.

For the case in Figure 8;

$$\text{XX} = (\text{HYP}) \sin \epsilon$$

$$\text{XC} = \text{XX} + r_f \quad (\text{a})$$

$$\text{YC} = \sqrt{\text{HYP}^2 - \text{XC}^2} \quad (\text{b}) \quad (51)$$

and for the case in Figure 9;

$$\phi_1 = \cos^{-1} \frac{(d_b/2)}{HYP}$$

$$OPP_1 = (HYP) \sin \phi_1$$

$$OPP_2 \cong OPP_1 - r_1$$

$$HYP_1 = \sqrt{OPP_2^2 + (d_b/2)^2}$$

$$\phi_2 = \cos^{-1} \frac{(d_b/2)}{HYP_1}$$

$$\lambda = \phi_1 \pm \text{inv } \phi_2 - \phi_2$$

"-" for external gears

"+" for internal gears (see Figure 11)

$$\delta = \lambda + \epsilon$$

$$XC = (HYP) \sin \delta \quad (a)$$

(52)

$$YC = (HYP) \cos \delta \quad (b)$$

With the reference values of r_1 , XC , and YC determined, the values of h , $t=2T$, and x can be analytically determined. From Figure 7;

$$XT = r_1 \cos \alpha \quad (53)$$

$$YH = r_1 \sin \alpha \quad (54)$$

$$h = r_v - YC + YH = r_v - YC + r_1 \sin \alpha \quad (55)$$

$$T = (t/2) = XC - XT = XC - r_1 \cos \alpha \quad (56)$$

$$YK = \frac{T}{\tan \alpha} \quad (57)$$

where α must be determined such that;

$$YK = 2h \quad \text{or} \quad YK - 2h = 0 \quad (58)$$

Substituting equations 53 through 57 into 58 yields;

$$F(\alpha) = XC - r_1 \cos \alpha - 2 \tan \alpha (r_1 - YC + r_1 \sin \alpha) = 0 \quad (59)$$

Equation 59 is the function in FALFA called by RTFNDR to solve for α . Once α is determined, h can be determined from equation 55 and T and t from equation 56. To obtain x , observe the following;

$$\gamma = \tan^{-1} (h/T)$$

$$\gamma_1 = (\pi/2) - \gamma$$

and

$$x = T \tan \gamma_1 \quad (60)$$

While not precise, the identical methodology is used for internal gears. Figures 10 and 11 apply. The expressions for internal gears are given without further development;

$$h = -r_v + YC + r, \sin \alpha$$

$$T = XC - r, \cos \alpha$$

$$t = 2T$$

$$\gamma = \tan^{-1} (h/T)$$

$$\gamma_1 = (\pi/2) - \gamma$$

$$x = T \tan \gamma_1$$

(61)

The values for h , t , and x are now used to determine the stress concentration factor, equation 42, and the tooth form factor, equation 44, required to compute the strength geometry factor, J , in equation 41.

APPENDIX B

LIST OF PARAMETERS

While it is not practical to list all variables used in the formulations or the program, it is useful to provide a list of the major variables with a cross-reference between the analytical names and the FORTRAN names. A detailed listing of each common block is also useful when studying the program.

I. PARAMETER CROSS-REFERENCE

This section provides a listing of parameters with both their analytical and FORTRAN names.

<u>Math</u> <u>Symbol</u>	<u>FORTRAN</u> <u>Name</u>	<u>Variable</u> <u>Definition</u>
K_L	AKL	life factor
K_m	AKM	load distribution factor
K_o	AKO	overload factor
K_R	AKR	reliability factor
K_s	AKS	size factor
K_T	AKT	temperature factor
K_v	AKV	dynamic factor
SF	SFB	service factor
C	CDE	center distance (theoretical) (in)

	CDP	(E=epicyclic, P=parallel axis)
C_f	CF	surface finish factor
C_H	CH	hardness factor
C_L	CL	life factor
C_m	CM	load distribution factor
C_o	CO	overload factor
C_p	CP	elastic properties factor
C_R	CR	reliability factor
C_s	CS	size factor
C_T	CT	temperature factor
C_v	CV	dynamic factor
SF	SPH	service factor
ψ	DHELIX	helix angle (deg)
	HELIX	helix angle (rad)
ϕ_t	DPHI	transverse pressure angle (deg)
	PHI	transverse pressure angle (rad)
ϕ_n	DPHIN	normal pressure angle (deg)
	PHIN	normal pressure angle (rad)
D	DG	diameter of gear (in)
d	DP	diameter of pinion (in)
d_{PLN}	DPLN	diameter of planet gears (in)
d_R	DR	diameter of ring gear (in)
		root diameter of a gear (in)
d_s	DS	diameter of sun gear (in)
F	FACEE	facewidth (in)
	FACEP	(E=epicyclic, P=parallel axis)

F/d	FBYDE	f/d ratio (facewidth/diameter)
	FBYDP	(E=epicyclic, P=parallel axis)
I	GEOMI	durability geometry factor (pinion)
	GI	durability geometry factor (sun)
J	GEOMJG	strength geometry factor (gear)
	GEOMJP	strength geometry factor (pinion)
	GJS	strength geometry factor (sun)
	GJPL	strength geometry factor (planet)
K	KFCTRE	computed k-factor
	KFCTRP	(E=epicyclic, P=parallel axis)
f	MFE	mesh frequency (Hz)
	MFP	(E=epicyclic, P=parallel axis)
M _o	MGOE	overall reduction ratio
	MGOP	(E=epicyclic, P=parallel axis)
m _G	MGE	stage reduction ratio
	MGP	(E=epicyclic, P=parallel axis)
N _G	NG	number of teeth, gear
N _p	NP	number of teeth, pinion
NP	NPLNT	number of planet gears in epicyclic set
N _{PLN}	NPLN	number of teeth, planet
N _R	NR	number of teeth, ring
N _S	NS	number of teeth, sun
P _d	PD	transverse diametral pitch
P _{nd}	PND	normal diametral pitch
V	PLVE	pitch line velocity (fpm)
	PLVP	(E=epicyclic, P=parallel axis)

PWR	PWRE	power split per gear pair (hp)
	PWRP	(E=epicyclic, P=parallel axis)
n_{in}	RPMIN	source speed input (rpm)
n_{out}	RPMOUT	output shaft/propeller speed (rpm)
n_i	RPMI	stage input speed, epicyclic (rpm)
n_o	RPMO	stage output speed, epicyclic (rpm)
n_{PLN}	RPMPL	planet speed, epicyclic (rpm)
n_P, n_G	RPMP	stage pinion and gear speed, parallel axis (rpm)
s_{ac}	SAC	allowable contact stress number
s_{at}	SAT	allowable bending stress number
SHP	SHP	shaft horsepower, output (hp)
s_t	SIGBE	bending stress (psi)
	SIGBP	(E=epicyclic, P=parallel axis)
s_c	SIGHE	contact stress (psi)
	SIGHP	(E=epicyclic, P=parallel axis)
T	TORQE	torgue (k in-lb)
	TORQP	(E=epicyclic, P=parallel axis)
W_t	WTE	tangential tooth load (lb)
	WTP	(E=epicyclic, P=parallel axis)

II. COMMON BLOCK DETAILS

The following provides information concerning the variables in each common block. The numbers in parentheses are the size of the array where applicable.

COMMON BLOCK AGMAB (FOR STRENGTH RATING)

SFB : R^{*4} (2,2); service factor
AKV : R^{*4} ; dynamic factor
AKS : R^{*4} ; size factor
AKM : R^{*4} ; load distribution factor
AKO : R^{*4} (2); overload factor
SAT : R^{*4} (6); allowable bending stress number
AKL : R^{*4} (2); life factor
AKR : R^{*4} (6); reliability factor
AKT : R^{*4} ; temperature factor

COMMON BLOCK AGMAH (FOR DURABILITY RATING)

SFH : R^{*4} (2,2); service factor
CV : R^{*4} (3); dynamic factor
CS : R^{*4} ; size factor
CM : R^{*4} (2); load distribution factor
CF : R^{*4} ; surface finish factor
CO : R^{*4} (2); overload factor
SAC : R^{*4} (6); allowable contact stress number
CP : R^{*4} ; elastic properties factor
CL : R^{*4} (2); life factor
CH : R^{*4} ; hardness factor
CT : R^{*4} ; temperature factor
CR : R^{*4} (6); reliability factor

COMMON BLOCK DESDAT (DESIGN PARAMETERS, INPUT)

PWRIN : R*4 (2); source power input (hp)
RPMIN : R*4 (2); source speed input (rpm)
RPMOUT: R*4; output shaft/propeller speed (rpm)
DHELIX: R*4 (3); helix angle (deg)
HELIX : R*4 (3); helix angle (rad)
PD : R*4 (3); transverse diametral pitch
PND : R*4 (3); normal diametral pitch
DPHI : R*4 (3); transverse pressure angle (deg)
PHI : R*4 (3); transverse pressure angle (rad)
DPHIN : R*4 (3); normal pressure angle (deg)
PHIN : R*4 (3); normal pressure angle (rad)
NDIFP : I*4; number of different power sources
IARR : I*4; arrangement code (1=parallel axis,
2=epicyclic)
IEPIC : I*4 (3); epicyclic code (1=planetary, 2=star)
IHARD : I*4 (3,2); hardness range code (1-6, see SUBR.
AGMA)
IOPRO : I*4; operational profile code (1=naval pro-
file full power 5% max; 2=other, max
power continuous)
NPWRIN: I*4; number of power sources (inputs)
IPWRSR: I*4 (2); power source code (1=turbine or motor,
2=multicylinder internal combustion
engine)
NRED : I*4; number of reduction stages

NPATH : I*4; number of power paths (1=single,2=dual)
 NPLNT : I*4 (3); number of planet gears in epicyclic set
 NHELX : I*4; number of helicies (1=single, 2=double)

COMMON BLOCK DESEPC (EPICYCLIC DESIGN PARAMETERS)

MGOE : R*4; overall reduction ratio
 MGE : R*4 (3); stage reduction ratio
 RPMI : R*4 (3); stage input speed (rpm)
 RPMPL : R*4 (3); planet speed (rpm)
 RPMO : R*4 (3); stage output speed (rpm)
 PWRE : R*4 (3); stage power split per planet (hp)
 DS : R*4 (3); diameter of sun gear (in)
 DPLN : R*4 (3); diameter of planet gears (in)
 DR : R*4 (3); diameter of ring gear (in)
 FACEE : R*4 (3); facewidth (in)
 GI : R*4 (3); durability geometry factor (sun/planet)
 GJS : R*4 (3); strength geometry factor (sun)
 GJPL : R*4 (3); strength geometry factor (planet)
 NS : I*4 (3); number of teeth, sun
 NPLN : I*4 (3); number of teeth, planet
 NR : I*4 (3); number of teeth, ring

COMMON BLOCK DESPRL (PARALLEL AXIS DESIGN PARAMETERS)

PWRFAC: R*4 (2,3); stage power split factor
 MGOP : R*4 (2); overall reduction ratio
 MGP : R*4 (3,2); stage reduction ratio

RPMP : R*4 (6,2); stage pinion and gear speed (rpm)
 PWRP : R*4 (6,2); stage power split per gear (hp)
 DP : R*4 (3,2); diameter of pinion (in)
 DG : R*4 (3,2); diameter of gear (in)
 FACEP : R*4 (3,2); facewidth (in)
 GEOMI : R*4 (3,2); durability geometry factor
 GEOMJG: R*4 (3,2); strength geometry factor (gear)
 GEOMJP: R*4 (3,2); strength geometry factor (pinion)
 NP : I*4 (3,2); number of teeth, pinion
 NG : I*4 (3,2); number of teeth, gear

COMMON BLOCK RESEPC (EPICYCLIC PARAMETERS, RESULTS)

PLVE : R*4 (3); pitch line velocity (fpm)
 FBYDE : R*4 (3); f/d ratio (facewidth/sun diameter)
 CDE : R*4 (3); center distance (theoretical) (in)
 WTE : R*4 (3); tangential tooth load (lb)
 TLPIE : R*4 (3); tooth load per in (lb/in)
 UNTLDE: R*4 (3); unit load (psi)
 MFE : R*4 (3,3); mesh frequency (Hz)
 KFCTRE: R*4 (3); computed k-factor
 SIGHE : R*4 (3); contact stress (psi)
 SIGBE : R*4 (3); bending stress (psi)
 TORQE : R*4 (3,3); torque (k in-lb)
 RPME : R*4 (3,3); gear speeds (rpm)
 PDIAME: R*4 (3,3); pitch diameters (in)
 WGHTTE : R*4; gear set weight estimate (lb)

SPCWTE: R*4; specific weight (lb/hp)
MTHE : I*4 (3,3); tooth numbers
ISIZEE: I*4 (3); length, width, height estimates (in)

COMMON BLOCK RESPRL (PARALLEL AXIS PARAMETERS, RESULTS)

PLVP : R*4 (3,2); pitch line velocity (fpm)
FBYDP : R*4 (3,2); f/d ratio (facewidth/pinion diameter)
CDP : R*4 (3,2); center distance (theoretical) (in)
WTP : R*4 (6,2); tangential tooth load (lb)
TLPIP : R*4 (6,2); tooth load per inch (lb/in)
UNTLDP: R*4 (6,2); unit load (psi)
MFP : R*4 (3,2); mesh frequency (Hz)
KFCTRP: R*4 (6,2); computed k-factor
SIGHP : R*4 (3,2); contact stress (psi)
SIGBP : R*4 (6,2); bending stress (psi)
TORQP : R*4 (6,2); torque (k in-lb)
PDIAMP: R*4 (6,2); pitch diameters (in)
SCDMIN: R*4; minimum source center distance (in)
SCDMAX: R*4; maximum source center distance (in)
SHP : R*4; shaft horsepower, output (hp)
WGHTP : R*4; gear set weight estimate (lb)
SPCWTP: R*4; specific weight (lb/hp)
TRQOUT: R*4; torque, output (k in-lb)
MTHP : I*4 (6,2); tooth numbers
ISIZEP: I*4 (3); length, width, height estimates (in)

APPENDIX C

REGAD SAMPLE RUNS

This appendix contains samples of actual terminal sessions using REGAD. For the sake of brevity, only two complete sessions are included. However, a number of analysis and design runs were made using a full range of options and configurations, and they compared favorably to actual designs. The comparisons are not shown here due to the proprietary nature of the designs used for verification. The first example is an analysis run for a locked train, double reduction gear set with two different inputs. Following it, is the results section from a design run using the identical parameters as the analysis run. The second example is a double reduction epicyclic gear set with the complete analysis session followed by the results section of a design run as before. The analysis and design sessions are identical with one exception. A seed for a random number generator is requested in the design option instead of diameters and facewidths as in the analysis option. For those cases where an infeasible design is generated, a message will alert the user and the program will continue. To obtain a feasible design, or just a different one, rerun the program and

provide a different seed for the random number generator. This method was used on several occasions to obtain the desired results. Once a feasible design is obtained, the user can then use the analysis option to obtain a design that more closely suits his needs.

I. PARALLEL AXIS GEAR SET
Analysis Session

REGAD
REDUCTION GEAR ANALYSIS AND DESIGN

DO YOU DESIRE A PROGRAM DESCRIPTION? (Y OR N) :

Y

THIS PROGRAM IS CAPABLE OF PERFORMING PRELIMINARY DESIGN

OR ANALYSIS OF MULTIREDUCTION, PARALLEL AXIS AND EPICYCLIC
REDUCTION GEARS. THE CAPABILITIES AND FEATURES OF THE PRO-
GRAM ARE AS FOLLOWS:

- 1) MAXIMUM OF THREE REDUCTION STAGES ALLOWED
- 2) CHOICE OF SINGLE OR DOUBLE HELICALS
- 3) WEIGHT AND SIZE ESTIMATES PROVIDED
- 4) FOR PARALLEL AXIS GEARS:
 - ONE OR TWO POWER SOURCES ALLOWED
 - SINGLE OR DUAL POWER PATHS ALLOWED
- 5) FOR EPICYCLIC GEARS:
 - ONLY ONE POWER SOURCE ALLOWED
 - LIMITED TO 3, 4, OR 5 PLANET GEARS
 - ONLY SIMPLE EPICYCLICS PER REDUCTION STAGE
 - PLANETARY OR STAR ARRANGEMENTS POSSIBLE

THE STANDARDS OF THE AMERICAN GEAR MANUFACTURING ASSOCI-
ATION WERE USED AS A BASIS FOR THIS PROGRAM. THE CONSTANTS

USED IN THE AGMA FORMULATIONS ARE BASED ON THOSE PUBLISHED BY F. A. THOMA, OF DELAVAL TURBINE, FOR MARINE PROPULSION GEARS. AN OPTION IS PROVIDED DURING EXECUTION OF THE PROGRAM TO OBTAIN A LISTING OF THESE CONSTANTS, AND TO CHANGE ANY OF THEM FOR OTHER POSSIBLE APPLICATIONS.

IT SHOULD BE NOTED THAT THE STRESSES LISTED IN THE OUTPUT ARE THOSE COMPUTED FROM THE AGMA FORMULATIONS AND ARE NOT FROM A DETAILED STRESS ANALYSIS.

FOR MORE SPECIFIC INFORMATION, SEE THE USERS MANUAL OR OBTAIN A LISTING OF THE PROGRAM.

DO YOU WISH THE PROGRAM TO CONTINUE INTO THE ANALYSIS AND DESIGN SEGMENTS? (Y OR N):

Y

YOU WILL NOW BE ASKED TO PROVIDE THE PARAMETERS REQUIRED FOR THE ANALYSIS OR DESIGN IN THIS RUN.

*** ENTER PROGRAM OPTION CODE (1=DESIGN, 2=ANALYSIS):

?
2

*** ENTER ARRANGEMENT CODE (1=PARALLEL AXIS, 2=EPICYCLIC):

?
1

CHOOSE OPERATIONAL PROFILE CODE BELOW:
OPERATIONAL MODE SERVICE PROFILE CODE

FULL POWER 5 PERCENT MAX 1
MAXIMUM LOAD CONTINUOUS 2

ENTER OPERATIONAL PROFILE CODE:

?
1

ENTER NUMBER OF REDUCTIONS (1, 2, OR 3):

?
2

CHOOSE DESIRED HELIX TYPE BELOW:

TYPE	ANGLE	CODE
SINGLE	15-25	1
DOUBLE	25-50	2

ENTER HELIX CODE:

?
2

ENTER NUMBER OF POWER PATHS (1=SINGLE, 2=DUAL):

?
2

ENTER NUMBER OF POWER SOURCES (1 OR 2):

?
2

WILL ANY POWER SOURCE BE A MULTICYLINDER I. C. ENGINE? (Y OR N):

N

ENTER POWER AND SPEED OF HIGH POWER SOURCE (HP, RPM):

?
21250, 6990

ENTER POWER AND SPEED OF LOW POWER SOURCE (HP, RPM) :
?
21250, 5980

ENTER OUTPUT SHAFT/PROPELLER SPEED (RPM) :
?
300

WHICH DIAMETRAL PITCH WILL YOU SPECIFY?
(1=TRANSVERSE, 2=NORMAL) :
?
1

WHICH PRESSURE ANGLE WILL YOU SPECIFY?
(1=TRANSVERSE, 2=NORMAL) :
?
2

THE FOLLOWING PARAMETERS ARE REQUESTED FOR STAGE 1 :

ENTER HELIX ANGLE (DEGREES) :
?
35

ENTER TRANSVERSE DIAMETRAL PITCH :
?
4.5

ENTER NORMAL PRESSURE ANGLE (DEGREES) :
?
20

CHOOSE GEAR HARDNESS RANGE BELOW:

BHN	CODE
160 - 200	1
200 - 240	2
240 - 300	3
300 - 360	4
360 - 400	5
400 - 640	6

ENTER HARDNESS CODES FOR PINION AND GEAR (HCPIN, HCGEAR) :

?
2,1

THE FOLLOWING PARAMETERS ARE REQUESTED FOR STAGE 2 :

ENTER HELIX ANGLE (DEGREES) :

?
35

ENTER TRANSVERSE DIAMETRAL PITCH:

?
3.5

ENTER NORMAL PRESSURE ANGLE (DEGREES) :

?
20

CHOOSE GEAR HARDNESS RANGE BELOW:

BHN	CODE
160 - 200	1
200 - 240	2
240 - 300	3
300 - 360	4
360 - 400	5

ENTER HARDNESS CODES FOR PINION AND GEAR (HCPIN, HCGEAR):
 ?
 2,1

TO MAKE CORRECTIONS TO DATA JUST ENTERED, THE PROGRAM MUST BE
 ABORTED AND RE-STARTED. DO YOU WISH TO ABORT THIS RUN? (Y OR N):
 n

DO YOU DESIRE A LISTING OF THE PRE-PROGRAMMED CONSTANTS USED
 IN THE AGMA FORMULATIONS? (Y OR N):
 Y

THE FOLLOWING IS A LISTING OF THE PRE-PROGRAMMED CONSTANTS
 USED IN THE AGMA FORMULATIONS WITH APPROPRIATE NOTES ON
 THEIR APPLICATION. NOTE: THOSE STARTING WITH A 'C' ARE
 DURABILITY CONSTANTS AND THOSE WITH A 'K' ARE STRENGTH
 CONSTANTS. SERVICE FACTOR APPLIES TO BOTH FORMULATIONS.

ID	CONST	VALUE(S)	NOTES
1	SF (1, 1)	1.00	SERVICE FACTOR; A1, B1
	SF (1, 2)	1.50	A1, B2
	SF (2, 1)	1.50	A2, B1
	SF (2, 2)	1.75	A2, B2
2	CV (1)	1.00	DYNAMIC FACTOR; C1
	CV (2)	0.83	C2
	CV (3)	0.69	C3
3	CS	1.00	SIZE FACTOR
4	CM (1)	1.25	LOAD DISTRIBUTION FACTOR; A1
	CM (2)	1.35	A2

5	CF	1.00	SURFACE CONDITION FACTOR
6	CO(1)	1.15	OVERLOAD FACTOR; A1
	CO(2)	1.14	A2
7	CP	2300.0	ELASTIC PROPERTIES FACTOR
8	CL(1)	0.80	LIFE FACTOR; A1
	CL(2)	0.68	A2
9	CH	1.00	HARDNESS RATIO FACTOR
10	CT	1.00	TEMPERATURE FACTOR
11	CR(1)	1.16	RELIABILITY FACTOR; D1
	CR(2)	1.19	D2
	CR(3)	1.22	D3
	CR(4)	1.27	D4
	CR(5)	1.31	D5
	CR(6)	1.35	D6
12	SAC(1)	95000.	ALLOWABLE CONTACT STRESS; D1
	SAC(2)	108000.	D2
	SAC(3)	125000.	D3
	SAC(4)	146000.	D4
	SAC(5)	165000.	D5
	SAC(6)	182000.	D6
13	KV	0.70	DYNAMIC FACTOR
14	KS	1.00	SIZE FACTOR
15	KM	1.10	LOAD DISTRIBUTION FACTOR
16	KO(1)	1.21	OVERLOAD FACTOR; E1

17	KO (2)	1.28		
	KL (1)	0.80	LIFE FACTOR; A1	
	KL (2)	0.68	A2	
18	KT	1.00	TEMPERATURE FACTOR	
19	KR (1)	1.16	RELIABILITY FACTOR; D1	D1
	KR (2)	1.18		D2
	KR (3)	1.23		D3
	KR (4)	1.29		D4
	KR (5)	1.31		D5
	KR (6)	1.33		D6
20	SAT (1)	32900.	ALLOWABLE MATERIAL STRESS; D1	D1
	SAT (2)	38100.		D2
	SAT (3)	44500.		D3
	SAT (4)	51750.		D4
	SAT (5)	54250.		D5
	SAT (6)	61000.		D6

DEFINITIONS OF CODED NOTES FROM ABOVE:

A1 NAVAL PROFILE - FULL POWER, 5 PERCENT MAX

A2 OTHER - MAXIMUM LOAD, CONTINUOUS

B1 POWER SOURCE - TURBINE OR MOTOR

B2 POWER SOURCE - MULTICYLINDER I. C. ENGINE

C1 FIRST REDUCTION STAGE

C2 SECOND REDUCTION STAGE

C3 THIRD REDUCTION STAGE

D1 HARDNESS RANGE: 160 - 200 BHN

D2 HARDNESS RANGE: 200 - 240 BHN

D3 HARDNESS RANGE: 240 - 300 BHN

D4 HARDNESS RANGE: 300 - 360 BHN

D5 HARDNESS RANGE: 360 - 400 BHN
D6 HARDNESS RANGE: 400 - 640 BHN

E1 SINGLE POWER PATH
E2 DOUBLE POWER PATH

DO YOU DESIRE TO CHANGE ANY OF THE ABOVE VALUES? (Y OR N):

Y

TO CHANGE A CONSTANT ABOVE, ENTER THE ID NUMBER WHEN PROMPTED.
USE ID NUMBER 99 WHEN NO FURTHER CHANGES ARE TO BE MADE.
NOTE: WHEN ASKED FOR THE NEW VALUE OF THE CONSTANT, ENTERING
A ZERO WILL CAUSE THE ORIGINAL VALUE TO REMAIN UNCHANGED.
THIS IS USEFUL WHEN A CONSTANT HAS MULTIPLE VALUES, BUT NOT
ALL OF THEM ARE TO BE CHANGED.

*^ ENTER THE CONSTANT ID NUMBER (1-20, 99 TO STOP):

?

16

*^ ENTER KO(1):

?

1.14

*^ ENTER KO(2):

?

0

*^ ENTER THE CONSTANT ID NUMBER (1-20, 99 TO STOP):

?

99

THE FOLLOWING IS A LISTING OF THE PRE-PROGRAMMED CONSTANTS USED IN THE AGMA FORMULATIONS WITH APPROPRIATE NOTES ON THEIR APPLICATION. NOTE: THOSE STARTING WITH A 'C' ARE DURABILITY CONSTANTS AND THOSE WITH A 'K' ARE STRENGTH CONSTANTS. SERVICE FACTOR APPLIES TO BOTH FORMULATIONS.

ID	CONST	VALUE(S)	NOTES
1	SF (1, 1)	1.00	SERVICE FACTOR; A1, B1
	SF (1, 2)	1.50	A1, B2
	SF (2, 1)	1.50	A2, B1
	SF (2, 2)	1.75	A2, B2
2	CV (1)	1.00	DYNAMIC FACTOR; C1
	CV (2)	0.83	C2
	CV (3)	0.69	C3
3	CS	1.00	SIZE FACTOR
4	CM (1)	1.25	LOAD DISTRIBUTION FACTOR; A1
	CM (2)	1.35	A2
5	CF	1.00	SURFACE CONDITION FACTOR
6	CO (1)	1.15	OVERLOAD FACTOR; A1
	CO (2)	1.14	A2
7	CP	2300.0	ELASTIC PROPERTIES FACTOR
8	CL (1)	0.80	LIFE FACTOR; A1
	CL (2)	0.68	A2
9	CH	1.00	HARDNESS RATIO FACTOR
10	CT	1.00	TEMPERATURE FACTOR
11	CR (1)	1.16	RELIABILITY FACTOR; D1

CR (2)	1.19			D2
CR (3)	1.22			D3
CR (4)	1.27			D4
CR (5)	1.31			D5
CR (6)	1.35			D6
12	SAC (1)	95000.	ALLOWABLE CONTACT STRESS;	D1
	SAC (2)	108000.		D2
	SAC (3)	125000.		D3
	SAC (4)	146000.		D4
	SAC (5)	165000.		D5
	SAC (6)	182000.		D6
13	KV	0.70	DYNAMIC FACTOR	
14	KS	1.00	SIZE FACTOR	
15	KM	1.10	LOAD DISTRIBUTION FACTOR	
16	KO (1)	1.14	OVERLOAD FACTOR; E1	
	KO (2)	1.28	E2	
17	KL (1)	0.80	LIFE FACTOR; A1	
	KL (2)	0.68	A2	
18	KT	1.00	TEMPERATURE FACTOR	
19	KR (1)	1.16	RELIABILITY FACTOR; D1	
	KR (2)	1.18	D2	
	KR (3)	1.23	D3	
	KR (4)	1.29	D4	
	KR (5)	1.31	D5	
	KR (6)	1.33	D6	
20	SAT (1)	32900.	ALLOWABLE MATERIAL STRESS; D1	
	SAT (2)	38100.	D2	

D3
D4
D5
D6

SAT (3) 44500.
SAT (4) 51750.
SAT (5) 54250.
SAT (6) 61000.

DEFINITIONS OF CODED NOTES FROM ABOVE:

A1 NAVAL PROFILE - FULL POWER, 5 PERCENT MAX
A2 OTHER - MAXIMUM LOAD, CONTINUOUS

B1 POWER SOURCE - TURBINE OR MOTOR
B2 POWER SOURCE - MULTICYLINDER I. C. ENGINE

C1 FIRST REDUCTION STAGE
C2 SECOND REDUCTION STAGE
C3 THIRD REDUCTION STAGE

D1 HARDNESS RANGE: 160 - 200 BHN
D2 HARDNESS RANGE: 200 - 240 BHN
D3 HARDNESS RANGE: 240 - 300 BHN
D4 HARDNESS RANGE: 300 - 360 BHN
D5 HARDNESS RANGE: 360 - 400 BHN
D6 HARDNESS RANGE: 400 - 640 BHN

E1 SINGLE POWER PATH
E2 DOUBLE POWER PATH

THE INFORMATION REQUESTED BELOW IS FOR REDUCTION STAGE 1
IN POWER TRAIN 1.

ENTER DIAMETERS OF PINION AND GEAR, INCHES (DP,DG) :

?

9.31,29.35

英吋 ENTER FACEWIDTH OF GEAR PAIR, INCHES:

?

16.62

THE INFORMATION REQUESTED BELOW IS FOR REDUCTION STAGE 2
IN POWER TRAIN 1.

英吋 ENTER DIAMETERS OF PINION AND GEAR, INCHES (DP,DG):

?

14.24, 105.25

英吋 ENTER FACEWIDTH OF GEAR PAIR, INCHES:

?

25.97

THE INFORMATION REQUESTED BELOW IS FOR REDUCTION STAGE 1
IN POWER TRAIN 2.

英吋 ENTER DIAMETERS OF PINION AND GEAR, INCHES (DP,DG):

?

10.01, 26.98

英吋 ENTER FACEWIDTH OF GEAR PAIR, INCHES:

?

18.02

THE INFORMATION REQUESTED BELOW IS FOR REDUCTION STAGE 2
IN POWER TRAIN 2.

英吋 ENTER ONLY DIAMETER OF PINION, INCHES (DP):

?

14.24

POWER SOURCE 1: TURBINE OR MOTOR
 INPUT POWER (HP): 21250. INPUT SPEED (RPM): 6990.

ARRANGEMENT: PARALLEL AXIS, 2 INPUT(S), 2 POWER PATH(S), 2 REDUCTION(S)
 OUTPUT POWER (HP): 42500.0 OUTPUT SPEED (RPM): 300.
 RATIO: 23.301 OUTPUT TORQUE (K IN-LB): 8925.0

SOURCE CENTER DISTANCE (IN): MIN= 38.8 MAX= 49.4

SIZING ESTIMATES FOR THE ENTIRE REDUCTION SET:

WEIGHT (LB): 93200. SPECIFIC WEIGHT (LB/HP): 2.19
 LENGTH (IN): 99 WIDTH (IN): 135 HEIGHT (IN): 144

	REDUCTION 1	REDUCTION 2
	PINION GEAR	PINION GEAR
POWER SPLIT	HP 21250. 10625.	10625. 21250.
SPEED	RPM 6990. 2217.	2217. 300.
NUMBER OF TEETH	42 132	50 368
NORMAL DIAMETRAL PITCH	5.493	4.273
TRANS. DIAMETRAL PITCH	4.500	3.500
NORMAL PRESSURE ANGLE	20.0	20.0
TRANS. PRESSURE ANGLE	24.0	24.0
HELIX ANGLE	35.0	35.0
GEAR RATIO	3.153	7.391
PITCH DIAMETER	IN 9.31 29.35	14.24 105.25
EFFECTIVE FACEWIDTH	IN 16.62	25.97
F/DP	1.79	1.82
CENTER DISTANCE	IN 19.33	59.74
PITCHLINE VELOCITY	FPM 17037.	8266.
TANGENTIAL LOAD	LB 41160. 20580.	42417. 84835.
TOOTH LOAD/IN	LB/IN 2477. 1238.	1633. 3267.
UNIT LOAD	PSI 13605. 6802.	6979. 13957.

MESH FREQUENCY HZ 4893. | 1848. |
 K FACTOR (COMPUTED) 350. | 175. | 130. | 260. |
 CONTACT STRESS PSI 89094. | 59079. |
 BENDING STRESS PSI 37381. | 17155. | 18416. | 33846. |
 TORQUE K IN-LB 191.6 | 302.0 | 302.0 | 4464.4 |
 HARDNESS RANGE BHN 200-240 | 160-200 | 200-240 | 160-200 |

此表係根據設計者提供之數據計算所得，其準確度與設計者提供之數據之準確度有關。

POWER SOURCE 2: TURBINE OR MOTOR
 INPUT POWER (HP): 21250. INPUT SPEED (RPM): 5980.

ARRANGEMENT: PARALLEL AXIS, 2 INPUT(S), 2 POWER PATH(S), 2 REDUCTION(S)
 OUTPUT POWER (HP): 42500.0 OUTPUT SPEED (RPM): 300.
 RATIO: 19.921 OUTPUT TORQUE (K IN-LB): 8925.0

SOURCE CENTER DISTANCE (IN): MIN= 38.8 MAX= 49.4

SIZING ESTIMATES FOR THE ENTIRE REDUCTION SET:

WEIGHT (LB): 93200. SPECIFIC WEIGHT (LB/HP): 2.19
 LENGTH (IN): 99 WIDTH (IN): 135 HEIGHT (IN): 144

	REDUCTION 1		REDUCTION 2	
	PINION	GEAR	PINION	GEAR
POWER SPLIT	HP	21250.	10625.	21250.
SPEED	RPM	5980.	2219.	300.
NUMBER OF TEETH		45	121	50
NORMAL DIAMETRAL PITCH		5.493		4.273
TRANS. DIAMETRAL PITCH		4.500		3.500
NORMAL PRESSURE ANGLE		20.0		20.0
TRANS. PRESSURE ANGLE		24.0		24.0
HELIX ANGLE		35.0		35.0
GEAR RATIO		2.695		7.391
PITCH DIAMETER	IN	10.01	26.98	14.24
				105.25

EFFECTIVE FACEWIDTH	IN	18.02		25.97
F/DP		1.80		1.82
CENTER DISTANCE	IN	18.49		59.74
PITCHLINE VELOCITY	FPM	15671.		8271.
TANGENTIAL LOAD	LB	44747.	22374.	42390.
TOOTH LOAD/IN	LB/IN	2483.	1242.	1632.
UNIT LOAD	PSI	13641.	6821.	6974.
MESH FREQUENCY	HZ	4485.		1849.
K FACTOR (COMPUTED)		340.	170.	130.
CONTACT STRESS	PSI	87718.		59060.
BENDING STRESS	PSI	37061.	17230.	18404.
TORQUE	K IN-LB	224.0	301.8	301.8
HARDNESS RANGE	BHN	200-240	160-200	200-240
				160-200

此係在設計及製造時應注意之事項，請參閱本廠之技術手冊，以資參考。

Results from Design Session

ENTER SEED FOR RANDOM NUMBER GENERATOR (X.XX):

0.76

POWER SOURCE 1: TURBINE OR MOTOR INPUT SPEED (RPM): 6990.

ARRANGEMENT: PARALLEL AXIS, 2 INPUT(S), 2 POWER PATH(S), 2 REDUCTION(S)
 OUTPUT POWER (HP): 42500.0 OUTPUT SPEED (RPM): 300.
 RATIO: 23.300 OUTPUT TORQUE (K IN-LB): 8925.0

SOURCE CENTER DISTANCE (IN): MIN= 57.7 MAX= 57.7

SIZING ESTIMATES FOR THE ENTIRE REDUCTION SET:

WEIGHT (LB): 125000. SPECIFIC WEIGHT (LB/HP): 2.94
 LENGTH (IN): 143 WIDTH (IN): 165 HEIGHT (IN): 177

	REDUCTION 1		REDUCTION 2	
	PINION	GEAR	PINION	GEAR
POWER SPLIT	21250.	10625.	10625.	21250.
SPEED	6990.	2411.	2411.	300.
NUMBER OF TEETH	72	209	56	452
NORMAL DIAMETRAL PITCH	5.493			
TRANS. DIAMETRAL PITCH	4.500			
NORMAL PRESSURE ANGLE	20.0			
TRANS. PRESSURE ANGLE	24.0			
HELIX ANGLE	35.0			
GEAR RATIO	2.899			
PITCH DIAMETER IN	15.99	46.35	16.06	129.07
EFFECTIVE FACEWIDTH IN	30.91			

F/DP		1.93		2.00	
CENTER DISTANCE	IN	31.17		72.57	
PITCHLINE VELOCITY	FPM	29258.		10137.	
TANGENTIAL LOAD	LB	23968.		34587.	
TOOTH LOAD/IN	LB/IN	775.		1074.	
UNIT LOAD	PSI	4259.		4591.	
MESH FREQUENCY	HZ	8388.		2250.	
K FACTOR (COMPUTED)		65.		75.	
CONTACT STRESS	PSI	37970.		44786.	
BENDING STRESS	PSI	10804.		11900.	
TORQUE	K IN-LB	191.6		277.7	
HARDNESS RANGE	BHN	200-240		200-240	

POWER SOURCE 2: TURBINE OR MOTOR INPUT SPEED (RPM): 5980.

ARRANGEMENT: PARALLEL AXIS, 2 INPUT(S), 2 POWER PATH(S), 2 REDUCTION(S)
 OUTPUT POWER (HP): 42500.0 OUTPUT SPEED (RPM): 300.
 RATIO: 19.933

SOURCE CENTER DISTANCE (IN): MIN= 57.7 MAX= 57.7

SIZING ESTIMATES FOR THE ENTIRE REDUCTION SET:

WEIGHT (LB): 125000. SPECIFIC WEIGHT (LB/HP): 2.94
 LENGTH (IN): 143 WIDTH (IN): 165 HEIGHT (IN): 177

	REDUCTION 1	REDUCTION 2
	PINION GEAR	PINION GEAR
POWER SPLIT	HP 21250. 10625.	10625. 21250.
SPEED	RPM 5980. 1851.	1851. 300.
NUMBER OF TEETH	76 247	73 452
NORMAL DIAMETRAL PITCH	5.493	4.273

II. EPICYCLIC GEAR SET
Analysis Session

REGAD
REDUCTION GEAR ANALYSIS AND DESIGN

DO YOU DESIRE A PROGRAM DESCRIPTION? (Y OR N) :

N

YOU WILL NOW BE ASKED TO PROVIDE THE PARAMETERS REQUIRED
FOR THE ANALYSIS OR DESIGN IN THIS RUN.

ENTER PROGRAM OPTION CODE (1=DESIGN, 2=ANALYSIS) :

?
2

ENTER ARRANGEMENT CODE (1=PARALLEL AXIS, 2=EPICYCLIC) :

?
2

CHOOSE OPERATIONAL PROFILE CODE BELOW:
OPERATIONAL MODE SERVICE PROFILE CODE
FULL POWER 5 PERCENT MAX 1
MAXIMUM LOAD CONTINUOUS 2

ENTER OPERATIONAL PROFILE CODE:

?
1

ENTER NUMBER OF REDUCTIONS (1, 2, OR 3):

?
2

CHOOSE DESIRED HELIX TYPE BELOW:

TYPE	ANGLE	CODE
SINGLE	15-25	1
DOUBLE	25-50	2

ENTER HELIX CODE:

?
2

84

WILL ANY POWER SOURCE BE A MULTICYLINDER I. C. ENGINE? (Y OR N):

n

ENTER POWER AND SPEED OF THE POWER SOURCE (HP, RPM):

?
8250, 3600

ENTER OUTPUT SHAFT/PROPELLER SPEED (RPM):

?
155

WHICH DIAMETRAL PITCH WILL YOU SPECIFY?
(1=TRANSVERSE, 2=NORMAL):

?
2

WHICH PRESSURE ANGLE WILL YOU SPECIFY?
(1=TRANSVERSE, 2=NORMAL):

?
2

THE FOLLOWING PARAMETERS ARE REQUESTED FOR STAGE 1 :

* ENTER HELIX ANGLE (DEGREES) :

?
25

* ENTER NORMAL DIAMETRAL PITCH:

?
8

* ENTER NORMAL PRESSURE ANGLE (DEGREES) :

?
20

* ENTER EPICYCLIC CODE (1=PLANETARY, 2=STAR) :

?
1

* ENTER NUMBER OF PLANET GEARS (3 TO 5) :

?
4

CHOOSE GEAR HARDNESS RANGE BELOW:

BHN	CODE
160 - 200	1
200 - 240	2
240 - 300	3
300 - 360	4
360 - 400	5

ENTER HARDNESS CODES FOR SUN/PLANETS AND RING (HCSUN,HCRING) :
?
4,2

THE FOLLOWING PARAMETERS ARE REQUESTED FOR STAGE 2 :

ENTER HELIX ANGLE (DEGREES) :
?
25

ENTER NORMAL DIAMETRAL PITCH :
?
6

ENTER NORMAL PRESSURE ANGLE (DEGREES) :
?
20

ENTER EPICYCLIC CODE (1=PLANETARY, 2=STAR) :
?
1

ENTER NUMBER OF PLANET GEARS (3 TO 5) :
?
5

CHOOSE GEAR HARDNESS RANGE BELOW:

BHN	CODE
160 - 200	1
200 - 240	2
240 - 300	3
300 - 360	4

360 - 400 5
400 - 640 6

ENTER HARDNESS CODES FOR SUN/PLANETS AND RING (HCSUN,HCRING):
?
4,2

TO MAKE CORRECTIONS TO DATA JUST ENTERED, THE PROGRAM MUST BE
ABORTED AND RE-STARTED. DO YOU WISH TO ABORT THIS RUN? (Y OR N):
n

DO YOU DESIRE A LISTING OF THE PRE-PROGRAMMED CONSTANTS USED
IN THE AGMA FORMULATIONS? (Y OR N):
n

THE INFORMATION REQUESTED BELOW IS FOR REDUCTION STAGE 1.

ENTER DIAMETERS, IN INCHES, OF SUN, PLANET, AND RING GEARS
(DS, DPLN, DR):
?
12.55, 18.76, 50.34

ENTER FACEWIDTH OF GEARS, IN INCHES:
?
19.13

THE INFORMATION REQUESTED BELOW IS FOR REDUCTION STAGE 2.

ENTER DIAMETERS, IN INCHES, OF SUN, PLANET, AND RING GEARS
(DS, DPLN, DR):
?
22.99, 30.16, 83.67

ENTER FACEWIDTH OF GEARS, IN INCHES:
?
27.57

POWER SOURCE: TURBINE OR MOTOR

INPUT POWER (HP): 8250. INPUT SPEED (RPM): 3600.

ARRANGEMENT: EPICYCLIC, 2 REDUCTION (S)

OUTPUT POWER (HP): 8250. OUTPUT SPEED (RPM): 155.
RATIO: 23.226 OUTPUT TORQUE (K IN-LB): 3356.5

SIZING ESTIMATES FOR THE ENTIRE REDUCTION SET:

WEIGHT (LB): 65400. SPECIFIC WEIGHT (LB/HP): 7.93
LENGTH (IN): 133 WIDTH (IN): 109 HEIGHT (IN): 100

GEAR ARRANGEMENT

	REDUCTION 1		
	SUN	PLANETS	RING-CAGE
NUMBER OF PLANETS			
POWER SPLIT	8250.	2063.	8250.
SPEED	3600.	1928.	718.
NUMBER OF TEETH	91	136	365
NORMAL DIAMETRAL PITCH		8.000	
TRANS. DIAMETRAL PITCH		7.250	
NORMAL PRESSURE ANGLE		20.0	
TRANS. PRESSURE ANGLE		21.9	

HELIX ANGLE				25.0			
GEAR RATIO				5.011			
PITCH DIAMETER	IN			18.76			50.34
EFFECTIVE FACEWIDTH	IN			19.13			
F/DP				1.52			
CENTER DISTANCE	IN			15.65			
PITCHLINE VELOCITY	FPM			11828.			
TANGENTIAL LOAD	LB			23017.			
TOOTH LOAD/IN	LB/IN			1203.			
UNIT LOAD	PSI			9626.			
MESH FREQUENCY	HZ			1928.			2874.
K FACTOR (COMPUTED)				160.			
CONTACT STRESS	PSI			62354.			
BENDING STRESS	PSI			26284.			
TORQUE	K IN-LB			215.9			723.5
HARDNESS RANGE	BHN			300 - 360			200 - 240

此圖係根據各參數計算所得之結果，其計算過程及所用之公式，均詳列於後。

GEAR ARRANGEMENT	REDUCTION 2		RING-CAGE
	SUN	PLANETS	
NUMBER OF PLANETS		PLANETARY	
POWER SPLIT	8250.	1650.	8250.
SPEED	718.	430.	155.
NUMBER OF TEETH	125	164	455
NORMAL DIAMETRAL PITCH		6.000	
TRANS. DIAMETRAL PITCH		5.438	
NORMAL PRESSURE ANGLE		20.0	
TRANS. PRESSURE ANGLE		21.9	
HELIX ANGLE		25.0	
GEAR RATIO		4.639	
PITCH DIAMETER	22.99	30.16	83.67
EFFECTIVE FACEWIDTH		27.57	

Results from Design Session

ENTER SEED FOR RANDOM NUMBER GENERATOR (X.XX):

?

0.076

POWER SOURCE: TURBINE OR MOTOR

INPUT POWER (HP): 8250. INPUT SPEED (RPM): 3600.

ARRANGEMENT: EPICYCLIC, 2 REDUCTION (S)

OUTPUT POWER (HP): 8250. OUTPUT SPEED (RPM): 155.
 RATIO: 23.093 OUTPUT TORQUE (K IN-LB): 3334.1

SIZING ESTIMATES FOR THE ENTIRE REDUCTION SET:

WEIGHT (LB): 66300. SPECIFIC WEIGHT (LB/HP): 8.04
 LENGTH (IN): 154 WIDTH (IN): 92 HEIGHT (IN): 85

GEAR ARRANGEMENT

	REDUCTION 1		RING-CAGE
	SUN	PLANETS	
NUMBER OF PLANETS		4	
POWER SPLIT	8250.	2063.	8250.
SPEED	3600.	1786.	684.
NUMBER OF TEETH	98	160	418
NORMAL DIAMETRAL PITCH		8.000	
TRANS. DIAMETRAL PITCH		7.250	
NORMAL PRESSURE ANGLE		20.0	

TRANS. PRESSURE ANGLE				21.9	
HELIX ANGLE				25.0	
GEAR RATIO				5.265	
PITCH DIAMETER	IN			22.07	57.65
EFFECTIVE FACEWIDTH	IN	13.54		14.52	
F/DP				1.07	
CENTER DISTANCE	IN			17.80	
PITCHLINE VELOCITY	FPM			12762.	
TANGENTIAL LOAD	LB			21333.	
TOOTH LOAD/IN	LB/IN			1470.	
UNIT LOAD	PSI			11756.	
MESH FREQUENCY	HZ	11665.		1786.	2735.
K FACTOR (COMPUTED)				175.	
CONTACT STRESS	PSI			59881.	
BENDING STRESS	PSI			32049.	
TORQUE	K IN-LB	144.4		235.4	760.2
HARDNESS RANGE	BHN	300 - 360	300 - 360		200 - 240

此項設計係根據標準齒輪設計手冊之規定而得之結果

		REDUCTION 2		
		SUN	PLANETS	RING-CAGE

GEAR ARRANGEMENT			PLANETARY	
NUMBER OF PLANETS			5	
POWER SPLIT	HP	8250.	1650.	8250.
SPEED	RPM	684.	442.	156.
NUMBER OF TEETH		114	136	386
NORMAL DIAMETRAL PITCH			6.000	
TRANS. DIAMETRAL PITCH			5.438	
NORMAL PRESSURE ANGLE			20.0	
TRANS. PRESSURE ANGLE			21.9	
HELIX ANGLE			25.0	
GEAR RATIO			4.386	
PITCH DIAMETER	IN	21.03	25.01	70.98

EFFECTIVE FACEWIDTH	IN		39.42	
F/DP			1.87	
CENTER DISTANCE	IN		23.02	
PITCHLINE VELOCITY	FPM		3764.	
TANGENTIAL LOAD	LB		72331.	
TOOTH LOAD/IN	LB/IN		1835.	
UNIT LOAD	PSI		11009.	
MESH FREQUENCY	HZ		442.	
K FACTOR (COMPUTED)			779.	
CONTACT STRESS	PSI		161.	
BENDING STRESS	PSI		53704.	
TORQUE	K IN-LB		30069.	
HARDNESS RANGE	BHN		904.5	
			300 - 360	
			300 - 360	
			760.5	
			300 - 360	
			300 - 360	
			3334.1	
			200 - 240	

此表係根據試驗結果計算所得之數據，其數值係根據試驗結果計算所得之數據，其數值係根據試驗結果計算所得之數據。


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COMMON /AGMAB/ SFB (2,2) , AKV, AKS, AKM, AKO (2) , SAT (6) , AKL (2) , AKR (6) , AK 1MOD0240
1T 1MOD0250
1MOD0260
1MOD0270
1MOD0280
1MOD0290
1MOD0300
1MOD0310
1MOD0320
1MOD0330
1MOD0340
1MOD0350
1MOD0360
1MOD0370
COMMON /AGMAH/ SFH (2,2) , CV (3) , CS, CM (2) , CF, CO (2) , SAC (6) , CP, CL (2) , CH 1MOD0380
1,CT,CR (6) 1MOD0390
1MOD0400
1MOD0410
1MOD0420
1MOD0430
1MOD0440
1MOD0450
1MOD0460
1MOD0470
1MOD0480
1MOD0490
1MOD0500
1MOD0510
1MOD0520
1MOD0530
1MOD0540
COMMON /DES DAT/ PWRIN (2) , RPMIN (2) , RPMOUT, DHELIX (3) , HELIX (3) , PD (3) , 1MOD0550
1PND (3) , DPHI (3) , PHI (3) , DPHIN (3) , PHIN (3) , NDIFP, IARR, IEPIC (3) , IHARD (3) 1MOD0560
2,2) , IOPRO, NPWRIN, IPWRSR (2) , NRED, NPATH, NPLNT (3) , NHELX 1MOD0570
1MOD0580
COMMON BLOCK DES DAT (DESIGN PARAMETERS, INPUT) 1MOD0590

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C DHELIX : R⁴4 ARRAY (3); HELIX ANGLE (DEG) 1MOD0600
C DPHI : R⁴4 ARRAY (3); TRANSVERSE PRESSURE ANGLE (DEG) 1MOD0610
C DPHIN : R⁴4 ARRAY (3); NORMAL PRESSURE ANGLE (DEG) 1MOD0620
C HELIX : R⁴4 ARRAY (3); HELIX ANGLE (RAD) 1MOD0630
C IARR : I⁴4; ARRANGEMENT CODE (1=PARALLEL AXIS, 2=EPI-1MOD0640
C CYCLIC) 1MOD0650
C NDIFP : I⁴4; NUMBER OF DIFFERENT POWER SOURCES 1MOD0660
C IEPIC : I⁴4 ARRAY (3); EPICYCLIC CODE (1=PLANETARY, 2=STAR) 1MOD0670
C IHARD : I⁴4 ARRAY (3,2); HARDNESS RANGE CODE (1-6, SEE SUBR. AGMA) 1MOD0680
C IOPRO : I⁴4; OPERATIONAL PROFILE CODE (1=NAVAL PROFILE 1MOD0690
C FULL POWER 5% MAX; 2=OTHER, MAX POWER 1MOD0700
C CONTINUOUS) 1MOD0710
C IPWRSR : I⁴4 ARRAY (2); POWER SOURCE CODE (1=TURBINE OR MOTOR, 2=1MOD0720
C MULTICYLINDER INTERNAL COMBUSTION ENGINE) 1MOD0730
C NHELY : I⁴4; NUMBER OF HELICIES (1=SINGLE, 2=DOUBLE) 1MOD0740
C NPATH : I⁴4; NUMBER OF POWER PATHS (1=SINGLE, 2=DUAL) 1MOD0750
C NPLNT : I⁴4 ARRAY (3); NUMBER OF PLANET GEARS IN EPICYCLIC SET 1MOD0760
C NPWRIN : I⁴4; NUMBER OF POWER SOURCES (INPUTS) 1MOD0770
C NRED : I⁴4; NUMBER OF REDUCTION STAGES 1MOD0780
C PD : R⁴4 ARRAY (3); TRANSVERSE DIAMETRAL PITCH 1MOD0790
C PHI : R⁴4 ARRAY (3); TRANSVERSE PRESSURE ANGLE (RAD) 1MOD0800
C PHIN : R⁴4 ARRAY (3); NORMAL PRESSURE ANGLE (RAD) 1MOD0810
C PND : R⁴4 ARRAY (3); NORMAL DIAMETRAL PITCH 1MOD0820
C PWRIN : R⁴4 ARRAY (2); SOURCE POWER INPUT (HP) 1MOD0830
C RPMIN : R⁴4 ARRAY (2); SOURCE SPEED INPUT (RPM) 1MOD0840
C RPMOUT : R⁴4; OUTPUT SHAFT/PROPELLER SPEED (RPM) 1MOD0850
C 1MOD0860
COMMON /DESPRL/ PWRFAC (2,3), MGOP (2), MGP (3,2), RPMP (6,2), PWRP (6,2), D1MOD0870
1P (3,2), DG (3,2), FACEP (3,2), GEOMI (3,2), GEOMJP (3,2), GEOMJG (3,2), NP (3,1MOD0880
22), NG (3,2) 1MOD0890
C 1MOD0900
C ⁴ COMMON BLOCK DESPRL (PARALLEL AXIS DESIGN PARAMETERS) 1MOD0910
C DG : R⁴4 ARRAY (3,2); DIAMETER OF GEAR (IN) 1MOD0920
C DP : R⁴4 ARRAY (3,2); DIAMETER OF PINION (IN) 1MOD0930
C FACEP : R⁴4 ARRAY (3,2); FACEWIDTH (IN) 1MOD0940
C GEOMI : R⁴4 ARRAY (3,2); DURIBILITY GEOMETRY FACTOR 1MOD0950

C GEOMJG: R**4 ARRAY (3,2); STRENGTH GEOMETRY FACTOR (GEAR) 1MOD0960
 C GEOMJP: R**4 ARRAY (3,2); STRENGTH GEOMETRY FACTOR (PINION) 1MOD0970
 C MGOP : R**4 ARRAY (2); OVERALL REDUCTION RATIO 1MOD0980
 C MGP : R**4 ARRAY (3,2); STAGE REDUCTION RATIO 1MOD0990
 C NG : I**4 ARRAY (3,2); NUMBER OF TEETH, GEAR 1MOD1000
 C NP : I**4 ARRAY (3,2); NUMBER OF TEETH, PINION 1MOD1010
 C PWRFAC: R**4 ARRAY (2,3); STAGE POWER SPLIT FACTOR 1MOD1020
 C PWRP : R**4 ARRAY (6,2); STAGE POWER SPLIT PER GEAR (HP) 1MOD1030
 C RPMP : R**4 ARRAY (6,2); STAGE PINION AND GEAR SPEED (RPM) 1MOD1040
 C 1MOD1050
 C COMMON /RESPRL/ PLVP (3,2), FBYDP (3,2), CDP (3,2), WTP (6,2), TLPPI (6,2), 1MOD1060
 1UNTLDP (6,2), MFP (3,2), KFCTRP (6,2), SIGHP (3,2), SIGBP (6,2), TORQP (6,2), 1MOD1070
 2PDIAMP (6,2), SCDMIN, SCDMAX, SHP, WGHTP, SPCWTP, MTHP (6,2), ISIZEP (3) 1MOD1080
 C 1MOD1090
 C *** COMMON BLOCK RESPRL (PARALLEL AXIS PARAMETERS, RESULTS) 1MOD1100
 C CDP : R**4 ARRAY (3,2); CENTER DISTANCE (THEORETICAL) (IN) 1MOD1110
 C FBYDP : R**4 ARRAY (3,2); F/D RATIO (FACEWIDTH/PINION DIAMETER) 1MOD1120
 C ISIZEP: I**4 ARRAY (3); LENGTH, WIDTH, HEIGHT ESTIMATES (IN) 1MOD1130
 C KFCTRP: R**4 ARRAY (6,2); COMPUTED K-FACTOR 1MOD1140
 C MFP : R**4 ARRAY (3,2); MESH FREQUENCY (HZ) 1MOD1150
 C MTHP : I**4 ARRAY (6,2); TOOTH NUMBERS 1MOD1160
 C PDIAMP: R**4 ARRAY (6,2); PITCH DIAMETERS (IN) 1MOD1170
 C PLVP : R**4 ARRAY (3,2); PITCH LINE VELOCITY (FPM) 1MOD1180
 C SCDMAX: R**4; MAXIMUM SOURCE CENTER DISTANCE (IN) 1MOD1190
 C SCDMIN: R**4; MINIMUM SOURCE CENTER DISTANCE (IN) 1MOD1200
 C SHP : R**4; SHAFT HORSEPOWER, OUTPUT (HP) 1MOD1210
 C SIGBP : R**4 ARRAY (6,2); BENDING STRESS (PSI) 1MOD1220
 C SIGHP : R**4 ARRAY (3,2); CONTACT STRESS (PSI) 1MOD1230
 C SPCWTP: R**4; SPECIFIC WEIGHT (LB/HP) 1MOD1240
 C TLPPI : R**4 ARRAY (6,2); TOOTH LOAD PER INCH (LB/IN) 1MOD1250
 C TORQP : R**4 ARRAY (6,2); TORQUE (K IN-LB) 1MOD1260
 C UNTLDP: R**4 ARRAY (6,2); UNIT LOAD (PSI) 1MOD1270
 C WGHTP : R**4; GEAR SET WEIGHT ESTIMATE (LB) 1MOD1280
 C WTP : R**4 ARRAY (6,2); TANGENTIAL TOOTH LOAD (LB) 1MOD1290
 C 1MOD1300
 C COMMON /DESEPC/ MGOE, MGE (3), RPMI (3), RPMPL (3), RPMO (3), PWRE (3), DS (3) 1MOD1310

1, DPLN(3), DR(3), FACEE(3), GI(3), GJS(3), GJPL(3), NS(3), NPLN(3), NR(3)
1 MOD 1320
1 MOD 1330
1 MOD 1340
1 MOD 1350
1 MOD 1360
1 MOD 1370
1 MOD 1380
1 MOD 1390
1 MOD 1400
1 MOD 1410
1 MOD 1420
1 MOD 1430
1 MOD 1440
1 MOD 1450
1 MOD 1460
1 MOD 1470
1 MOD 1480
1 MOD 1490
1 MOD 1500
1 MOD 1510
1 MOD 1520
1 MOD 1530
1 MOD 1540
1 MOD 1550
1 MOD 1560
1 MOD 1570
1 MOD 1580
1 MOD 1590
1 MOD 1600
1 MOD 1610
1 MOD 1620
1 MOD 1630
1 MOD 1640
1 MOD 1650
1 MOD 1660
1 MOD 1670

C COMMON BLOCK DESEPC (EPICYCLIC DESIGN PARAMETERS)
C DPLN : R*4 ARRAY (3); DIAMETER OF PLANET GEARS (IN)
C DR : R*4 ARRAY (3); DIAMETER OF RING GEAR (IN)
C DS : R*4 ARRAY (3); DIAMETER OF SUN GEAR (IN)
C FACEE : R*4 ARRAY (3); FACEWIDTH (IN)
C GI : R*4 ARRAY (3); DURABILITY GEOMETRY FACTOR (SUN/PLANETS)
C GJS : R*4 ARRAY (3); STRENGTH GEOMETRY FACTOR (SUN)
C GJPL : R*4 ARRAY (3); STRENGTH GEOMETRY FACTOR (PLANET)
C MGE : R*4 ARRAY (3); STAGE REDUCTION RATIO
C MGOE : R*4;
C NPLN : I*4 ARRAY (3); OVERALL REDUCTION RATIO
C NR : I*4 ARRAY (3); NUMBER OF TEETH, PLANET
C NS : I*4 ARRAY (3); NUMBER OF TEETH, RING
C PWRE : R*4 ARRAY (3); STAGE POWER SPLIT PER GEAR PAIR (HP)
C RPMI : R*4 ARRAY (3); STAGE INPUT SPEED (RPM)
C RPMO : R*4 ARRAY (3); STAGE OUTPUT SPEED (RPM)
C RPMPL : R*4 ARRAY (3); PLANET SPEED (RPM)
C

COMMON /RESEPC/ PLVE(3), FBYDE(3), CDE(3), WTE(3), TLP(3), UNTLDE(3),
1MFE(3,3), KFCTRE(3), SIGHE(3), SIGBE(3), TORQE(3,3), RPME(3,3), PDIAAME(3,
2,3), WGHTE, SPCWTE, MTHE(3,3), ISIZEE(3)
1 MOD 1520
1 MOD 1530
1 MOD 1540
1 MOD 1550
1 MOD 1560
1 MOD 1570
1 MOD 1580
1 MOD 1590
1 MOD 1600
1 MOD 1610
1 MOD 1620
1 MOD 1630
1 MOD 1640
1 MOD 1650
1 MOD 1660
1 MOD 1670

C COMMON BLOCK RESEPC (EPICYCLIC PARAMETERS, RESULTS)
C CDE : R*4 ARRAY (3); CENTER DISTANCE (THEORETICAL) (IN)
C FBYDE : R*4 ARRAY (3); F/D RATIO (FACEWIDTH/SUN DIAMETER)
C ISIZEE: I*4 ARRAY (3); LENGTH, WIDTH, HEIGHT ESTIMATES (IN)
C KFCTRE: R*4 ARRAY (3); COMPUTED K-FACTOR
C MFE : R*4 ARRAY (3,3); MESH FREQUENCY (HZ)
C MTHE : I*4 ARRAY (3,3); TOOTH NUMBERS
C PDIAAME: R*4 ARRAY (3,3); PITCH DIAMETERS (IN)
C PLVE : R*4 ARRAY (3); PITCH LINE VELOCITY (FPM)
C RPME : R*4 ARRAY (3,3); GEAR SPEEDS (RPM)
C SIGBE : R*4 ARRAY (3); BENDING STRESS (PSI)
C SIGHE : R*4 ARRAY (3); CONTACT STRESS (PSI)
C


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C      SPCWTE: R*4;
C      TLPTE : R*4 ARRAY (3);
C      TORQE : R*4 ARRAY (3,3);
C      UNTLDE: R*4 ARRAY (3);
C      WGHTE : R*4;
C      WTE   : R*4 ARRAY (3);
C
C      EXECUTE REGAD
C
C      DATA YES/1HY/
C      WRITE (6,90)
C      WRITE (6,100)
C      READ (5,130) REP
C      IF (REP.EQ.YES) CALL DSCRPT
C      WRITE (6,120)
C
C      DESIGN / ANALYSIS OPTION SELECTION
C
C      WRITE (6,70)
C      READ (5,*) ICODE
C      IF (ICODE.LT.1) ICODE=1
C      IF (ICODE.GT.2) ICODE=2
C
C      CONFIGURATION SELECTION
C
C      WRITE (6,80)
C      READ (5,*) IARR
C      IF (IARR.LT.1) IARR=1
C      IF (IARR.GT.2) IARR=2
C      CALL INPUT
C      WRITE (6,110)
C      READ (5,130) REP
C      IF (REP.EQ.YES) CALL AGMA
C      L=ICODE+(IARR-1)*2
C      GO TO (10,20,40,50), L

```

```

1MOD1680
1MOD1690
1MOD1700
1MOD1710
1MOD1720
1MOD1730
1MOD1740
1MOD1750
1MOD1760
1MOD1770
1MOD1780
1MOD1790
1MOD1800
1MOD1810
1MOD1820
1MOD1830
1MOD1840
1MOD1850
1MOD1860
1MOD1870
1MOD1880
1MOD1890
1MOD1900
1MOD1910
1MOD1920
1MOD1930
1MOD1940
1MOD1950
1MOD1960
1MOD1970
1MOD1980
1MOD1990
1MOD2000
1MOD2010
1MOD2020
1MOD2030

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SPECIFIC WEIGHT (LB/HP)
TOOTH LOAD PER IN (LB/IN)
TORQUE (K IN-LB)
UNIT LOAD (PSI)
GEAR SET WEIGHT ESTIMATE (LB)
TANGENTIAL TOOTH LOAD (LB)

```



```

CALL EPCOUT
STOP
C
C
C
C
C
70
80
90
100
110
120
130
C
C
C
BLOCK DATA
REAL MGOP,MGP,MGOE,MGE,MFP,MFE,KFCTRP,KFCTRE
COMMON /AGMAH/ SFH(2,2),CV(3),CS,CM(2),CF,CO(2),SAC(6),CF,CL(2),CH
1,CT,CR(6)
COMMON /AGMAB/ SFB(2,2),AKV,AKS,AKM,AKO(2),SAT(6),AKL(2),AKR(6),AK
1T
COMMON /DESDAT/ PWRIN(2),RPMIN(2),RPMOUT,DHELIX(3),HELIX(3),PD(3),
1PND(3),DPHI(3),PHI(3),DPHIN(3),PHIN(3),NDIFF,IARR,IEPIC(3),IHARD(3
2,2),IOPRO,NPWRIN,IPWRSR(2),NRED,NPATH,NPLNT(3),NHELX
COMMON /DESPRL/ PWRFAC(2,3),MGOP(2),MGP(3,2),RPMP(6,2),PWRP(6,2),
1P(3,2),DG(3,2),FACEP(3,2),GEOMI(3,2),GEOMJP(3,2),GEOMJG(3,2),NP(3,
22),NG(3,2)
COMMON /RESPRL/ PLVP(3,2),FBYDIP(3,2),CDP(3,2),WTP(6,2),TLPIP(6,2),
1MOD2400
1MOD2410
1MOD2420
1MOD2430
1MOD2440
1MOD2450
1MOD2460
1MOD2470
1MOD2480
1MOD2490
1MOD2500
1MOD2510
1MOD2520
1MOD2530
1MOD2540
1MOD2550
1MOD2560
1MOD2570
1MOD2580
1MOD2590
1MOD2600
1MOD2610
1MOD2620
1MOD2630
1MOD2640
1MOD2650
1MOD2660
1MOD2670
1MOD2680
1MOD2690
1MOD2700
1MOD2710
1MOD2720
1MOD2730
1MOD2740
1MOD2750

```



```

WRITE (6,60)
READ (5,80) REP
IF (REP.EQ.YES) RETURN
C
C STOP BY USER
C
WRITE (6,70)
STOP
C
C FORMAT STATEMENTS
C
C
C
C
10 FORMAT (1H1,65(1H*),/4X,59H THIS PROGRAM IS CAPABLE OF PERFORMING1MOD3240
1 PRELIMINARY DESIGN /4X,59HOR ANALYSIS OF MULTIREDUCTION, PARALLEL1MOD3250
2 AXIS AND EPICYCLIC /4X,59HREDUCTION GEARS. THE CAPABILITIES AND 1MOD3260
3 FEATURES OF THE PRO-/4X,20HGRAM ARE AS FOLLOWS:.)
1MOD3270
20 FORMAT (/9X,44H1) MAXIMUM OF THREE REDUCTION STAGES ALLOWED/9X,38H1MOD3280
12) CHOICE OF SINGLE OR DOUBLE HELICALS/9X,37H3) WEIGHT AND SIZE ES1MOD3290
2TIMATES PROVIDED/9X,27H4) FOR PARALLEL AXIS GEARS:/12X,34H- ONE OR1MOD3300
3 TWO POWER SOURCES ALLOWED/12X,36H- SINGLE OR DUAL POWER PATHS ALL1MOD3310
4OWED)
1MOD3320
30 FORMAT (9X,23H5) FOR EPICYCLIC GEARS:/12X,31H- ONLY ONE POWER SOUR1MOD3330
1CE ALLOWED/12X,36H- LIMITED TO 3, 4, OR 5 PLANET GEARS/12X,44H- ON1MOD3340
2LY SIMPLE EPICYCLICS PER REDUCTION STAGE/12X,41H- PLANETARY OR STA1MOD3350
3R ARRANGEMENTS POSSIBLE)
1MOD3360
40 FORMAT (/4X,59H THE STANDARDS OF THE AMERICAN GEAR MANUFACTURING 1MOD3370
1ASSOCI-/4X,59HATION WERE USED AS A BASIS FOR THIS PROGRAM. THE C1MOD3380
2ONSTANTS/4X,59HUSED IN THE AGMA FORMULATIONS ARE BASED ON THOSE PU1MOD3390
3BLISHED /4X,59HBY F. A. THOMA, OF DELAVAL TURBINE, FOR MARINE PROP1MOD3400
4ULSION /4X,59HGEARS. AN OPTION IS PROVIDED DURING EXECUTION OF T1MOD3410
5HE PRO- /4X,59HGRAM TO OBTAIN A LISTING OF THESE CONSTANTS, AND TO1MOD3420
6 CHANGE /4X,44HANY OF THEM FOR OTHER POSSIBLE APPLICATIONS./4X,59H1MOD3430
7 IT SHOULD BE NOTED THAT THE STRESSES LISTED IN THE OUTPUT/4X,59H1MOD3440
8ARE THOSE COMPUTED FROM THE AGMA FORMULATIONS AND ARE NOT /4X,32H1MOD3450
9FROM A DETAILED STRESS ANALYSIS.)
1MOD3460
50 FORMAT (//,4X,54HFOR MORE SPECIFIC INFORMATION, SEE THE USERS MANU1MOD3470

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1 AL OR,/,4X,32HOBTAI A LISTING OF THE PROGRAM.,/,1X,65 (1H*)
60 FORMAT (/,4X,53HDO YOU WISH THE PROGRAM TO CONTINUE INTO THE ANAL
1 YSIS,/,4X,30HAND DESIGN SEGMENTS? (Y OR N):)
70 FORMAT (/,/,5X,35HPROGRAM STOPPED BY USER
80 FORMAT (1A1)
END
SUBROUTINE INPUT
C
C CODED BY: LT J.L. PAQUETTE, USN JAN 1982
C NAVAL POSTGRADUATE SCHOOL MONTEREY, CA 93940
C
C SUBPROGRAM TO PROVIDE USER INPUT OF ALL USER-SPECIFIED DESIGN
C VARIABLES AND OPTIONS
C
COMMON /DESDAT/ PWRIN(2), RPMIN(2), RPMOUT,DHELIX(3), HELIX(3), PD(3),
1PND(3), DPHI(3), PHI(3), DPHIN(3), PHIN(3), NDIFF,IARR,IEPIC(3), IHARD(3)
2,2), IOPRO,NPWRIN,IPWRSR(2),NRED,NPATH,NPLNT(3),NHELX
C
C INITIALIZE
C
DATA YES/1HY/
DEGRAD=4.*ATAN(1.)/180.
C
COMMON DATA
C
WRITE (6,230)
READ (5,*) IOPRO
IF (IOPRO.LT.1) IOPRO=1
IF (IOPRO.GT.2) IOPRO=2
WRITE (6,240)
READ (5,*) NRED
IF ((NRED.GE.1).AND.(NRED.LE.3)) GO TO 20
WRITE (6,250) NRED
10
1 MOD3480
1 MOD3490
1 MOD3500
1 MOD3510
1 MOD3520
1 MOD3530
1 MOD3540
1 MOD3550
1 MOD3560
1 MOD3570
1 MOD3580
1 MOD3590
1 MOD3600
1 MOD3610
1 MOD3620
1 MOD3630
1 MOD3640
1 MOD3650
1 MOD3660
1 MOD3670
1 MOD3680
1 MOD3690
1 MOD3700
1 MOD3710
1 MOD3720
1 MOD3730
1 MOD3740
1 MOD3750
1 MOD3760
1 MOD3770
1 MOD3780
1 MOD3790
1 MOD3800
1 MOD3810
1 MOD3820
1 MOD3830

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20	GO TO 10	1MOD3840
	WRITE (6,260)	1MOD3850
	READ (5,*) NHELX	1MOD3860
	IF (NHELX.LT.1) NHELX=1	1MOD3870
	IF (NHELX.GT.2) NHELX=2	1MOD3880
	GO TO (30,40), IARR	1MOD3890
C		1MOD3900
C	PARALLEL AXIS DATA	1MOD3910
C		1MOD3920
30	WRITE (6,270)	1MOD3930
	READ (5,*) NPATH	1MOD3940
	IF (NPATH.LT.1) NPATH=1	1MOD3950
	IF (NPATH.GT.2) NPATH=2	1MOD3960
	GO TO 50	1MOD3970
C		1MOD3980
C	EPICYCLIC DATA	1MOD3990
C		1MOD4000
40	NPWRIN=1	1MOD4010
	NPATH=2	1MOD4020
	GO TO 60	1MOD4030
C		1MOD4040
C	COMMON DESIGN DATA	1MOD4050
C		1MOD4060
C		1MOD4070
C	POWER SOURCE DATA	1MOD4080
C		1MOD4090
50	WRITE (6,280)	1MOD4100
	READ (5,*) NPWRIN	1MOD4110
	IF (NPWRIN.LT.1) NPWRIN=1	1MOD4120
	IF (NPWRIN.GT.2) NPWRIN=2	1MOD4130
60	WRITE (6,290)	1MOD4140
	READ (5,550) REP1	1MOD4150
	GO TO (70,80), NPWRIN	1MOD4160
C		1MOD4170
C	SINGLE POWER SOURCE	1MOD4180
C		1MOD4190


```

70 IF (REP1.EQ.YES) IPWRSR(1)=2
WRITE (6,300)
READ (5,*) PWRIN(1),RPMIN(1)
GO TO 110
C
C DUAL POWER SOURCES
C
80 IF (REP1.NE.YES) GO TO 100
WRITE (6,310)
READ (5,*) IENG
IF ((IENG.GE.1).AND.(IENG.LE.3)) GO TO 90
WRITE (6,320) IENG
GO TO 80
90 IF ((IENG.EQ.1).OR.(IENG.EQ.3)) IPWRSR(1)=2
IF ((IENG.EQ.2).OR.(IENG.EQ.3)) IPWRSR(2)=2
100 WRITE (6,330)
READ (5,*) PWRIN(1),RPMIN(1)
WRITE (6,340)
READ (5,*) PWRIN(2),RPMIN(2)
C
C OUTPUT SPEED OF REDUCTION SET
C
C
110 NDIFP=NPWRIN
IF ((NPWRIN.EQ.2).AND.(PWRIN(1).EQ.PWRIN(2)).AND.(RPMIN(1).EQ.RPMI
1N(2))) NDIFP=1
WRITE (6,350)
READ (5,*) RPMOUT
C
C DESIGN PARAMETER DATA
C
WRITE (6,360)
READ (5,*) IPD
IF (IPD.LT.1) IPD=1
IF (IPD.GT.2) IPD=2
WRITE (6,370)
READ (5,*) IPHI
1MOD4200
1MOD4210
1MOD4220
1MOD4230
1MOD4240
1MOD4250
1MOD4260
1MOD4270
1MOD4280
1MOD4290
1MOD4300
1MOD4310
1MOD4320
1MOD4330
1MOD4340
1MOD4350
1MOD4360
1MOD4370
1MOD4380
1MOD4390
1MOD4400
1MOD4410
1MOD4420
1MOD4430
1MOD4440
1MOD4450
1MOD4460
1MOD4470
1MOD4480
1MOD4490
1MOD4500
1MOD4510
1MOD4520
1MOD4530
1MOD4540
1MOD4550

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120 IF (IPHI.LT.1) IPHI=1
    IF (IPHI.GT.2) IPHI=2
    DO 220 I=1,NRED
    WRITE (6,380) I
    WRITE (6,390)
    READ (5,*) DHELIX(I)
    IF ((NHELX.EQ.1).AND.(DHELIX(I).GE.15.0).AND.(DHELIX(I).LE.25.0))
    1GO TO 130
    IF ((NHELX.EQ.2).AND.(DHELIX(I).GE.25.0).AND.(DHELIX(I).LE.50.0))
    1GO TO 130
    WRITE (6,400) DHELIX(I),NHELX
    GO TO 120
130 HELIX(I)=DHELIX(I)*DEGRAD
    GO TO (140,150), IPD
140 WRITE (6,410)
    READ (5,*) PD(I)
    PND(I)=PD(I)/COS(HELIX(I))
    GO TO 160
150 WRITE (6,420)
    READ (5,*) PND(I)
    PD(I)=PND(I)*COS(HELIX(I))
    GO TO (170,180), IPHI
170 WRITE (6,430)
    READ (5,*) DPHI(I)
    PHI(I)=DPHI(I)*DEGRAD
    ARG=TAN(PHI(I))*COS(HELIX(I))
    PHIN(I)=ATAN(ARG)
    DPHIN(I)=PHIN(I)/DEGRAD
    GO TO 190
180 WRITE (6,440)
    READ (5,*) DPHIN(I)
    PHIN(I)=DPHIN(I)*DEGRAD
    ARG=TAN(PHIN(I))/COS(HELIX(I))
    PHI(I)=ATAN(ARG)
    DPHI(I)=PHI(I)/DEGRAD
    IF (IARR.EQ.1) GO TO 210
190

```

```

1MOD4560
1MOD4570
1MOD4580
1MOD4590
1MOD4600
1MOD4610
1MOD4620
1MOD4630
1MOD4640
1MOD4650
1MOD4660
1MOD4670
1MOD4680
1MOD4690
1MOD4700
1MOD4710
1MOD4720
1MOD4730
1MOD4740
1MOD4750
1MOD4760
1MOD4770
1MOD4780
1MOD4790
1MOD4800
1MOD4810
1MOD4820
1MOD4830
1MOD4840
1MOD4850
1MOD4860
1MOD4870
1MOD4880
1MOD4890
1MOD4900
1MOD4910

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WRITE (6,450)
READ (5,*) IEPIC(I)
IF (IEPIC(I).LT.1) IEPIC(I)=1
IF (IEPIC(I).GT.2) IEPIC(I)=2
200 WRITE (6,460)
READ (5,*) NPLNT(I)
IF ((NPLNT(I).GE.3).AND.(NPLNT(I).LE.5)) GO TO 210
WRITE (6,470) NPLNT(I)
GO TO 200
210 WRITE (6,480)
IF (IARR.EQ.1) WRITE (6,490)
IF (IARR.EQ.2) WRITE (6,500)
READ (5,*) IHARD(I,1), IHARD(I,2)
IF ((IHARD(I,1).GE.1).AND.(IHARD(I,1).LE.6).AND.(IHARD(I,2).GE.1).
1AND.(IHARD(I,2).LE.6)) GO TO 220
IF (IARR.EQ.1) WRITE (6,510) IHARD(I,1), IHARD(I,2)
IF (IARR.EQ.2) WRITE (6,520) IHARD(I,1), IHARD(I,2)
GO TO 210
220 CONTINUE
C DATA CORRECTION
C
C WRITE (6,530)
READ (5,550) REP
IF (REP.NE.YES) RETURN
WRITE (6,540)
STOP
C FORMAT STATEMENTS
C
C
C 230 FORMAT (//,4X,38HCHOOSE OPERATIONAL PROFILE CODE BELOW:,,4X,16HOP1MOD5230
1ERATIONAL MODE,4X,15HSERVICE PROFILE,4X,4HCODE,/,7X,10HFULL POWER,1MOD5240
28X,13H5 PERCENT MAX,7X,1H1,/,6X,12HMAXIMUM LOAD,9X,10HCONTINUOUS,81MOD5250
3X,1H2,/,1X,34H*** ENTER OPERATIONAL PROFILE CODE:)
240 FORMAT (//,1X,43H*** ENTER NUMBER OF REDUCTIONS (1, 2, OR 3):)
1MOD4920
1MOD4930
1MOD4940
1MOD4950
1MOD4960
1MOD4970
1MOD4980
1MOD4990
1MOD5000
1MOD5010
1MOD5020
1MOD5030
1MOD5040
1MOD5050
1MOD5060
1MOD5070
1MOD5080
1MOD5090
1MOD5100
1MOD5110
1MOD5120
1MOD5130
1MOD5140
1MOD5150
1MOD5160
1MOD5170
1MOD5180
1MOD5190
1MOD5200
1MOD5210
1MOD5220
1MOD5230
1MOD5240
1MOD5250
1MOD5260
1MOD5270

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250 FORMAT (4X,I2,4,2H IS NOT A LEGITIMATE NUMBER OF REDUCTIONS.) 1MOD5280
 260 FORMAT (//,4X,32HCHOOSE DESIRED HELIX TYPE BELOW:,,/8X,6H TYPE ,4X1MOD5290
 1,5HANGLE,4X,4HCODE,/,8X,6HSINGLE,4X,5H15-25,6X,1H1,/,8X,6HDOUBLE,41MOD5300
 2X,5H25-50,6X,1H2,/,1X,20H** ENTER HELIX CODE:)
 270 FORMAT (//,1X,50H** ENTER NUMBER OF POWER PATHS (1=SINGLE, 2=DUAL):1MOD5320
 1) 1MOD5330
 280 FORMAT (//,1X,42H** ENTER NUMBER OF POWER SOURCES (1 OR 2):) 1MOD5340
 290 FORMAT (//,4X,64HWILL ANY POWER SOURCE BE A MULTICYLINDER I. C. ENG1MOD5350
 1INE? (Y OR N):) 1MOD5360
 300 FORMAT (//,1X,54H** ENTER POWER AND SPEED OF THE POWER SOURCE (HP,R1MOD5370
 1PM):) 1MOD5380
 310 FORMAT (//,4X,52HCHOOSE WHICH SOURCE(S) WILL BE AN I.C. ENGINE BEL1MOD5390
 1OW:,,/20X,12HPOWER SOURCE,4X,4HCODE,/,21X,10HHIGH POWER,7X,1H1,/,21MOD5400
 21X,10HLOW POWER,7X,1H2,/,24X,4HBT,10X,1H3,/,1X,26H** ENTER I.C1MOD5410
 3. ENGINE CODE:)
 320 FORMAT (4X,I2,38H IS NOT A LEGITIMATE I.C. ENGINE CODE.) 1MOD5430
 330 FORMAT (//,1X,55H** ENTER POWER AND SPEED OF HIGH POWER SOURCE (HP,1MOD5440
 1RPM):) 1MOD5450
 340 FORMAT (//,1X,54H** ENTER POWER AND SPEED OF LOW POWER SOURCE (HP,R1MOD5460
 1PM):) 1MOD5470
 350 FORMAT (//,1X,44H** ENTER OUTPUT SHAFT/PROPELLER SPEED (RPM):) 1MOD5480
 360 FORMAT (//,4X,39HWICH DIAMETRAL PITCH WILL YOU SPECIFY?,,/4X,25H(1MOD5490
 11=TRANSVERSE, 2=NORMAL):)
 370 FORMAT (//,4X,38HWICH PRESSURE ANGLE WILL YOU SPECIFY?,,/4X,25H(11MOD5510
 1=TRANSVERSE, 2=NORMAL):)
 380 FORMAT (//,4X,48HTHE FOLLOWING PARAMETERS ARE REQUESTED FOR STAGE,1MOD5530
 1I2,2H :)
 390 FORMAT (//,1X,31H** ENTER HELIX ANGLE (DEGREES):) 1MOD5550
 400 FORMAT (//,4X,24HTHE HELIX ANGLE ENTERED,,/F5.1,25H, DOES NOT AGREE1MOD5560
 1 WITH THE,/,4X,11HHELIX TYPE=,I2,36H CHOSEN. TYPE=1, SINGLE: 15-21MOD5570
 25DEG.,/,4X,26HTYPE=2, DOUBLE: 25-50 DEG.) 1MOD5580
 410 FORMAT (//,1X,36H** ENTER TRANSVERSE DIAMETRAL PITCH:)
 420 FORMAT (//,1X,32H** ENTER NORMAL DIAMETRAL PITCH:)
 430 FORMAT (//,1X,45H** ENTER TRANSVERSE PRESSURE ANGLE (DEGREES):)
 440 FORMAT (//,1X,41H** ENTER NORMAL PRESSURE ANGLE (DEGREES):)
 450 FORMAT (//,1X,46H** ENTER EPICYCLIC CODE (1=PLANETARY, 2=STAR):) 1MOD5630


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460 FORMAT (/,1X,41H* ENTER NUMBER OF PLANET GEARS (3 TO 5):) 1MOD5640
470 FORMAT (/,4X,I2,39H IS NOT A LEGITIMATE NUMBER OF PLANETS.) 1MOD5650
480 FORMAT (/,4X,33HCHOOSE GEAR HARDNESS RANGE BELOW:/,9X,3HBHN,8X,41MOD5660
1HCODE,/,6X,9H160 - 200,7X,1H1,/,6X,9H200 - 240,7X,1H2,/,6X,9H240 -1MOD5670
2 300,7X,1H3,/,6X,9H300 - 360,7X,1H4,/,6X,9H360 - 400,7X,1H5,/,6X,91MOD5680
3H400 - 640,7X,1H6) 1MOD5690
490 FORMAT (/,1X,59H* ENTER HARDNESS CODES FOR PINION AND GEAR (HCPIN1MOD5700
1,HCGEAR):) 1MOD5710
500 FORMAT (/,1X,64H* ENTER HARDNESS CODES FOR SUN/PLANETS AND RING (1MOD5720
1HCSUN,HCRING):) 1MOD5730
510 FORMAT (/,4X,25HTHE PINION HARDNESS CODE,,I2,31H AND/OR THE GEAR 1MOD5740
1HARDNESS CODE,/,4X,I2,26H ARE NOT LEGITIMATE CODES.) 1MOD5750
520 FORMAT (/,4X,29HTHE SUN/PLANET HARDNESS CODE,,I2,25H AND/OR THE R1MOD5760
1ING HARDNESS,/,4X,5HCODE,,I2,26H ARE NOT LEGITIMATE CODES.) 1MOD5770
530 FORMAT (/,4X,61HTO MAKE CORRECTIONS TO DATA JUST ENTERED, THE PRO1MOD5780
1GRAM MUST BE,/,4X,65HABORTED AND RE-STARTED. DO YOU WISH TO ABORT1MOD5790
2 THIS RUN? (Y OR N):) 1MOD5800
540 FORMAT (/,5X,44H* RUN ABORTED BY USER --- RE-START * * * * *) 1MOD5810
550 FORMAT (1A1) 1MOD5820
END 1MOD5830
C * * * * * 1MOD5840
C * * * * * 1MOD5850
C * * * * * 1MOD5860
SUBROUTINE AGMA 1MOD5870
1MOD5880
C CODED BY: LT J.L. PAQUETTE, USN JAN 1982 1MOD5890
C NAVAL POSTGRADUATE SCHOOL MONTEREY, CA 93940 1MOD5900
C 1MOD5910
C SUBPROGRAM TO LIST AND OPTIONALLY CHANGE THE PRE-PROGRAMMED 1MOD5920
C AGMA CONSTANTS 1MOD5930
C 1MOD5940
C EXTERNAL SUBPROGRAM(S) REQUIRED: FUNCTION CKDATA 1MOD5950
C 1MOD5960
C COMMON /AGMAB/ SFB(2,2),AKV,AKS,AKM,AKO(2),SAT(6),AKL(2),AKR(6),AK 1MOD5970
1T 1MOD5980
C COMMON /AGMAH/ SFH(2,2),CV(3),CS,CM(2),CF,CO(2),SAC(6),CP,CL(2),CH1MOD5990

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```

WRITE (6,480) I,J
SFB(I,J)=CKDATA(SFB(I,J))
SFB(I,J)=SFB(I,J)
CONTINUE
40 GO TO 20
50 DO 60 I=1,3
WRITE (6,490) I
CV(I)=CKDATA(CV(I))
CONTINUE
60 GO TO 20
70 WRITE (6,500)
CS=CKDATA(CS)
GO TO 20
80 DO 90 I=1,2
WRITE (6,510) I
CM(I)=CKDATA(CM(I))
CONTINUE
90 GO TO 20
100 WRITE (6,520)
CF=CKDATA(CF)
GO TO 20
110 DO 120 I=1,2
WRITE (6,530) I
CO(I)=CKDATA(CO(I))
CONTINUE
120 GO TO 20
130 WRITE (6,540) EP,EG,VP,VG
WRITE (6,550)
READ (5,*) VAL1,VAL2
IF (VAL1.NE.0.0) EP=VAL1
IF (VAL2.NE.0.0) EG=VAL2
WRITE (6,560)
READ (5,*) VAL1,VAL2
IF (VAL1.NE.0.0) VP=VAL1
IF (VAL1.NE.0.0) VG=VAL2
AP=(1.-VP*VP)/EP
1MOD6360
1MOD6370
1MOD6380
1MOD6390
1MOD6400
1MOD6410
1MOD6420
1MOD6430
1MOD6440
1MOD6450
1MOD6460
1MOD6470
1MOD6480
1MOD6490
1MOD6500
1MOD6510
1MOD6520
1MOD6530
1MOD6540
1MOD6550
1MOD6560
1MOD6570
1MOD6580
1MOD6590
1MOD6600
1MOD6610
1MOD6620
1MOD6630
1MOD6640
1MOD6650
1MOD6660
1MOD6670
1MOD6680
1MOD6690
1MOD6700
1MOD6710

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```

AG=(1.-VG3VG)/EG
A=4.*ATAN(1.)*(AP+AG)
CP=SQRT(1./A)
GO TO 20
140 DO 150 I=1,2
WRITE (6,570) I
CL(I)=CKDATA(CL(I))
CONTINUE
GO TO 20
150 WRITE (6,580)
CH=CKDATA(CH)
GO TO 20
160 WRITE (6,590)
CT=CKDATA(CT)
GO TO 20
170 DO 190 I=1,6
WRITE (6,600) I
CK(I)=CKDATA(CR(I))
CONTINUE
GO TO 20
180 DO 210 I=1,6
WRITE (6,610) I
SAC(I)=CKDATA(SAC(I))
CONTINUE
GO TO 20
190 WRITE (6,620)
AKV=CKDATA(AKV)
GO TO 20
200 WRITE (6,630)
AKS=CKDATA(AKS)
GO TO 20
210 WRITE (6,640)
AKM=CKDATA(AKM)
GO TO 20
220 DO 260 I=1,2
WRITE (6,650) I

```

```

1MOD6720
1MOD6730
1MOD6740
1MOD6750
1MOD6760
1MOD6770
1MOD6780
1MOD6790
1MOD6800
1MOD6810
1MOD6820
1MOD6830
1MOD6840
1MOD6850
1MOD6860
1MOD6870
1MOD6880
1MOD6890
1MOD6900
1MOD6910
1MOD6920
1MOD6930
1MOD6940
1MOD6950
1MOD6960
1MOD6970
1MOD6980
1MOD6990
1MOD7000
1MOD7010
1MOD7020
1MOD7030
1MOD7040
1MOD7050
1MOD7060
1MOD7070

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```

260 AKO(I)=CKDATA(AKO(I))
CONTINUE
GO TO 20
270 DO 280 I=1,2
WRITE(6,660) I
AKL(I)=CKDATA(AKL(I))
CONTINUE
280 GO TO 20
WRITE(6,670)
AKT=CKDATA(AKT)
GO TO 20
300 DO 310 I=1,6
WRITE(6,680) I
AKR(I)=CKDATA(AKR(I))
CONTINUE
310 GO TO 20
320 DO 330 I=1,6
WRITE(6,690) I
SAT(I)=CKDATA(SAT(I))
CONTINUE
GO TO 20
C
C
C
C
340 FORMAT(1H1,4X,58HTHE FOLLOWING IS A LISTING OF THE PRE-PROGRAMMED
1 CONSTANTS,/,4X,55HUSED IN THE AGMA FORMULATIONS WITH APPROPRIATE
2 NOTES ON,/,4X,55HTHEIR APPLICATION. NOTE: THOSE STARTING WITH A
3 C' ARE,/,4X,54HDURABILITY CONSTANTS AND THOSE WITH A 'K' ARE STREN
4 GTH,/,4X,56HCONSTANTS. SERVICE FACTOR APPLIES TO BOTH FORMULATION
5 S.,/)
350 FORMAT(4X,2HID,4X,5HCONST,4X,8HVALUE(S),3X,5HNOTES,/,4X,12H 1 SF1MOD7390
1(1,1),5X,F4.2,5X,21HSERVICE FACTOR; A1,B1,/,4X,12H SF(1,2),5X,F1MOD7400
24.2,5X,21H A1,B2,/,4X,12H SF(2,1),5X,F4.2,5X,21H1MOD7410
3 A2,B1,/,4X,12H SF(2,2),5X,F4.2,5X,21H
4 A2,B2,/)
1MOD7080
1MOD7090
1MOD7100
1MOD7110
1MOD7120
1MOD7130
1MOD7140
1MOD7150
1MOD7160
1MOD7170
1MOD7180
1MOD7190
1MOD7200
1MOD7210
1MOD7220
1MOD7230
1MOD7240
1MOD7250
1MOD7260
1MOD7270
1MOD7280
1MOD7290
1MOD7300
1MOD7310
1MOD7320
1MOD7330
1MOD7340
1MOD7350
1MOD7360
1MOD7370
1MOD7380
1MOD7390
1MOD7400
1MOD7410
1MOD7420
1MOD7430

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360 FORMAT (4X,12H 2 CV(1) ,5X,F4.2,5X,18HDYNAMIC FACTOR; C1,/4X,121MOD7440
1H CV(2) ,5X,F4.2,5X,18H C2,/4X,12H CV(3) 1MOD7450
2,5X,F4.2,5X,18H C3,/4X,12H 3 CS ,5X,F4.2,5X,1MOD7460
311HSIZE FACTOR, //4X,12H 4 CM(1) ,5X,F4.2,5X,28HLOAD DISTRIBUTIO1MOD7470
4N FACTOR; A1,/4X,12H CM(2) ,5X,F4.2,5X,28H 1MOD7480
5 A2,/4X,12H 5 CF ,5X,F4.2,5X,24HSURFACE CONDITION FAC1MOD7490
6TOR,/) 1MOD7500
370 FORMAT (4X,12H 6 CO(1) ,5X,F4.2,5X,19HOVERLOAD FACTOR; A1,/4X,11MOD7510
12H CO(2) ,5X,F4.2,5X,19H A2,/4X,12H 7 C1MOD7520
2P ,4X,F6.1,4X,25HELASTIC PROPERTIES FACTOR, //4X,12H 8 CL(1) ,51MOD7530
3X,F4.2,5X,15HLIFE FACTOR; A1,/4X,12H CL(2) ,5X,F4.2,5X,15H 1MOD7540
4 A2,/4X,12H 9 CH ,5X,F4.2,5X,21HARDNESS RATIO FAC1MOD7550
5TOR, //4X,12H10 CT ,5X,F4.2,5X,18TEMPERATURE FACTOR,/) 1MOD7560
380 FORMAT (4X,12H11 CR(1) ,5X,F4.2,5X,22HRELIABILITY FACTOR; D1,/41MOD7570
1X,12H CR(2) ,5X,F4.2,5X,22H D2,/4X,12H 1MOD7580
2 CR(3) ,5X,F4.2,5X,22H D3,/4X,12H CR(4) ,1MOD7590
35X,F4.2,5X,22H D4,/4X,12H CR(5) ,5X,F4.2,51MOD7600
4X,22H D5,/4X,12H CR(6) ,5X,F4.2,5X,22H 1MOD7610
5 D6,/) 1MOD7620
390 FORMAT (4X,12H12 SAC(1) ,4X,F7.0,3X,28HALLOWABLE CONTACT STRESS; 1MOD7630
1 D1,/4X,12H SAC(2) ,4X,F7.0,3X,28H D3,/41MOD7640
2,/4X,12H SAC(3) ,4X,F7.0,3X,28H D4,/4X,11MOD7650
3X,12H SAC(4) ,4X,F7.0,3X,28H D5,/4X,12H 1MOD7660
42H SAC(5) ,4X,F7.0,3X,28H D6,/) 1MOD7670
5 SAC(6) ,4X,F7.0,3X,28H KV ,5X,F4.2,5X,14HDYNAMIC FACTOR, //4X,12H141MOD7690
400 FORMAT (4X,12H13 KS ,5X,F4.2,5X,11HSIZE FACTOR, //4X,12H15 KM ,5X,F4.2,1MOD7700
1 KS ,5X,F4.2,5X,11HSIZE FACTOR, //4X,12H16 KO(1) ,5X,F4.2,5X,19H1MOD7710
25X,24HLOAD DISTRIBUTION FACTOR, //4X,12H17 KO(2) ,5X,F4.2,5X,19H 1MOD7720
3OVERLOAD FACTOR; E1,/4X,12H KL(1) ,5X,F4.2,5X,15HLIFE FACTOR; A1,/4X,12H1MOD7730
4 E2,/4X,12H17 KL(2) ,5X,F4.2,5X,15H A2,/) 1MOD7740
5 KL(2) ,5X,F4.2,5X,15H KT ,5X,F4.2,5X,18TEMPERATURE FACTOR, //4X,11MOD7750
410 FORMAT (4X,12H18 KR(1) ,5X,F4.2,5X,22HRELIABILITY FACTOR; D1,/4X,12H K1MOD7760
12H19 KR(2) ,5X,F4.2,5X,22H D2,/4X,12H KR(3) ,5X,1MOD7770
2R(2) ,5X,F4.2,5X,22H D3,/4X,12H KR(4) ,5X,F4.2,5X,21MOD7780
3F4.2,5X,22H D4,/4X,12H KR(5) ,5X,F4.2,5X,22H 1MOD7790
42H


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5          D5, /4X, 12H          KR(6) ,5X, F4.2,5X, 22H          1MOD7800
6          D6, /)          1MOD7810
420  FORMAT (4X, 12H20          SAT(1), 4X, F6.0, 4X, 29HALLOWABLE MATERIAL STRESS1MOD7820
1: D1, /4X, 12H          SAT(2), 4X, F6.0, 4X, 29H          1MOD7830
2D2, /4X, 12H          SAT(3), 4X, F6.0, 4X, 29H          D31MOD7840
3, /4X, 12H          SAT(4), 4X, F6.0, 4X, 29H          D4, /1MOD7850
44X, 12H          SAT(5), 4X, F6.0, 4X, 29H          D5, /4X1MOD7860
5, 12H          SAT(6), 4X, F6.0, 4X, 29H          D6, /)          1MOD7870
430  FORMAT (5X, 38HDEFINITIONS OF CODED NOTES FROM ABOVE: /6X, 45HA1 NA1MOD7880
1VAL PROFILE - FULL POWER, 5 PERCENT MAX, /6X, 36HA2 OTHER - MAXIMUM1MOD7890
2 LOAD, CONTINUOUS, /6X, 35HB1 POWER SOURCE - TURBINE OR MOTOR, /6X, 1MOD7900
345HB2 POWER SOURCE - MULTICYLINDER I. C. ENGINE, /6X, 25HC1 FIRST1MOD7910
4 REDUCTION STAGE, /6X, 26HC2 SECOND REDUCTION STAGE, /6X, 25HC3 THIR1MOD7920
5D REDUCTION STAGE, /)          1MOD7930
440  FORMAT (6X, 33HD1 HARDNESS RANGE: 160 - 200 BHN, /6X, 33HD2 HARDNES1MOD7940
1S RANGE: 200 - 240 BHN, /6X, 33HD3 HARDNESS RANGE: 240 - 300 BHN, /61MOD7950
2X, 33HD4 HARDNESS RANGE: 300 - 360 BHN, /6X, 33HD5 HARDNESS RANGE: 1MOD7960
3360 - 400 BHN, /6X, 33HD6 HARDNESS RANGE: 400 - 640 BHN, /6X, 21HE1 1MOD7970
4 SINGLE POWER PATH, /6X, 21HE2 DOUBLE POWER PATH, /)          1MOD7980
450  FORMAT (4X, 58HDO YOU DESIRE TO CHANGE ANY OF THE ABOVE VALUES? (Y 1MOD7990
1OR N):)          1MOD8000
460  FORMAT (/ /, 4X, 61HTO CHANGE A CONSTANT ABOVE, ENTER THE ID NUMBER W1MOD8010
1HEN PROMPTED. /, 4X, 56HUSE ID NUMBER 99 WHEN NO FURTHER CHANGES ARE 1MOD8020
2TO BE MADE. /, 4X, 60HNOTE: WHEN ASKED FOR THE NEW VALUE OF THE CONS1MOD8030
3TANT, ENTERING, /, 4X, 57HA ZERO WILL CAUSE THE ORIGINAL VALUE TO REM1MOD8040
4AIN UNCHANGED. /, 4X, 59HTHIS IS USEFUL WHEN A CONSTANT HAS MULTIPLE1MOD8050
5 VALUES, BUT NOT, /, 4X, 30HALL OF THEM ARE TO BE CHANGED.)          1MOD8060
470  FORMAT (/ /, 4X, 51H** ENTER THE CONSTANT ID NUMBER (1-20, 99 TO STOP1MOD8070
1):)          1MOD8080
480  FORMAT (/ /, 4X, 12H** ENTER SF(, I1, 1H, , I1, 2H):)          1MOD8090
490  FORMAT (/ /, 4X, 12H** ENTER CV(, I1, 2H):)          1MOD8100
500  FORMAT (/ /, 4X, 12H** ENTER CS:)          1MOD8110
510  FORMAT (/ /, 4X, 12H** ENTER CM(, I1, 2H):)          1MOD8120
520  FORMAT (/ /, 4X, 12H** ENTER CF:)          1MOD8130
530  FORMAT (/ /, 4X, 12H** ENTER CO(, I1, 2H):)          1MOD8140
540  FORMAT (/ /, 4X, 31HCURRENT YOUNG'S MODULI ARE: EP=, 2PE9.1, 6H, EG=, E91MOD8150

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1. 1, /, 4X, 33HCURRENT POISSON'S RATIOS ARE: VP=,OPF5.3,6H, VG=,F5.3) 1MOD8160
FORMAT (/ , 4X, 52H** ENTER YOUNG'S MODULI FOR PINION AND GEAR (EP,EG 1MOD8170
1) : ) 1MOD8180
FORMAT (/ , 4X, 53H** ENTER POISSON'S RATIO FOR PINION AND GEAR (VP, V 1MOD8190
1G) : ) 1MOD8200
FORMAT (/ , 4X, 12H** ENTER CL(, I1, 2H) : ) 1MOD8210
FORMAT (/ , 4X, 12H** ENTER CH: ) 1MOD8220
FORMAT (/ , 4X, 12H** ENTER CT: ) 1MOD8230
FORMAT (/ , 4X, 12H** ENTER CR(, I1, 2H) : ) 1MOD8240
FORMAT (/ , 4X, 13H** ENTER SAC(, I1, 2H) : ) 1MOD8250
FORMAT (/ , 4X, 12H** ENTER KV: ) 1MOD8260
FORMAT (/ , 4X, 12H** ENTER KS: ) 1MOD8270
FORMAT (/ , 4X, 12H** ENTER KM: ) 1MOD8280
FORMAT (/ , 4X, 12H** ENTER KO(, I1, 2H) : ) 1MOD8290
FORMAT (/ , 4X, 12H** ENTER KL(, I1, 2H) : ) 1MOD8300
FORMAT (/ , 4X, 12H** ENTER KT: ) 1MOD8310
FORMAT (/ , 4X, 12H** ENTER KR(, I1, 2H) : ) 1MOD8320
FORMAT (/ , 4X, 13H** ENTER SAT(, I1, 2H) : ) 1MOD8330
FORMAT (1A1) 1MOD8340
END 1MOD8350

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NP(I,J)=INT(PD(I)*DP(I,J)+.5)
NG(I,J)=INT(PD(I)*DG(I,J)+.5)
WRITE(6,90)
READ(5,*)FACEP(I,J)
GO TO 20
10 WRITE(6,100)
READ(5,*)DP(NRED,2)
NP(NRED,2)=INT(PD(I)*DP(NRED,2)+.5)
NG(NRED,2)=NG(NRED,1)
DG(NRED,2)=DG(NRED,1)
FACEP(NRED,2)=FACEP(NRED,1)
MGP(NRED,2)=DG(NRED,2)/DP(NRED,2)
CONTINUE
20
C
C COMPUTE RATIOS, SPEED AND POWER SPLITS, AND GEOMETRY FACTORS
C
DO 30 J=1,NDIFP
L=1
RPMP(1,J)=RPMIN(J)
RPMP(2,J)=RPMP(1,J)/MGP(L,J)
IF(NRED.EQ.1)GO TO 30
DO 30 I=3,NRED2,2
L=L+1
IM1=I-1
IP1=I+1
RPMP(I,J)=RPMP(IM1,J)
RPMP(IP1,J)=RPMP(I,J)/MGP(L,J)
CONTINUE
30 DO 50 J=1,NDIFP
PWR1=PWRIN(J)
L=0
DO 40 I=1,NRED2,2
IP1=I+1
L=L+1
PWRP(I,J)=PWR1*PWRFAC(NPATH,L)
PWRP(IP1,J)=PWRP(I,J)/FLOAT(L*NPATH)

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2MOD0340
2MOD0350
2MOD0360
2MOD0370
2MOD0380
2MOD0390
2MOD0400
2MOD0410
2MOD0420
2MOD0430
2MOD0440
2MOD0450
2MOD0460
2MOD0470
2MOD0480
2MOD0490
2MOD0500
2MOD0510
2MOD0520
2MOD0530
2MOD0540
2MOD0550
2MOD0560
2MOD0570
2MOD0580
2MOD0590
2MOD0600
2MOD0610
2MOD0620
2MOD0630
2MOD0640
2MOD0650
2MOD0660
2MOD0670
2MOD0680
2MOD0690

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2MOD0700
2MOD0710
2MOD0720
2MOD0730
2MOD0740
2MOD0750
2MOD0760
2MOD0770
2MOD0780
2MOD0790
2MOD0800
2MOD0810
2MOD0820
2MOD0830
2MOD0840
2MOD0850
2MOD0860
2MOD0870
2MOD0880
2MOD0890
2MOD0900
2MOD0910
2MOD0920
2MOD0930
2MOD0940
2MOD0950
2MOD0960
2MOD0970
2MOD0980
2MOD0990
2MOD1000
2MOD1010
2MOD1020
2MOD1030
2MOD1040
2MOD1050

40 CONTINUE
PWRP (NRED2,J)=PWR1
50 CONTINUE
DO 60 J=1,NDIFF
MGOP (J)=1.
DO 60 I=1,NRED
MGOP (J)=MGOP (J)*MGP (I,J)
CALL GFI (GEOMI (I,J),I,MGP (I,J),DP (I,J),DG (I,J),0)
CALL GFJ (GEOMJP (I,J),I,DP (I,J),DG (I,J),1,0)
CALL GPFJ (GEOMJG (I,J),I,DP (I,J),DG (I,J),2,0)
60 CONTINUE
RETURN
C
C FORMAT STATEMENTS
C
C
C
70 FORMAT (//,4X,54HTHE INFORMATION REQUESTED BELOW IS FOR REDUCTION
1STAGE,I2,/,4X,14HIN POWER TRAIN,I2,1H.)
80 FORMAT (//,1X,54H* ENTER DIAMETERS OF PINION AND GEAR, INCHES (DP,
1DG):)
90 FORMAT (//,1X,40H* ENTER FACEWIDTH OF GEAR PAIR, INCHES:.)
100 FORMAT (//,1X,46H* ENTER ONLY DIAMETER OF PINION, INCHES (DP):)
END
C
C SUBROUTINE PRLDES
C
C CODED BY: LT J.L. PAQUETTE, USN JAN 1982
C
C NAVAL POSTGRADUATE SCHOOL MONTEREY, CA 93940
C
C SUBPROGRAM TO PERFORM DESIGN CALCULATIONS FOR PARALLEL AXIS
C REDUCTION GEARS USING A BASIC RANDOM SEARCH OPTIMIZATION
C TECHNIQUE TO FIND THE GEAR DIMENSIONS SUBJECT TO SIZE AND
C POWER CONSTRAINTS.
C

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```

C THE VARIABLES ENDING IN A 'Q' ARE LOCAL TO THIS SUBROUTINE AND 2MOD1060
C REPRESENT THE VALUES OF THE REAL VARIABLE DURING A SPECIFIC 2MOD1070
C ITERATION. IF ALL CONSTRAINTS ARE MET (DESIGN IS FEASIBLE) THE 2MOD1080
C GLOBAL VARIABLES WILL TAKE ON THESE VALUES. 2MOD1090
C 2MOD1100
C EXTERNAL SUBPROGRAM(S) REQUIRED: FUNCTION POWERB, FUNCTION POWERH, 2MOD1110
C SUBROUTINE GFI, SUBROUTINE GFJ, FUNCTION AGMAE1, FUNCTION ARCCOS, 2MOD1120
C FUNCTION ARCSIN, FUNCTION RTFNDR, FUNCTION FALFA, FUNCTION SHRLD, 2MOD1130
C FUNCTION THICK, FUNCTION RNDGEN 2MOD1140
C 2MOD1150
C REAL MGOP, MGP, MGO, MGQ, KK 2MOD1160
C LOGICAL FLAG, FLAGG 2MOD1170
C DIMENSION G(10,3,2), SPDP(3,2), SPDG(3,2), HPP(3,2), HPG(3,2) 2MOD1180
C DIMENSION DPQ(3,2), DGQ(3,2), MGQ(3,2), FACEQ(3,2), SCALE(3) 2MOD1190
C DIMENSION GI(3,2), GJP(3,2), GJG(3,2), REDFAC(3), S(20), IGG(3) 2MOD1200
C COMMON /AGMAB/ SFB(2,2), AKV, AKS, AKM, AKO(2), SAT(6), AKL(2), AKR(6), AK2MOD1210
1T 2MOD1220
C COMMON /AGMAH/ SFH(2,2), CV(3), CS, CM(2), CF, CO(2), SAC(6), CP, CL(2), CH2MOD1230
1, CT, CR(6) 2MOD1240
C COMMON /DESDAT/ PWRIN(2), RPMIN(2), RPMOUT, DHELIX(3), HELIX(3), PD(3), 2MOD1250
1PND(3), DPHI(3), PHI(3), DPHIN(3), PHIN(3), NDIFF, IARR, IEPIC(3), IHARD(3)2MOD1260
2, 2), IOPRO, NPWRIN, IPWRSR(2), NRED, NPATH, NPLNT(3), NHELX 2MOD1270
C COMMON /DESPRL/ PWRFAC(2,3), MGOP(2), MGP(3,2), RPMP(6,2), PWRP(6,2), D2MOD1280
1P(3,2), DG(3,2), FACEP(3,2), GEOMI(3,2), GEOMJP(3,2), NP(3), 2MOD1290
22), NG(3,2) 2MOD1300
C 2MOD1310
C INITIALIZATION 2MOD1320
C 2MOD1330
C DATA REDFAC/1.2, 1., .83/, IQMAX/7500/, IQ/0/, M/0/, BB/.5/, FDP/1./ 2MOD1340
C DATA ALPHA/1./, SCALE/65., 5., 65./, IGG/6, 8, 10/, FLAG/.FALSE./, IK/0/ 2MOD1350
C FOURPI=16.*ATAN(1.) 2MOD1360
C LL=NPATH+(NRED-1)*2 2MOD1370
C E=1./3. 2MOD1380
C NDV=6*NRED-4 2MOD1390
C NDVP=NRED*NDIFF 2MOD1400
C NDVP3=3*NNDVP 2MOD1410

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```

MM=10*NDV
IF (NHELX.EQ.2) FDP=2.25
IF (NHELX.EQ.2) SCALE(3)=150.
IRET=1
WRITE (6,440)
READ (5,*) RND
C
C COMPUTE THE STAGE GEAR RATIOS FOR THE INITIAL DESIGN
C
DO 60 J=1,NDIFF
MGOP(J)=RPMIN(J)/RPMOUT
MGO=MGOP(J)
GO TO (10,20,30,40,50,50), LL
10 IF ((MGO.LE.1.0).OR.(MGO.GT.10.0)) GO TO 370
MGO(1,J)=MGO
GO TO 60
20 IF ((MGO.LE.2.24).OR.(MGO.GT.10.0)) GO TO 370
MGO(1,J)=MGO
GO TO 60
30 IF ((MGO.LE.2.0).OR.(MGO.GT.20.0)) GO TO 370
MGO(2,J)=SQRT(MGO)-1.
MGO(1,J)=MGO/MGO(2,J)
GO TO 60
40 IF ((MGO.LE.2.9).OR.(MGO.GT.48.4)) GO TO 370
MGO(2,J)=SQRT(MGO)+3.
MGO(1,J)=MGO/MGO(2,J)
GO TO 60
50 IF (MGO.LT.5) GO TO 370
MGO(2,J)=MGO**E
MGO(3,J)=MGO(2,J)+3.
MGO(1,J)=MGO/(MGO(2,J)*MGO(3,J))
60 CONTINUE
C
C COMPUTE POWER AND SPEED SPLITS FOR THE INITIAL DESIGN
C
DO 80 J=1,NDIFF
2MOD 1420
2MOD 1430
2MOD 1440
2MOD 1450
2MOD 1460
2MOD 1470
2MOD 1480
2MOD 1490
2MOD 1500
2MOD 1510
2MOD 1520
2MOD 1530
2MOD 1540
2MOD 1550
2MOD 1560
2MOD 1570
2MOD 1580
2MOD 1590
2MOD 1600
2MOD 1610
2MOD 1620
2MOD 1630
2MOD 1640
2MOD 1650
2MOD 1660
2MOD 1670
2MOD 1680
2MOD 1690
2MOD 1700
2MOD 1710
2MOD 1720
2MOD 1730
2MOD 1740
2MOD 1750
2MOD 1760
2MOD 1770

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RPM1=RPMIN(J)
PWR1=PWRIN(J)
DO 70 I=1,NRED
SPDP(I,J)=RPM1
SPDG(I,J)=RPM1/MGQ(I,J)
RPM1=SPDG(I,J)
HPP(I,J)=PWR1*PWRFAC(NPATH,I)
HPG(I,J)=PWR1/FLOAT(NPATH*I)
CONTINUE
HPG(NRED,J)=PWR1
CONTINUE
C
C ESTIMATE INITIAL DESIGN AS START POINT FOR OPTIMIZATION
C
DO 90 J=1,NDIFP
DO 90 I=1,NRED
IH=IHARD(I,1)
BRAC=SAC(IH)*1.E-04/CR(IH)
KK=BRAC*BRAC*REDFAC(I)*2.80/(CO(IOPRO)*CM(IOPRO))
ANUM=126050.*HPP(I,J)*MGQ(I,J)
DEN=SPDP(I,J)*FDP*KK*MGQ(I,J)
DPQ(I,J)=(ANUM/DEN)**E
FACEQ(I,J)=FDP*DPQ(I,J)
CONTINUE
90
C
C COMPUTE VALUES OF DEPENDENT VARIABLES
C
100 GO TO (110,120,130),NRED
C ***
110 MGQ(1,1)=MGOP(1)
    DGQ(1,1)=MGQ(1,1)*DPQ(1,1)
    IF(NDIFP.EQ.1) GO TO 150
    MGQ(1,2)=MGOP(2)
    DGQ(1,2)=DGQ(1,1)
    DPQ(1,2)=DGQ(1,2)/MGQ(1,2)
    FACEQ(1,2)=FACEQ(1,1)

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2 MOD 1780
2 MOD 1790
2 MOD 1800
2 MOD 1810
2 MOD 1820
2 MOD 1830
2 MOD 1840
2 MOD 1850
2 MOD 1860
2 MOD 1870
2 MOD 1880
2 MOD 1890
2 MOD 1900
2 MOD 1910
2 MOD 1920
2 MOD 1930
2 MOD 1940
2 MOD 1950
2 MOD 1960
2 MOD 1970
2 MOD 1980
2 MOD 1990
2 MOD 2000
2 MOD 2010
2 MOD 2020
2 MOD 2030
2 MOD 2040
2 MOD 2050
2 MOD 2060
2 MOD 2070
2 MOD 2080
2 MOD 2090
2 MOD 2100
2 MOD 2110
2 MOD 2120
2 MOD 2130

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2MOD2140
2MOD2150
2MOD2160
2MOD2170
2MOD2180
2MOD2190
2MOD2200
2MOD2210
2MOD2220
2MOD2230
2MOD2240
2MOD2250
2MOD2260
2MOD2270
2MOD2280
2MOD2290
2MOD2300
2MOD2310
2MOD2320
2MOD2330
2MOD2340
2MOD2350
2MOD2360
2MOD2370
2MOD2380
2MOD2390
2MOD2400
2MOD2410
2MOD2420
2MOD2430
2MOD2440
2MOD2450
2MOD2460
2MOD2470
2MOD2480
2MOD2490

GO TO 150
DOUBLE REDUCTION
120  MGQ(1,1)=MGOP(1)/MGQ(2,1)
      DGQ(1,1)=MGQ(1,1)*DPQ(1,1)
      DGQ(2,1)=MGQ(2,1)*DPQ(2,1)
      IF (NDIFP.EQ.1) GO TO 150
      DGQ(2,2)=DGQ(2,1)
      MGQ(2,2)=DGQ(2,2)/DPQ(2,2)
      MGQ(1,2)=MGOP(2)/MGQ(2,2)
      DGQ(1,2)=MGQ(1,2)*DPQ(1,2)
      FACEQ(2,2)=FACEQ(1,2)
GO TO 150
C  * * *
TRIPLE REDUCTION
130  MGQ(1,1)=MGOP(1)/(MGQ(2,1)*MGQ(3,1))
      DO 140 I=1,3
140  DGQ(I,1)=MGQ(I,1)*DPQ(I,1)
      IF (NDIFP.EQ.1) GO TO 150
      DGQ(3,2)=DGQ(3,1)
      MGQ(3,2)=DGQ(3,2)/DPQ(3,2)
      MGQ(1,2)=MGOP(2)/(MGQ(2,2)*MGQ(3,2))
      DGQ(2,2)=MGQ(2,2)*DPQ(2,2)
      DGQ(1,2)=MGQ(1,2)*DPQ(1,2)
      FACEQ(3,2)=FACEQ(3,1)
C
C  COMPUTE CONSTRAINTS AND OBJECTIVE FUNCTION
C
150  VQ=0.0
      FLAG=.FALSE.
      DO 190 J=1,NDIFP
      DO 190 I=1,NRED
      CALL GFI (GI(I,J),I, MGQ(I,J), DPQ(I,J), DGQ(I,J), 0)
      APWRH=POWERH(SPDP(I,J),FACEQ(I,J),DPQ(I,J),I,J,GI(I,J))
      CALL GFJ (GJP(I,J),I,DPQ(I,J),DGQ(I,J),1,0)
      APWRBP=POWERB(SPDP(I,J),FACEQ(I,J),DPQ(I,J),I,J,1,GJP(I,J))
      CALL GFJ (GJG(I,J),I,DPQ(I,J),DGQ(I,J),2,0)
      APWRBG=POWERB(SPDP(I,J),FACEQ(I,J),DPQ(I,J),I,J,2,GJG(I,J))

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FLB=FOURPI/(PD(I)*TAN(HELIX(I)))
IF (FDP.EQ.2.25) FLB=DPQ(I,J)
FUB=FDP*DPQ(I,J)
G(1,I,J)=HPP(I,J)/APWRH-1.0
G(2,I,J)=HPP(I,J)/APWRBP-1.0
G(3,I,J)=HPG(I,J)/APWRBG-1.0
IF ((G(1,I,J).GT.0.).OR.(G(2,I,J).GT.0.).OR.(G(3,I,J).GT.0.)) FLAG2MOD2560
1G=.TRUE.
G(4,I,J)=DGQ(I,J)/200.-1.0
G(5,I,J)=FLB/FACEQ(I,J)-1.0
G(6,I,J)=FACEQ(I,J)/FUB-1.0
GO TO (180,160,170), NRED
160 IF (NPATH.EQ.1) G(7,I,J)=DGQ(1,J)/DGQ(2,J)-1.0
IF (NPATH.EQ.2) G(7,I,J)=MGQ(1,J)/MGQ(2,J)-1.0
G(8,I,J)=DPQ(1,J)/DPQ(2,J)-1.0
GO TO 180
170 G(7,I,J)=MGQ(1,J)/MGQ(2,J)-1.0
G(8,I,J)=MGQ(2,J)/MGQ(3,J)-1.0
G(9,I,J)=DPQ(1,J)/DPQ(2,J)-1.0
G(10,I,J)=DPQ(2,J)/DPQ(3,J)-1.0
VQ=VQ+.25*(MGQ(I,J)+1.)*(MGQ(I,J)+1.)*DPQ(I,J)*DPQ(I,J)*FACEQ(I,J)
180 CONTINUE
190
C
C CHECK FOR CONSTRAINT VIOLATIONS (CONSTRAINTS VIOLATED IF AT
C LEAST ONE HAS A VALUE GREATER THAN ZERO)
C
GMAX=-1.0E+20
IG=IGG(NRED)
DO 200 K=1,IG
DO 200 I=1,NRED
DO 200 J=1,NDIFF
GMAX=AMAX1(GMAX,G(K,I,J))
CONTINUE
200 GO TO (210,300), IRET
C
C SAVE THIS ITERATION'S DESIGN
2MOD2500
2MOD2510
2MOD2520
2MOD2530
2MOD2540
2MOD2550
2MOD2560
2MOD2570
2MOD2580
2MOD2590
2MOD2600
2MOD2610
2MOD2620
2MOD2630
2MOD2640
2MOD2650
2MOD2660
2MOD2670
2MOD2680
2MOD2690
2MOD2700
2MOD2710
2MOD2720
2MOD2730
2MOD2740
2MOD2750
2MOD2760
2MOD2770
2MOD2780
2MOD2790
2MOD2800
2MOD2810
2MOD2820
2MOD2830
2MOD2840
2MOD2850

```



```

C
210 GMXSTR=GMAX
220 FLAG=.FALSE.
IF (GMAX.GT.0.0) FLAG=.TRUE.
VSTR=VQ
KS=1
DO 230 J=1,NDIFF
L=0
DO 230 I=1,NRED
MGP(I,J)=MGQ(I,J)
DP(I,J)=DPQ(I,J)
DG(I,J)=DGQ(I,J)
FACEP(I,J)=FACEQ(I,J)
NP(I,J)=INT(DP(I,J)*PD(I)+.5)
NG(I,J)=INT(DG(I,J)*PD(I)+.5)
GEOMI(I,J)=GI(I,J)
GEOMJP(I,J)=GJP(I,J)
GEOMJG(I,J)=GJG(I,J)
L=L+1
RPMP(L,J)=SPDP(I,J)
PWRP(L,J)=HPP(I,J)
L=L+1
RPMP(L,J)=SPDG(I,J)
PWRP(L,J)=HPG(I,J)
CONTINUE
230 IF (IRET.EQ.2) GO TO 280
C
C PERFORM LOCAL RANDOM SEARCHES NEAR INITIAL/MOST RECENT DESIGN
C
IRET=2
M=M+1
240 IF (M.LT.MM) GO TO 250
ALPHA=BB*ALPHA
IF (ALPHA.LT.1.E-04) GO TO 340
M=0
250 SMAX=-1.E+10
2MOD2860
2MOD2870
2MOD2880
2MOD2890
2MOD2900
2MOD2910
2MOD2920
2MOD2930
2MOD2940
2MOD2950
2MOD2960
2MOD2970
2MOD2980
2MOD2990
2MOD3000
2MOD3010
2MOD3020
2MOD3030
2MOD3040
2MOD3050
2MOD3060
2MOD3070
2MOD3080
2MOD3090
2MOD3100
2MOD3110
2MOD3120
2MOD3130
2MOD3140
2MOD3150
2MOD3160
2MOD3170
2MOD3180
2MOD3190
2MOD3200
2MOD3210

```



```

IS=0
DO 260 JJ=1, NDVP
DO 260 II=1, 3
IS=IS+1
RND=RNDGEN(RND)
S(IS)=(2.*RND-1.)*SCALE(II)
SMAX=AMAX1(SMAX, ABS(S(IS)))
260 DO 270 IS=1, NDVP3
S(IS)=S(IS)/SMAX
KS=0
L=0
280 DO 290 JJ=1, NDIFP
DO 290 II=1, NRED
L=L+1
IF (FLAGG) S(L)=ABS(S(L))
DPQ(II, JJ)=DP(II, JJ)+ALPHA*S(L)
L=L+1
IF (FLAGG) S(L)=ABS(S(L))
MGQ(II, JJ)=MGP(II, JJ)+ALPHA*S(L)
L=L+1
290 FACEQ(II, JJ)=FACEP(II, JJ)+ALPHA*S(L)
GO TO 100
300 IQ=IQ+1
IF (IQ.GT.IQMAX) GO TO 340
IF (GMAX.GT.0.0) GO TO 330
IK=IK+1
IF (IK.EQ.1) APLHA=1.0
IF (VQ.LT.VSTR) GO TO 220
IF (KS.EQ.1) GO TO 240
310 DO 320 IS=1, NDVP3
S(IS)=-S(IS)
KS=1
GO TO 280
330 IF (GMAX.GT.GMXSTR) GO TO 310
GMXSTR=GMAX
GO TO 220

```

```

2MOD3220
2MOD3230
2MOD3240
2MOD3250
2MOD3260
2MOD3270
2MOD3280
2MOD3290
2MOD3300
2MOD3310
2MOD3320
2MOD3330
2MOD3340
2MOD3350
2MOD3360
2MOD3370
2MOD3380
2MOD3390
2MOD3400
2MOD3410
2MOD3420
2MOD3430
2MOD3440
2MOD3450
2MOD3460
2MOD3470
2MOD3480
2MOD3490
2MOD3500
2MOD3510
2MOD3520
2MOD3530
2MOD3540
2MOD3550
2MOD3560
2MOD3570

```



```

C      COMPUTE ACTUAL OVERALL GEAR RATIOS AND SPEEDS TO BE USED
C
C      DO 350 J=1,NDIFF
C      MGOP(J)=1.
C      RPM1=RPMIN(J)
C      L=0
C      DO 350 I=1,NRED
C      MGOP(J)=MGOP(J)*MGP(I,J)
C      L=L+1
C      RPMP(L,J)=RPM1
C      L=L+1
C      RPMP(L,J)=RPM1/MGP(I,J)
C      RPM1=RPMP(L,J)
C      CONTINUE
C      END OF DESIGN ITERATIONS
C
C      IF (FLAG) GO TO 360
C      RETURN
C      ERROR CONDITION HANDLING
C
C      WRITE (6,430)
C      RETURN
C      GO TO (380,390,400,410,420,420), L
C      WRITE (6,450) MGO
C      WRITE (6,500)
C      STOP
C      WRITE (6,460) MGO
C      WRITE (6,500)
C      STOP
C      WRITE (6,470) MGO
C      WRITE (6,500)
C      STOP
C      WRITE (6,480) MGO
C
2MOD3580
2MOD3590
2MOD3600
2MOD3610
2MOD3620
2MOD3630
2MOD3640
2MOD3650
2MOD3660
2MOD3670
2MOD3680
2MOD3690
2MOD3700
2MOD3710
2MOD3720
2MOD3730
2MOD3740
2MOD3750
2MOD3760
2MOD3770
2MOD3780
2MOD3790
2MOD3800
2MOD3810
2MOD3820
2MOD3830
2MOD3840
2MOD3850
2MOD3860
2MOD3870
2MOD3880
2MOD3890
2MOD3900
2MOD3910
2MOD3920
2MOD3930

```



```

C      AXIS GEAR SETS
C      2MOD4300
C      2MOD4310
C      2MOD4320
C      2MOD4330
C      2MOD4340
C      2MOD4350
C      2MOD4360
C      2MOD4370
C      2MOD4380
C      2MOD4390
C      2MOD4400
C      2MOD4410
C      2MOD4420
C      2MOD4430
C      2MOD4440
C      2MOD4450
C      2MOD4460
C      2MOD4470
C      2MOD4480
C      2MOD4490
C      2MOD4500
C      2MOD4510
C      2MOD4520
C      2MOD4530
C      2MOD4540
C      2MOD4550
C      2MOD4560
C      2MOD4570
C      2MOD4580
C      2MOD4590
C      2MOD4600
C      2MOD4610
C      2MOD4620
C      2MOD4630
C      2MOD4640
C      2MOD4650

EXTERNAL SUBPROGRAM(S) REQUIRED: FUNCTION ARCSIN

REAL MGOP,MGP,MFP,KFCTRP
COMMON /AGMAB/ SFB(2,2),AKV,AKS,AKM,AKO(2),SAT(6),AKL(2),AKR(6),AK2MOD4350
1T
COMMON /AGMAH/ SFH(2,2),CV(3),CS,CM(2),CF,CO(2),SAC(6),CP,CL(2),CH2MOD4370
1,CT,CR(6)
COMMON /DESDAT/ PWRIN(2),RPMIN(2),RPMOUT,DHELIX(3),HELIX(3),PD(3),2MOD4380
1PND(3),DPHI(3),PHI(3),DPHIN(3),PHIN(3),NDIFF,IARR,IEPIC(3),IHARD(3)2MOD4390
2,2),IOPRO,NPWRIN,IPWRSR(2),NRED,NPATH,NPLNT(3),NHELX
COMMON /DESPRL/ PWRFAC(2,3),MGOP(2),MGP(3,2),RPMP(6,2),PWRP(6,2),D2MOD4420
1P(3,2),DG(3,2),FACEP(3,2),GEOMI(3,2),GEOMJP(3,2),GEOMJG(3,2),NP(3,2MOD4430
22),NG(3,2)
COMMON /RESPRL/ PLVP(3,2),FBYDP(3,2),CDP(3,2),WTP(6,2),ILPIP(6,2),2MOD4450
1UNTLDP(6,2),MFP(3,2),KFCTRP(6,2),SIGHP(3,2),SIGBP(6,2),TORQP(6,2),2MOD4460
2PDIAMP(6,2),SCDMIN,SCDMAX,SHP,WGHTP,SPCWTP,TRQOUT,MTHP(6,2),ISIZEP2MOD4470
3(3)
INITIALIZE
PI=4.*ATAN(1.)
L=NPATH+(NRED-1)*2
COMPUTE ALL OUTPUT PARAMETERS
SHP=PWRIN(1)+PWRIN(2)
TRQOUT=63.*SHP/RPMOUT
DO 10 J=1,NDIFF
M=0
DO 10 I=1,NRED
M=M+1
DEN=FACEP(I,J)*DP(I,J)*MGP(I,J)/(MGP(I,J)+1.)
PLVP(I,J)=PI*DP(I,J)*RPMP(M,J)/12.
FBYDP(I,J)=FACEP(I,J)/DP(I,J)

```



```

CDP(I,J) = (DP(I,J) + DG(I,J)) / 2.
MFP(I,J) = NP(I,J) * RPMP(M,J) / 60.
WTP(M,J) = 126050. * PWRP(M,J) / (RPMP(M,J) * DP(I,J))
C1 = WTP(M,J) * CO(IOPRO) / CV(I)
C2 = CS / (FACEP(I,J) * DP(I,J))
C3 = CM(IOPRO) * CF / GEOMI(I,J)
SIGHP(I,J) = CP * SQR(C1 * C2 * C3)
NPTH = NPATH
IF ((NRED.EQ.3) .AND. (I.GE.2)) NPTH = 2
C1 = AKO(NPTH) / AKV
C2 = PD(I) / FACEP(I,J)
C3 = AKS * AKM
SIG = C1 * C2 * C3
SIGBP(M,J) = WTP(M,J) * SIG / GEOMJP(I,J)
TORQP(M,J) = WTP(M,J) * DP(I,J) / 2000.
TLPIP(M,J) = WTP(M,J) / FACEP(I,J)
UNTLDP(M,J) = TLPIP(M,J) * PND(I)
KFCTRP(M,J) = WTP(M,J) / DEN
MTHP(M,J) = NP(I,J)
PDIAMP(M,J) = DP(I,J)
M = M + 1
WTP(M,J) = 126050. * PWRP(M,J) / (RPMP(M,J) * DG(I,J))
SIGBP(M,J) = WTP(M,J) * SIG / GEOMJG(I,J)
TORQP(M,J) = WTP(M,J) * DG(I,J) / 2000.
TLPIP(M,J) = WTP(M,J) / FACEP(I,J)
UNTLDP(M,J) = TLPIP(M,J) * PND(I)
KFCTRP(M,J) = WTP(M,J) / DEN
MTHP(M,J) = NG(I,J)
PDIAMP(M,J) = DG(I,J)
CONTINUE
IF (NPWRIN.EQ.1) RETURN

```

10

C

C

C

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C

COMPUTE SOURCE CENTERLINE DISTANCE LIMITS
A MINIMUM 12.0 INCH CLEARANCE IS USED BETWEEN EACH POWER
TRAINS' FIRST REDUCTION GEARS' PITCH DIAMETERS.


```

C  *  *  *  *  *
20  GO TO (20,30,40,50,60,70), L
    SINGLE REDUCTION, SINGLE POWER PATH
    SCDMIN=DP(1,1)/2.+6.
    SCDMAX=SQRT(CDP(1,1)*CDP(1,1)-SCDMIN*SCDMIN)
    RETURN
C  *  *  *  *  *
30  SINGLE REDUCTION, DUAL POWER PATH
    A=DP(1,1)/2.+6.
    A1=ARCSIN(A,CDP(1,1))
    ARG=DP(1,1)/CDP(1,1)
    G1=A 1+ATAN(ARG)
    CSTR=SQRT(CDP(1,1)*CDP(1,1)+DP(1,1)*DP(1,1))
    SCDMIN=CSTR*SIN(G1)
    SCDMAX=CSTR*COS(G1)
    RETURN
C  *  *  *  *  *
40  DOUBLE REDUCTION, SINGLE POWER PATH
    SCDMIN=DG(1,1)/2.+6.
    A1=ARCSIN(SCDMIN,CDP(2,1))
    SCDMAX=CDP(1,1)+CDP(2,1)*COS(A1)
    RETURN
C  *  *  *  *  *
50  DOUBLE REDUCTION, DUAL POWER PATH
    A=DG(1,1)/2.+6.
    A1=ARCSIN(A,CDP(2,1))
    ARG=CDP(1,1)/CDP(2,1)
    G1=A 1+ATAN(ARG)
    CSTR=SQRT(CDP(1,1)*CDP(1,1)+CDP(2,1)*CDP(2,1))
    SCDMIN=CSTR*SIN(G1)
    SCDMAX=CSTR*COS(G1)
    IF (G1.GE.ATAN(1.)) SCDMAX=SCDMIN
    RETURN
C  *  *  *  *  *
60  TRIPLE REDUCTION, FIRST RED. HAS SINGLE POWER PATH
    A=DP(1,1)/2.+6.
    B=DG(1,1)/2.+6.
    B1=ARCSIN(B,CDP(3,1))
    ARG=CDP(2,1)/CDP(3,1)
    G1=B 1+ATAN(ARG)
    CSTR=SQRT(CDP(2,1)*CDP(2,1)+CDP(3,1)*CDP(3,1))

```

```

2MOD5020
2MOD5030
2MOD5040
2MOD5050
2MOD5060
2MOD5070
2MOD5080
2MOD5090
2MOD5100
2MOD5110
2MOD5120
2MOD5130
2MOD5140
2MOD5150
2MOD5160
2MOD5170
2MOD5180
2MOD5190
2MOD5200
2MOD5210
2MOD5220
2MOD5230
2MOD5240
2MOD5250
2MOD5260
2MOD5270
2MOD5280
2MOD5290
2MOD5300
2MOD5310
2MOD5320
2MOD5330
2MOD5340
2MOD5350
2MOD5360
2MOD5370

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2MOD5380
2MOD5390
2MOD5400
2MOD5410
2MOD5420
2MOD5430
2MOD5440
2MOD5450
2MOD5460
2MOD5470
2MOD5480
2MOD5490
2MOD5500
2MOD5510
2MOD5520
2MOD5530
2MOD5540
2MOD5550
2MOD5560
2MOD5570
2MOD5580
2MOD5590
2MOD5600
2MOD5610
2MOD5620
2MOD5630
2MOD5640
2MOD5650
2MOD5660
2MOD5670
2MOD5680
2MOD5690
2MOD5700
2MOD5710
2MOD5720
2MOD5730

A1=CSTR* $\sin$ (G1)-CDP(1,1)
SCDMIN=AMAX1(A1,A)
SCDMAX=CDP(1,1)+CSTR* $\cos$ (G1)
RETURN
TRIPLE REDUCTION, FIRST RED. HAS DUAL POWER PATH
70 A=AMAX1(DG(2,1),DP(3,1))/2.+6.
A1=ARCSIN(A,CDP(3,1))
ARG=CDP(2,1)/CDP(3,1)
B1=ATAN(ARG)
CSTR1=SQRT(CDP(2,1)*CDP(2,1)+CDP(3,1)*CDP(3,1))
ARG=CDP(1,1)/CSTR1
C1=ATAN(ARG)
D1=A1+B1+C1
CSTR2=SQRT(CDP(1,1)*CDP(1,1)+CSTR1*CSTR1)
SCDMIN=CSTR2*SIN(D1)
SCDMAX=CSTR2*COS(D1)
IF(D1.GE.ATAN(1.))SCDMAX=SCDMIN
RETURN
END
SUBROUTINE PRLSIZ
C CODED BY: LT J.L. PAQUETTE, USN JAN 1982
C NAVAL POSTGRADUATE SCHOOL MONTEREY, CA 93940
C
C SUBPROGRAM TO PROVIDE ESTIMATES ON WEIGHT (LBS) AND DIMENSIONS
C (IN) OF PARALLEL AXIS MARINE PROPULSION REDUCTION GEARS
C ALL RELATIONS ARE EMPIRICAL AND BASED ON ONLY A LIMITED
C NUMBER OF ACTUAL DESIGNS. OVERALL WEIGHT, NEGLECTING THE
C WEIGHT OF AUXILIARY EQUIPMENT, IS BASED ON THE SUM OF ALL
C GEAR BLANK VOLUMNS (F*D*W). OVERALL DIMENSIONS ARE BASED
C ON BULL GEAR DIAMETER AND SUM OF EACH REDUCTION STAGE'S
C FACEWIDTH.
C

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2MOD6100
2MOD6110
2MOD6120
2MOD6130
2MOD6140
2MOD6150
2MOD6160
2MOD6170
2MOD6180
2MOD6190
2MOD6200
2MOD6210
2MOD6220
2MOD6230
2MOD6240
2MOD6250
2MOD6260
2MOD6270
2MOD6280
2MOD6290
2MOD6300
2MOD6310
2MOD6320
2MOD6330
2MOD6340
2MOD6350
2MOD6360
2MOD6370
2MOD6380
2MOD6390
2MOD6400
2MOD6410
2MOD6420
2MOD6430
2MOD6440
2MOD6450

D2F2=FNP*DG(1,J)*DG(1,J)*FACEP(1,J)
D2F3=2.*DP(2,J)*DP(2,J)*FACEP(2,J)
D2F4=4.*DG(2,J)*DG(2,J)*FACEP(2,J)
D2F5=4.*DP(3,J)*DP(3,J)*FACEP(3,J)
D2F6=D2F+D2F1+D2F2+D2F3+D2F4+D2F5
D2F7=D2F+DG(3,1)*DG(3,1)*FACEP(3,1)
WGHTP=1196.*(D2F)*0.34
IP=INT(ALOG10(WGHTP))-2
WGHTP=AINT(WGHTP/(10.*IP))* (10*IP)
SPCWTP=WGHTP/SHF

DIMENSIONS ESTIMATE

DO 80 I=1,NRED
IF (NPWRIN.EQ.1) SF=SF+FACEP(I,1)
IF (NPWRIN.EQ.2) SF=SF+AMAX1(FACEP(I,1),FACEP(I,2))
CONTINUE
80 ISIZEP(1)=INT(2.26*SF+.5)
ISIZEP(2)=INT(1.28*DG(NRED,1)+.5)
WC=1.20
IF (NPWRIN.EQ.2) WC=1.37
ISIZEP(3)=INT(WC*DG(NRED,1)+.5)
RETURN
END

SUBROUTINE PRLOUT

C CODED BY: LT J.L. PAQUETTE, USN
C NAVAL POSTGRADUATE SCHOOL MONTEREY, CA 93940
C
C SUBPROGRAM TO PRESENT ALL RESULTS FROM THE DESIGN/ANALYSIS
C FOR PARALLEL AXIS GEARS
C
C REAL MGOP,MGP,MFP,KFCTRP

```



```

DIMENSION KHARD (6,2), MHARD (12)
COMMON /DESDAT/ PWRIN (2), RPMIN (2), RPMOUT, DHELIX (3), HELIX (3), PD (3), 2MOD6460
1PND (3), DPHI (3), PHI (3), DPHIN (3), PHIN (3), NDIFP, IARR, IEPIC (3), IHARD (3) 2MOD6470
2,2), IOPRO, NPWRIN, IPWRSR (2), NRED, NPATH, NPLNT (3), NHELX 32MOD6480
COMMON /DESPRL/ PWRFAC (2,3), MGOP (2), MGP (3,2), RPMP (6,2), PWRP (6,2), D2MOD6490
1P (3,2), DG (3,2), FACEP (3,2), GEOMI (3,2), GEOMJP (3,2), GEOMJG (3,2), NP (3, 2MOD6500
22), NG (3,2) 2MOD6510
COMMON /RESPRL/ PLVP (3,2), FBYDP (3,2), CDP (3,2), WTP (6,2), TLP (6,2), 2MOD6520
1UNTLP (6,2), MFP (3,2), KFCTRP (6,2), SIGHP (3,2), SIGBP (6,2), TORQP (6,2), 2MOD6530
2PDIAMP (6,2), SCDMIN, SCDMAX, SHP, WGHTP, SPCWTP, TRQOUT, MTHP (6,2), ISIZEP 2MOD6540
3 (3) 2MOD6550
2MOD6560
2MOD6570
2MOD6580
2MOD6590
2MOD6600
2MOD6610
2MOD6620
2MOD6630
2MOD6640
2MOD6650
2MOD6660
2MOD6670
2MOD6680
2MOD6690
2MOD6700
2MOD6710
2MOD6720
2MOD6730
2MOD6740
2MOD6750
2MOD6760
2MOD6770
2MOD6780
2MOD6790
2MOD6800
2MOD6810
INITIALIZATION
DATA KHARD/160,200,240,300,360,400,200,240,300,360,400,640/,M/0/
NRED2=2*NRED
NRED4=4*NRED
DO 10 II=1,NRED
DO 10 JJ=1,2
M=M+1
I=IHARD (II, JJ)
MHARD (M) =KHARD (I, 1)
M=M+1
MHARD (M) =KHARD (I, 2)
PRINT OUTPUT
DO 20 J=1,NDIFP
WRITE (6,30)
IF ((NPWRIN.EQ.2).AND.(NDIFP.EQ.1)) WRITE (6,70)
IF (IPWRSR (J).EQ.1) WRITE (6,40) J
IF (IPWRSR (J).EQ.2) WRITE (6,50) J
WRITE (6,60) PWRIN (J), RPMIN (J)
WRITE (6,80) NPWRIN, NPATH, NRED
WRITE (6,90) SHP, RPMOUT, MGOP (J), TRQOUT
IF (NPWRIN.EQ.2) WRITE (6,100) SCDMIN, SCDMAX

```

C
C
C

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C
C
C


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WRITE (6, 110) WGHTP, SPCWTP, (ISIZEP(I), I=1, 3)
IF (NRED.EQ.1) WRITE (6, 120)
IF (NRED.EQ.2) WRITE (6, 130)
IF (NRED.EQ.3) WRITE (6, 140)
WRITE (6, 150) (PWRP(I, J), I=1, NRED2)
WRITE (6, 160) (RPMP(I, J), I=1, NRED2)
WRITE (6, 170) (MTHP(I, J), I=1, NRED2)
WRITE (6, 180) (PND(I), I=1, NRED)
WRITE (6, 190) (PD(I), I=1, NRED)
WRITE (6, 200) (DPHIN(I), I=1, NRED)
WRITE (6, 210) (DPHI(I), I=1, NRED)
WRITE (6, 220) (DHELIX(I), I=1, NRED)
WRITE (6, 230) (MGP(I, J), I=1, NRED)
WRITE (6, 240) (PDIAMP(I, J), I=1, NRED2)
WRITE (6, 250) (FACEP(I, J), I=1, NRED)
WRITE (6, 260) (FBYDP(I, J), I=1, NRED)
WRITE (6, 270) (CDP(I, J), I=1, NRED)
WRITE (6, 280) (PLVP(I, J), I=1, NRED)
WRITE (6, 290) (WTP(I, J), I=1, NRED2)
WRITE (6, 300) (TLPIP(I, J), I=1, NRED2)
WRITE (6, 310) (UNTLDP(I, J), I=1, NRED2)
WRITE (6, 320) (MFP(I, J), I=1, NRED)
WRITE (6, 330) (KFCTRP(I, J), I=1, NRED2)
WRITE (6, 340) (SIGHP(I, J), I=1, NRED)
WRITE (6, 350) (SIGBP(I, J), I=1, NRED2)
WRITE (6, 360) (TORQP(I, J), I=1, NRED2)
WRITE (6, 370) (MHARD(I), I=1, NRED4)
CONTINUE
WRITE (6, 30)
RETURN

FORMAT STATEMENTS

FORMAT (/ , 1X, 72(1H*), /)
FORMAT (2X, 12HPower SOURCE, I2, 19H: TURBINE OR MOTOR)

```

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280	FORMAT	(1X,24HPITCHLINE VELOCITY	FPM	1,3(5X,F6.0,4X,1H))	2MOD7540
290	FORMAT	(1X,24HTANGENTIAL LOAD	LB	1,3(F6.0,1X,1H,1X,F6.0,1H))	2MOD7550
300	FORMAT	(1X,24HTOOTH LOAD/IN	LB/IN	1,6(1X,F5.0,1X,1H))	2MOD7560
310	FORMAT	(1X,24HUNIT LOAD	PSI	1,3(F6.0,1X,1H,1X,F6.0,1H))	2MOD7570
320	FORMAT	(1X,24HMESH FREQUENCY	HZ	1,3(5X,F6.0,4X,1H))	2MOD7580
330	FORMAT	(1X,24HK FACTOR (COMPUTED)		1,6(1X,F5.0,1X,1H))	2MOD7590
340	FORMAT	(1X,24HCONTACT STRESS	PSI	1,3(4X,F7.0,4X,1H))	2MOD7600
350	FORMAT	(1X,24HBENDING STRESS	PSI	1,6(F7.0,1H))	2MOD7610
360	FORMAT	(1X,24HTORQUE	K IN-LB	1,6(F7.1,1H))	2MOD7620
370	FORMAT	(1X,24HHARDNESS RANGE	BHN	1,6(I3,1H-,I3,1H))	2MOD7630
		END			2MOD7640

Module Three

```
3MOD0010
3MOD0020
3MOD0030
3MOD0040
3MOD0050
3MOD0060
3MOD0070
3MOD0080
3MOD0090
3MOD0100
3MOD0110
3MOD0120
3MOD0130
3MOD0140
3MOD0150
3MOD0160
3MOD0170
3MOD0180
3MOD0190
3MOD0200
3MOD0210
3MOD0220
3MOD0230
3MOD0240
3MOD0250
3MOD0260
3MOD0270
3MOD0280
3MOD0290
3MOD0300
3MOD0310
3MOD0320
3MOD0330

SUBROUTINE EPCANL
C
C CODED BY: LT J.L. PAQUETTE, USN JAN 1982
C NAVAL POSTGRADUATE SCHOOL MONTEREY, CA 93940
C
C SUBPROGRAM TO PERFORM AN ANALYSIS OF A GIVEN EPICYCLIC GEAR SET
C
C EXTERNAL SUBPROGRAM(S) REQUIRED: SUBROUTINE GFI, SUBROUTINE GFJ,
C FUNCTION AGMAE1, FUNCTION ARCCOS, FUNCTION ARCSIN, FUNCTION FALFA,
C FUNCTION RTFNDR, FUNCTION SHRLD, FUNCTION THICK
C
C REAL MGOE,MGE,MG1
C COMMON /DESDAT/ PWRIN(2),RPMIN(2),RPMOUT,DHELIX(3),HELIX(3),PD(3),
1PND(3),DPHI(3),PHI(3),DPHIN(3),PHIN(3),NDIFF,IARR,IEPIC(3),IHARD(3)
2,2),IOPRO,NPWRIN,IPWRSR(2),NRED,NPATH,NPLNT(3),NHELX
C COMMON /DESEPC/ MGOE,MGE(3),RPMI(3),RPMPL(3),RPMO(3),PWRE(3),DS(3)
1,DPLN(3),DR(3),FACEE(3),GI(3),GJS(3),GJPL(3),NS(3),NPLN(3),NR(3)
C
C ENTER REQUIRED INFORMATION: TOOTH NUMBERS, DIAMETERS, FACEWIDTHS
C
C DO 10 I=1,NRED
C WRITE (6,30) I
C WRITE (6,40)
C READ (5,*) DS(I),DPLN(I),DR(I)
C WRITE (6,50)
C READ (5,*) FACEE(I)
C CONTINUE
10
C
C COMPUTE RATIOS, SPEED AND POWER SPLITS, AND GEOMETRY FACTORS
C
C RPM1=RPMIN(1)
C MGOE=RPMIN(1)/RPMOUT
C DO 20 I=1,NRED
```



```

C      3MOD0700
C      3MOD0710
C      3MOD0720
C      3MOD0730
C      3MOD0740
C      3MOD0750
C      3MOD0760
C      3MOD0770
C      3MOD0780
C      3MOD0790
C      3MOD0800
C      3MOD0810
C      3MOD0820
C      3MOD0830
C      3MOD0840
C      3MOD0850
C      3MOD0860
C      3MOD0870
C      3MOD0880
C      3MOD0890
C      3MOD0900
C      3MOD0910
C      3MOD0920
C      3MOD0930
C      3MOD0940
C      3MOD0950
C      3MOD0960
C      3MOD0970
C      3MOD0980
C      3MOD0990
C      3MOD1000
C      3MOD1010
C      3MOD1020
C      3MOD1030
C      3MOD1040
C      3MOD1050

TECHNIQUE TO FIND THE GEAR DIMENSIONS SUBJECT TO SIZE AND
POWER CONSTRAINTS.

THE VARIABLES ENDING IN A 'Q' ARE LOCAL TO THIS SUBROUTINE AND
REPRESENT THE VALUES OF THE REAL VARIABLE DURING A SPECIFIC
ITERATION. IF ALL CONSTRAINTS ARE MET (DESIGN IS FEASIBLE) THE
GLOBAL VARIABLES WILL TAKE ON THESE VALUES.

EXTERNAL SUBPROGRAM(S) REQUIRED: FUNCTION POWERB, FUNCTION POWERH,
SUBROUTINE GFI, SUBROUTINE GFJ, FUNCTION AGMAE1, FUNCTION ARCCOS,
FUNCTION ARCSIN, FUNCTION RTFNDR, FUNCTION FALFA, FUNCTION SHRLD,
FUNCTION THICK, FUNCTION RNDGEN

REAL MGOE,MGE,MG1,MGQ, KK
LOGICAL FLAG, FLAGG
DIMENSION G(9,3), SPD(3), HP(3,2), SCALE(3)
DIMENSION DSQ(3), DRQ(3), DPLNQ(3), MGQ(3), FACEQ(3)
DIMENSION GFIS(3), GFJS(3), GFJP(3), S(10), NSQ(3), NRQ(3), NPLNQ(3)
COMMON /AGMAB/ SFB(2,2), AKV, AKS, AKM, AKO(2), SAT(6), AKL(2), AKR(6), AK3
1T
COMMON /AGMAH/ SFH(2,2), CV(3), CS, CM(2), CP, CO(2), SAC(6), CP, CL(2), CH3
1, CT, CR(6)
COMMON /DESDAT/ PWRIN(2), RPMIN(2), RPMOUT, DHELIX(3), HELIX(3), PD(3),
1PND(3), DPHI(3), PHI(3), DPHIN(3), PHIN(3), NDIFP, IARR, IEPC(3), IHARD(3)
2,2), IOPRO, NPWRIN, IPWRSR(2), NRED, NPATH, NPLNT(3), NHELX
COMMON /DESEPC/ MGOE, MGE(3), RPMI(3), RPMPL(3), PWRE(3), DS(3)
1, DPLN(3), DR(3), FACEE(3), GI(3), GJS(3), GJPL(3), NS(3), NPLN(3), NR(3)

INITIALIZATION

DATA IQMAX/7500/, IQ/0/, M/0/, BB/.5/, FDP/1./, ALPHA/1./
DATA SCALE/30., 5., 30./, FLAG/.FALSE./, IK/0/
FOURPI=16.*ATAN(1.)
E3=1./3.
E=1./FLOAT(NRED)
NRED3=3*NRED

```



```

NDV=NRED3
MM=10*NDV
IF (NHELX.EQ.2) FDP=2.25
IF (NHELX.EQ.2) SCALE(3)=75.
IRET=1
WRITE (6,340)
READ (5,*) RND
RND=RNDGEN(RND)
C
C COMPUTE THE OVERALL GEAR RATIO, INITIAL STAGE GEAR RATIOS,
C AND POWER AND SPEED SPLITS
C
MGOE=RPMIN(1)/RPMOUT
MGMAX=8.*NRED
IF (MGOE.GT.MGMAX) GO TO 320
RPM1=RPMIN(1)
DO 20 I=1,NRED
MGQ(I)=MGOE**E
SPD(I)=RPM1
RPM1=SPD(I)/MGQ(I)
HP(I,1)=PWRIN(1)
HP(I,2)=HP(I,1)/NPLNT(I)
CONTINUE
20
C
C ESTIMATE INITIAL DESIGN AS START POINT FOR OPTIMIZATION
C
DO 30 I=1,NRED
MG1=(1.+RND)**1.5
IH=IHARD(I,1)
BRAC=SAC(IH)**1.E-04/CR(IH)
KK=BRAC**BRAC**3.36/(CO(IOPRO)**CM(IOPRO))
ANUM=126050.**HP(I,1)**(MG1+1.)
DEN=SPD(I)**FDP**KK**MG1
DSQ(I)=(ANUM/DEN)**E3
FACEQ(I)=FDP**DSQ(I)
CONTINUE
30

```



```

3MOD 1420
3MOD 1430
3MOD 1440
3MOD 1450
3MOD 1460
3MOD 1470
3MOD 1480
3MOD 1490
3MOD 1500
3MOD 1510
3MOD 1520
3MOD 1530
3MOD 1540
3MOD 1550
3MOD 1560
3MOD 1570
3MOD 1580
3MOD 1590
3MOD 1600
3MOD 1610
3MOD 1620
3MOD 1630
3MOD 1640
3MOD 1650
3MOD 1660
3MOD 1670
3MOD 1680
3MOD 1690
3MOD 1700
3MOD 1710
3MOD 1720
3MOD 1730
3MOD 1740
3MOD 1750
3MOD 1760
3MOD 1770

C
C
C
40
C
50
C
60
C
70
80
C
90
C
100
110
120

COMPUTE VALUES OF DEPENDENT VARIABLES

GO TO (50,60,70), NRED
SINGLE REDUCTION
MGQ(1)=MGOE
GO TO 80
DOUBLE REDUCTION
MGQ(1)=MGOE/MGQ(2)
GO TO 80
TRIPLE REDUCTION
MGQ(1)=MGOE/(MGQ(2)*MGQ(3))
RPM1=RPMIN(1)
DO 130 I=1,NRED
PNSQ=PD(I)*DSQ(I)
NSQ(I)=INT(PNSQ+.5)
IEP=IEPIC(I)
GO TO (90,100), IEP
PLANETARY GEAR CONFIGURATION
PNRQ=FLOAT(NSQ(I))*MGQ(I)-1.)
NRQ(I)=INT(PNRQ+.5)
PKCON=FLOAT(NRQ(I))*MGQ(I)/(FLOAT(NPLNT(I))*MGQ(I)-1.)
KCON=INT(PKCON+.5)
GO TO 110
STAR GEAR ARRANGEMENT
PNRQ=FLOAT(NSQ(I))*MGQ(I)
NRQ(I)=INT(PNRQ+.5)
PKCON=FLOAT(NRQ(I))*MGQ(I)+1.)/(MGQ(I)*FLOAT(NPLNT(I)))
KCON=INT(PKCON+.5)
NRQ(I)=KCON*NPLNT(I)-NSQ(I)
PLNTQ=(FLOAT(NRQ(I))-FLOAT(NSQ(I)))/2.
IF (PLNTQ.EQ.AINT(PLNTQ)) GO TO 120
KCON=KCON+1
GO TO 110
NPLNQ(I)=INT(PLNTQ)
MGQ(I)=FLOAT(NRQ(I))/FLOAT(NSQ(I))

```



```

3MOD2140
3MOD2150
3MOD2160
3MOD2170
3MOD2180
3MOD2190
3MOD2200
3MOD2210
3MOD2220
3MOD2230
3MOD2240
3MOD2250
3MOD2260
3MOD2270
3MOD2280
3MOD2290
3MOD2300
3MOD2310
3MOD2320
3MOD2330
3MOD2340
3MOD2350
3MOD2360
3MOD2370
3MOD2380
3MOD2390
3MOD2400
3MOD2410
3MOD2420
3MOD2430
3MOD2440
3MOD2450
3MOD2460
3MOD2470
3MOD2480
3MOD2490

C CHECK FOR CONSTRAINT VIOLATIONS (CONSTRAINTS VIOLATED IF AT
C LEAST ONE HAS A VALUE GREATER THAN ZERO)
C
C GMAX=-1.0E+20
C DO 150 K=1,9
C DO 150 I=1,NRED
C GMAX=AMAX1(GMAX,G(K,I))
C CONTINUE
150 GO TO (160,250), IRET
C
C SAVE THIS ITERATION'S DESIGN
C
C GMXSTR=GMAX
C FLAG=.FALSE.
C IF (GMAX.GT.0.0) FLAG=.TRUE.
C VSTR=VQ
C KS=1
C RPM1=RPMIN(1)
C DO 180 I=1,NRED
C MGE(I)=MGQ(I)
C DS(I)=DSQ(I)
C DPLN(I)=DPLNQ(I)
C DR(I)=DRQ(I)
C FACEE(I)=FACEQ(I)
C NS(I)=NSQ(I)
C NPLN(I)=NPLNQ(I)
C NR(I)=NRQ(I)
C GI(I)=GFIS(I)
C GJS(I)=GFJS(I)
C GJPL(I)=GFJP(I)
C RPMI(I)=RPM1
C RPMO(I)=RPM1/MGE(I)
C RPM1=RPMO(I)
C RPMPL(I)=RPM1*DR(I)/DPLN(I)
C PWRE(I)=HP(I,2)

```



```

180 CONTINUE
    IF (IRET.EQ.2) GO TO 230
C
C
C
    PERFORM LOCAL RANDOM SEARCHES NEAR INITIAL/MOST RECENT DESIGN
    IRET=2
    M=M+1
    IF (M.LT.MM) GO TO 200
    ALPHA=BB*ALPHA
    IF (ALPHA.LT.1.E-04) GO TO 290
    M=0
    SMAX=-1.E+10
    IS=0
    DO 210 JJ=1,NRED
    DO 210 II=1,3
    IS=IS+1
    RND=RDNGEN(RND)
    S(IS)=(2.*RND-1.)*SCALE(II)
    SMAX=AMAX1(SMAX,ABS(S(IS)))
    DO 220 IS=1,NRED3
    S(IS)=S(IS)/SMAX
    KS=0
    L=0
    DO 240 II=1,NRED
    L=L+1
    IF (FLAGG) S(L)=ABS(S(L))
    DSQ(II)=DS(II)+ALPHA*S(L)
    L=L+1
    IF (FLAGG) S(L)=ABS(S(L))
    MGQ(II)=MGE(II)+ALPHA*S(L)
    L=L+1
    FACEQ(II)=FACEE(II)+ALPHA*S(L)
    GO TO 40
    IQ=IQ+1
    IF (IQ.GT.IQMAX) GO TO 290
    IF (GMAX.GT.0.0) GO TO 280
    200
    210
    220
    230
    240
    250
3MOD2500
3MOD2510
3MOD2520
3MOD2530
3MOD2540
3MOD2550
3MOD2560
3MOD2570
3MOD2580
3MOD2590
3MOD2600
3MOD2610
3MOD2620
3MOD2630
3MOD2640
3MOD2650
3MOD2660
3MOD2670
3MOD2680
3MOD2690
3MOD2700
3MOD2710
3MOD2720
3MOD2730
3MOD2740
3MOD2750
3MOD2760
3MOD2770
3MOD2780
3MOD2790
3MOD2800
3MOD2810
3MOD2820
3MOD2830
3MOD2840
3MOD2850

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3MOD 2860
3MOD 2870
3MOD 2880
3MOD 2890
3MOD 2900
3MOD 2910
3MOD 2920
3MOD 2930
3MOD 2940
3MOD 2950
3MOD 2960
3MOD 2970
3MOD 2980
3MOD 2990
3MOD 3000
3MOD 3010
3MOD 3020
3MOD 3030
3MOD 3040
3MOD 3050
3MOD 3060
3MOD 3070
3MOD 3080
3MOD 3090
3MOD 3100
3MOD 3110
3MOD 3120
3MOD 3130
3MOD 3140
3MOD 3150
3MOD 3160
3MOD 3170
3MOD 3180
3MOD 3190
3MOD 3200
3MOD 3210

IK=IK+1
IF (IK.EQ.1) APLHA=1.0
IF (VQ.LT.VSTR) GO TO 170
IF (KS.EQ.1) GO TO 190
DO 270 IS=1,NRED3
S(IS)=-S(IS)
KS=1
GO TO 230
IF (GMAX.GT.GMXSTR) GO TO 260
GMXSTR=GMAX
GO TO 170

C
C COMPUTE ACTUAL OVERALL GEAR RATIOS AND SPEEDS TO BE USED
C
290 MGOE=1.
DO 300 I=1,NRED
MGOE=MGOE*MGE(I)
300
C
C END OF DESIGN ITERATIONS
C
IF (FLAG) GO TO 310
RETURN
C
C ERROR CONDITION HANDLING
C
310 WRITE (6,330)
RETURN
320 NP1=NRED+1
WRITE (6,350) MGOE,NRED,NP1
NRED=NP1
GO TO 10

C
C FORMAT STATEMENTS
C
330 FORMAT (//,4X,23H* * * * * WARNING * * * * *,/,4X,54H SIZE AND/OR ALLOW 3MOD 3210

```



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1ABLE POWER CONSTRAINTS WERE VIOLATED.,//,4X,32HTHIS DESIGN MAY NOT 3MOD33220
2BE FEASIBLE.,//,4X,34HPROGRAM CONTINUING 3MOD33230
340 FORMAT (//,2X,49H*# ENTER SEED FOR RANDOM NUMBER GENERATOR (X.XX):3MOD33240
1) 3MOD33250
350 FORMAT (//,4X,28HTHE OVERALL REDUCTION RATIO,,F7.3,17H, IS TO LARG3MOD33260
1E FOR,I2,,4X,30HREDUCTION STAGE(S); THEREFORE,,I2,33H REDUCTION S3MOD33270
2TAGE(S) WILL BE USED.) 3MOD33280
END 3MOD33290
C 3MOD33300
C 3MOD33310
C 3MOD33320
SUBROUTINE EPCRES 3MOD33330
C 3MOD33340
C CODED BY: LT J.L. PAQUETTE, USN JAN 1982 3MOD33350
C NAVAL POSTGRADUATE SCHOOL MONTEREY, CA 93940 3MOD33360
C 3MOD33370
C SUBPROGRAM TO COMPUTE ALL OUTPUT PARAMETERS FOR EPICYCLIC GEARS 3MOD33380
C 3MOD33390
REAL MGOE,MGE,MFE,KFCTRE,MG1 3MOD33400
COMMON /AGMAB/ SFB(2,2),AKV,AKS,AKM,AKO(2),SAT(6),AKL(2),AKR(6),AK 3MOD33410
1T COMMON /AGMAH/ SFH(2,2),CV(3),CS,CM(2),CF,CO(2),SAC(6),CP,CL(2),CH3MOD33430
1,CT,CR(6) 3MOD33440
COMMON /DESDAT/ PWRIN(2),RPMIN(2),RPMOUT,DHELIX(3),HELIX(3),PD(3), 3MOD33450
1PND(3),DPHI(3),PHI(3),DPHIN(3),PHIN(3),NDIFF,IARR,IEFFC(3),IHARD(33MOD33460
2,2),IOPRO,NPWRIN,IPWRSR(2),NRED,NPATH,NPLNT(3),NHELX 3MOD33470
COMMON /DESEPC/ MGOE,MGE(3),RPMI(3),RPMPL(3),RPMO(3),PWRE(3),DS(3) 3MOD33480
1,DPLN(3),DR(3),FACEE(3),GI(3),GJS(3),GJPL(3),NS(3),NPLN(3),NR(3) 3MOD33490
COMMON /RESEPC/ PLVE(3),FBYDE(3),CDE(3),WTE(3),TLPIE(3),UNTLDE(3), 3MOD33500
1MFE(3,3),KFCTRE(3),SIGHE(3),SIGBE(3),TORQE(3,3),RPME(3,3),PDIAME(33MOD33510
2,3),WGHTE,SPCWTF,MTHE(3,3),ISIZEE(3) 3MOD33520
C 3MOD33530
C INITIALIZE 3MOD33540
C 3MOD33550
C PI=4.*ATAN(1.) 3MOD33560
C 3MOD33570

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3MOD 3580
3MOD 3590
3MOD 3600
3MOD 3610
3MOD 3620
3MOD 3630
3MOD 3640
3MOD 3650
3MOD 3660
3MOD 3670
3MOD 3680
3MOD 3690
3MOD 3700
3MOD 3710
3MOD 3720
3MOD 3730
3MOD 3740
3MOD 3750
3MOD 3760
3MOD 3770
3MOD 3780
3MOD 3790
3MOD 3800
3MOD 3810
3MOD 3820
3MOD 3830
3MOD 3840
3MOD 3850
3MOD 3860
3MOD 3870
3MOD 3880
3MOD 3890
3MOD 3900
3MOD 3910
3MOD 3920
3MOD 3930

C COMPUTE ALL OUTPUT PARAMETERS
C
DO 40 I=1,NRED
PLVE(I)=PI*DS(I)*RPMI(I)/12.
FBYDE(I)=FACEE(I)/DS(I)
CDE(I)=(DS(I)+DPLN(I))/2.
WTE(I)=126050.*PWRIN(1)/(RPMI(I)*DS(I))
TLPIE(I)=WTE(I)/FACEE(I)
UNTLDE(I)=TLPIE(I)*PND(I)
ANPLNT=FLOAT(NPLNT(I))
ANR=FLOAT(NR(I))
ANPLN=FLOAT(NPLN(I))
ANS=FLOAT(NS(I))
IE=IEPIC(I)
GO TO (10,20), IE
10 ANRPNS=ANR+ANS
MFE(I,1)=ANPLNT*ANR*RPMI(I)/ANRPNS
MFE(I,2)=(ANR/ANPLN)*ANS*RPMI(I)/ANRPNS
MFE(I,3)=ANPLNT*ANS*RPMI(I)/ANRPNS
GO TO 30
20 MFE(I,1)=ANPLNT*RPMI(I)
MFE(I,2)=2.*ANS*RPMI(I)/ANPLN
MFE(I,3)=ANPLNT*ANS*RPMI(I)/ANR
30 MG1=DPLN(I)/DS(I)
KFACTRE(I)=WTE(I)*(MG1+1.)/(FACEE(I)*DS(I)*MG1)
C1=WTE(I)*CO(IOPRO)/CV(1)
C2=CS/(DS(I)*FACEE(I))
C3=CM(IOPRO)*CF/GI(I)
SIGHE(I)=CP*SQRT(C1*C2*C3)
C1=WTE(I)*AKO(2)/AKV
C2=PD(I)/FACEE(I)
C3=AKS*AKM/AMIN1(GJS(I),GJPL(I))
SIGBE(I)=C1*C2*C3
TW=WTE(I)/2000.
TORQE(I,1)=TW*DS(I)
TORQE(I,2)=TW*DPLN(I)

```



```

COMMON /RESEPC/ PLVE(3), FBYDE(3), CDE(3), WTE(3), TLPIE(3), UNTLDE(3), 3MOD4300
1MFE(3,3), KFCTRE(3), SIGHE(3), SIGBE(3), TORQE(3,3), RPME(3,3), PDIAME(3,3)MOD4310
2,3), WGHTE, SPCWTE, MTHE(3,3), ISIZEE(3) 3MOD4320
C 3MOD4330
C 3MOD4340
C 3MOD4350
C 3MOD4360
C 3MOD4370
C 3MOD4380
C 3MOD4390
C 3MOD4400
C 3MOD4410
C 3MOD4420
C 3MOD4430
C 3MOD4440
C 3MOD4450
C 3MOD4460
C 3MOD4470
C 3MOD4480
C 3MOD4490
C 3MOD4500
C 3MOD4510
C 3MOD4520
C 3MOD4530
C 3MOD4540
C 3MOD4550
C 3MOD4560
C 3MOD4570
C 3MOD4580
C 3MOD4590
C 3MOD4600
C 3MOD4610
C 3MOD4620
C 3MOD4630
C 3MOD4640
C 3MOD4650

COMMON /RESEPC/ PLVE(3), FBYDE(3), CDE(3), WTE(3), TLPIE(3), UNTLDE(3), 3MOD4300
1MFE(3,3), KFCTRE(3), SIGHE(3), SIGBE(3), TORQE(3,3), RPME(3,3), PDIAME(3,3)MOD4310
2,3), WGHTE, SPCWTE, MTHE(3,3), ISIZEE(3) 3MOD4320
C 3MOD4330
C 3MOD4340
C 3MOD4350
C 3MOD4360
C 3MOD4370
C 3MOD4380
C 3MOD4390
C 3MOD4400
C 3MOD4410
C 3MOD4420
C 3MOD4430
C 3MOD4440
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C 3MOD4460
C 3MOD4470
C 3MOD4480
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C 3MOD4520
C 3MOD4530
C 3MOD4540
C 3MOD4550
C 3MOD4560
C 3MOD4570
C 3MOD4580
C 3MOD4590
C 3MOD4600
C 3MOD4610
C 3MOD4620
C 3MOD4630
C 3MOD4640
C 3MOD4650

INITIALIZATION
D2F=0.0
SF=0.0
FNP=FLOAT(NPATH)
DRMAX=-1. E-04

COMPUTE WEIGHT ESTIMATE

DO 10 I=1, NRED
DRMAX=AMAX1(DRMAX, DR(I))
SF=SF+FACEE(I)
D2F1=DS(I)*DS(I)*FACEE(I)
D2F2=NPLNT(I)*DPLN(I)*DPLN(I)*FACEE(I)
USE 0.7 DR TO ACCOUNT FOR THE CARRIER
D2F3=.49*DR(I)*DR(I)*FACEE(I)
D2F=D2F1+D2F2+D2F3
WGHE=.905*(D2F)*.89
IP=INT(ALOG10(WGHE))-2
WGHE=AINT(WGHE/(10.*IP))* (10*IP)
SPCWTE=WGHE/PWRIN(1)

DIMENSIONS ESTIMATE

ISIZEE(1)=INT(2.85*SF+.5)
ISIZEE(2)=INT(1.30*DRMAX+.5)
ISIZEE(3)=INT(1.20*DRMAX+.5)
RETURN
END

```



```

3MOD4660
3MOD4670
3MOD4680
3MOD4690
3MOD4700
3MOD4710
3MOD4720
3MOD4730
3MOD4740
3MOD4750
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3MOD4770
3MOD4780
3MOD4790
3MOD4800
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3MOD4830
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3MOD4860
3MOD4870
3MOD4880
3MOD4890
3MOD4900
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3MOD4920
3MOD4930
3MOD4940
3MOD4950
3MOD4960
3MOD4970
3MOD4980
3MOD4990
3MOD5000
3MOD5010

SUBROUTINE EPCOUT
CODED BY:  LT J.L. PAQUETTE, USN      JAN 1982
           NAVAL POSTGRADUATE SCHOOL  MONTEREY, CA 93940

SUBPROGRAM TO PRESENT ALL RESULTS FROM THE DESIGN/ANALYSIS
FOR EPICYCLIC GEARS

REAL MGOE,MGE,MFE,KFCTRE
DIMENSION KHARD(6,2),MHARD(4)
COMMON /DESDAT/ PWRIN(2),RPMIN(2),RPMOUT,DHELIX(3),HELIX(3),PD(3),3MOD4760
1PND(3),DPHI(3),PHI(3),DPHIN(3),PHIN(3),NDIFF,IARR,IEPIC(3),IHARD(3)3MOD4770
2,2),IOPRO,NPWRIN,IPWRSR(2),NRED,NPATH,NPLNT(3),NHELX
COMMON /DESEPC/ MGOE,MGE(3),RPMI(3),RPMPL(3),RPMO(3),PWRE(3),DS(3)3MOD4790
1,DPLN(3),DR(3),FACEE(3),GI(3),GJS(3),GJPL(3),NS(3),NPLN(3),NR(3)3MOD4800
COMMON /RESEPC/ PLVE(3),FBYDE(3),CDE(3),WTE(3),TLPIE(3),UNTLDE(3),3MOD4810
1MFE(3,3),KFCTRE(3),SIGHE(3),SIGBE(3),TORQE(3,3),RPME(3,3),PDIAME(3)3MOD4820
2,3),WGHTe,SPCWTE,MTHE(3,3),ISIZEE(3)

INITIALIZATION
DATA KHARD/160,200,240,300,360,400,200,240,300,360,400,640/

PRINT OUTPUT

WRITE (6,30)
IF (IPWRSR(1).EQ.1) WRITE (6,40)
IF (IPWRSR(1).EQ.2) WRITE (6,50)
WRITE (6,60) PWRIN(1),RPMIN(1)
WRITE (6,70) NRED
WRITE (6,80) PWRIN(1),RPMOUT,MGOE,TORQE(NRED,3)
WRITE (6,90) WGHTe,SPCWTE,(ISIZEE(I),I=1,3)
DO 20 I=1,NRED
WRITE (6,30)
M=0
DO 10 J=1,2

```

C
C
C
C
C
C
C

C
C
C
C
C
C

3MOD5020
3MOD5030
3MOD5040
3MOD5050
3MOD5060
3MOD5070
3MOD5080
3MOD5090
3MOD5100
3MOD5110
3MOD5120
3MOD5130
3MOD5140
3MOD5150
3MOD5160
3MOD5170
3MOD5180
3MOD5190
3MOD5200
3MOD5210
3MOD5220
3MOD5230
3MOD5240
3MOD5250
3MOD5260
3MOD5270
3MOD5280
3MOD5290
3MOD5300
3MOD5310
3MOD5320
3MOD5330
3MOD5340
3MOD5350
3MOD5360
3MOD5370

IH=IHARD(I,J)
M=M+1
MHARD(M)=KHARD(IH,1)
M=M+1
MHARD(M)=KHARD(IH,2)
WRITE(6,100) I
IF (IEPIC(I).EQ.1) WRITE(6,110)
IF (IEPIC(I).EQ.2) WRITE(6,120)
WRITE(6,130) NPLNT(I)
WRITE(6,140) PWRIN(1),PWRE(I),PWRIN(1)
WRITE(6,150) (RPME(I,J),J=1,3)
WRITE(6,160) (MTHE(I,J),J=1,3)
WRITE(6,170) PND(I)
WRITE(6,180) PD(I)
WRITE(6,190) DPHIN(I)
WRITE(6,200) DPHI(I)
WRITE(6,210) DHELIX(I)
WRITE(6,220) MGE(I)
WRITE(6,230) (PDIAME(I,J),J=1,3)
WRITE(6,240) FACEE(I)
WRITE(6,250) FBYDE(I)
WRITE(6,260) CDE(I)
WRITE(6,270) PLVE(I)
WRITE(6,280) WTE(I)
WRITE(6,290) TLPTE(I)
WRITE(6,300) UNTLDE(I)
WRITE(6,310) (MFE(I,J),J=1,3)
WRITE(6,320) KFACTRE(I)
WRITE(6,330) SIGHE(I)
WRITE(6,340) SIGBE(I)
WRITE(6,350) (TORQE(I,J),J=1,3)
WRITE(6,360) (MHARD(J),J=1,2),(MHARD(J),J=1,4)
CONTINUE
WRITE(6,30)
RETURN

C	FORMAT STATEMENTS		3MOD5380
C			3MOD5390
C			3MOD5400
30	FORMAT (//,1X,72(1H*),/)		3MOD5410
40	FORMAT (2X,31HPower SOURCE: TURBINE OR MOTOR)		3MOD5420
50	FORMAT (2X,55HPower SOURCE: MULTICYLINDER INTERNAL COMBUSTION ENGINE)		3MOD5430
60	FORMAT (6X,18HINPUT POWER (HP): ,F7.0,4X,19HINPUT SPEED (RPM): ,F63MOD5450		3MOD5460
70	FORMAT (2X,23HARRANGEMENT: EPICYCLIC,,I2,13H REDUCTION (S))		3MOD5470
80	FORMAT (6X,19HOUTPUT POWER (HP): ,F7.0,3X,20HOUTPUT SPEED (RPM): ,3MOD5480		3MOD5490
90	FORMAT (2X,46HSIZING ESTIMATES FOR THE ENTIRE REDUCTION SET:,,/,,6X,3MOD5500		3MOD5510
100	FORMAT (43X,9HREDUCTION,I2,,/,,24X,1H ,6X,3HSUN,6X,1H ,4X,7HPLANETS,3MOD5530		3MOD5540
110	FORMAT (1X,24HGEAR ARRANGEMENT	1,19X,9HPLANETARY,19X,1H)	3MOD5560
120	FORMAT (1X,24HGEAR ARRANGEMENT	1,22X,4HSTAR,21X,1H)	3MOD5570
130	FORMAT (1X,24HNUMBER OF PLANETS	1,23X,I1,23X,1H)	3MOD5580
140	FORMAT (1X,24HPower SPLIT	HP	3MOD5590
150	FORMAT (1X,24HSPEED	RPM	3MOD5600
160	FORMAT (1X,24HNUMBER OF TEETH	1,3(6X,I4,5X,1H)	3MOD5610
170	FORMAT (1X,24HNORMAL DIAMETRAL PITCH	1,20X,F6.3,21X,1H)	3MOD5620
180	FORMAT (1X,24HTRANS. DIAMETRAL PITCH	1,20X,F6.3,21X,1H)	3MOD5630
190	FORMAT (1X,24HNORMAL PRESSURE ANGLE	1,21X,F4.1,22X,1H)	3MOD5640
200	FORMAT (1X,24HTRANS. PRESSURE ANGLE	1,21X,F4.1,22X,1H)	3MOD5650
210	FORMAT (1X,24HHELIX ANGLE	1,21X,F4.1,22X,1H)	3MOD5660
220	FORMAT (1X,24HGEAR RATIO	1,20X,F6.3,21X,1H)	3MOD5670
230	FORMAT (1X,24HPITCH DIAMETER	IN	3MOD5680
240	FORMAT (1X,24HEFFECTIVE FACEWIDTH	IN	3MOD5690
250	FORMAT (1X,24HF/DP	1,21X,F4.2,22X,1H)	3MOD5700
260	FORMAT (1X,24HCENTER DISTANCE	IN	3MOD5710
270	FORMAT (1X,24HPITCHLINE VELOCITY	FPM	3MOD5720
280	FORMAT (1X,24HTANGENTIAL LOAD	LB	3MOD5730


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290  FORMAT (1X,24H)TOOTH LOAD/IN      LB/IN  |,21X,F5.0,21X,1H|)
300  FORMAT (1X,24H)UNIT LOAD          PSI    |,20X,F6.0,21X,1H|)
310  FORMAT (1X,24H)MESH FREQUENCY     HZ     |,3(4X,F6.0,5X,1H|)|)
320  FORMAT (1X,24H)K FACTOR (COMPUTED) PSI   |,21X,F5.0,21X,1H|)
330  FORMAT (1X,24H)CONTACT STRESS     PSI   |,20X,F7.0,20X,1H|)
340  FORMAT (1X,24H)BENDING STRESS     PSI   |,20X,F7.0,20X,1H|)
350  FORMAT (1X,24H)TORQUE              K IN-LB |,3(4X,F7.1,4X,1H|)|)
360  FORMAT (1X,24H)HARDNESS RANGE     BHN   |,3(2X,I3,5H - ,I3,2X,1H|)|)
      END
3MOD5740
3MOD5750
3MOD5760
3MOD5770
3MOD5780
3MOD5790
3MOD5800
3MOD5810
3MOD5820

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Module Four

```
C      FUNCTION AGMAE1 (PHIN, II)
C
C      CODED BY:  LT J. L. PAQUETTE, USN      JAN 1982
C      NAVAL POSTGRADUATE SCHOOL  MONTEREY, CA 93940
C
C      SUBPROGRAM TO PROVIDE INTERPOLATION OF TABLE E-1 IN
C      AGMA 226.01, AUG 1970 FOR VALUES OF H, L, AND M USED
C      TO COMPUTE KF -- LAGRANGE INTERPOLATION USED
C
C      PHIN  NORMAL PRESSURE ANGLE IN RADIANs
C      II=1  INTERPOLATION OF H -- F (1) TO F (3)
C      II=2  INTERPOLATION OF L -- F (4) TO F (6)
C      II=3  INTERPOLATION OF M -- F (7) TO F (9)
C
C      DIMENSION F (9), A (3)
C
C      INITIALIZATION:
C      ARRAY F CONTAINS THE VALUES OF H, L, M FROM TABLE E-1
C
C      DATA F/0.22,0.18,0.14,0.20,0.15,0.11,0.40,0.45,0.50/
C      SUM=0.0
C      L=0
C
C      CONVERT PHIN TO DEGREES FOR USE IN INTERPOLATION
C
C      X=PHIN*180./ (4.*ATAN (1.))
C      A (1) = (X-20.)*(X-25.)/57.75
C      A (2) = (X-14.5)*(X-25.)/ (-27.5)
C      A (3) = (X-14.5)*(X-20.)/52.5
C      I = (II-1) *3+1
C      J =3*II
C      DO 10 K=I, J
C      L=L+1
```



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10 SUM=SUM+A(L) * F(K)
AGMAE1=SUM
RETURN
END
C
C
C
FUNCTION ARCCOS (A,B)
RATIO=A/B
ARG=SQRT (1.-RATIO*RATIO) /RATIO
ARCCOS=ATAN (ARG)
RETURN
END
C
C
C
FUNCTION ARCSIN (A,B)
RATIO=A/B
ARG=RATIO/SQRT (1.-RATIO*RATIO)
ARCSIN=ATAN (ARG)
RETURN
END
C
C
C
FUNCTION CKDATA (A)
CODED BY: LT J.L. PAQUETTE, USN JAN 1982
NAVAL POSTGRADUATE SCHOOL MONTEREY, CA 93940
SUBPROGRAM USED BY SUBROUTINE AGMA TO CHECK AND/OR MODIFY THE
PRE-PROGRAMMED AGMA CONSTANTS -- A VALUE OF ZERO (VAL=0.0) WILL
CAUSE THE PRE-PROGRAMMED CONSTANT TO REMAIN UNCHANGED.
READ (5,*) VAL
IF (VAL.NE.0.0) GO TO 10

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C          4MOD1060
C          4MOD1070
SUBROUTINE GFI (GEOMI, IRED, MG, DP, DG, I, II)
C          4MOD1080
C          4MOD1090
C          4MOD1100
C          4MOD1110
C          4MOD1120
C          4MOD1130
C          4MOD1140
C          4MOD1150
C          4MOD1160
C          4MOD1170
C          4MOD1180
C          4MOD1190
C          4MOD1200
C          4MOD1210
C          4MOD1220
C          4MOD1230
C          4MOD1240
C          4MOD1250
C          4MOD1260
C          4MOD1270
C          4MOD1280
C          4MOD1290
C          4MOD1300
C          4MOD1310
C          4MOD1320
C          4MOD1330
C          4MOD1340
C          4MOD1350
C          4MOD1360
C          4MOD1370
C          4MOD1380
C          4MOD1390
C          4MOD1400
C          4MOD1410

CODED BY:  LT J. L. PAQUETTE, USN          JAN 1982
           NAVAL POSTGRADUATE SCHOOL MONTEREY, CA 93940

SUBPROGRAM TO COMPUTE THE DURABILITY GEOMETRY FACTOR, I

OUTPUT VARIABLE:
GEOMI DURABILITY GEOMETRY FACTOR

INPUT VARIABLES:
IRED REDUCTION STAGE UNDER CONSIDERATION
MG REDUCTION RATIO (MG.GT. 1.0)
DP DIAMETER OF THE PINION
DG DIAMETER OF THE GEAR
III 0 FOR EXTERNAL GEARS; 1 FOR INTERNAL GEARS

EXTERNAL SUBPROGRAM(S) REQUIRED: FUNCTION SHRDL

REAL MG
COMMON /DESDAT/ PWRIN(2), RPMIN(2), RPMOUT, DHELIX(3), HELIX(3), PD(3),
1PND(3), DPHI(3), PHI(3), DPHIN(3), PHIN(3), NDIFF, IARR, IEPIC(3), IHARD(3),
2, 2), IOPRO, NPWRIN, IPWRSR(2), NRED, NPATH, NPLNT(3), NHELX
RATIO=MG/(MG+1.)
IF (III.EQ.1) RATIO=MG/(MG-1.)
TWO MN=2.*SHRDL(IRED, DP, DG)
GEOMI=SIN(PHI(IRED))*COS(PHI(IRED))**RATIO/TWO MN
RETURN
END
SUBROUTINE GFI (GEOMJ, IRED, DP, DG, IPG, I, II)

```



```

C      CODED BY:  LT J.L. PAQUETTE, USN          JAN 1982
C      NAVAL POSTGRADUATE SCHOOL  MONTEREY, CA 93940
C
C      SUBPROGRAM TO COMPUTE THE STRENGTH GEOMETRY FACTOR, J,
C      IAW AGMA 226.01, AUG 1970 -- ALL CALCULATIONS ARE BASED
C      ON ACTUAL GEOMETRY WITH NO REQUIREMENT TO SCALE THE
C      PROBLEM TO A NORMAL DIAMETRAL PITCH OF ONE (PND=1).
C      THE AGMA TOOTH FORM FACTOR, BASED ON A SCALE OF PND=1,
C      HAS BEEN APPROPRIATELY MODIFIED.
C
C      OUTPUT VARIABLE:
C      GEOMJ  STRENGTH GEOMETRY FACTOR
C
C      INPUT VARIABLES:
C      IRED  REDUCTION STAGE UNDER CONSIDERATION
C      DP    DIAMETER OF THE PINION
C      DG    DIAMETER OF THE GEAR
C      IPG   GEAR FOR WHICH GEOMJ IS COMPUTED:
C           IPG=1  GEOMJ FOR PINION
C           IPG=2  GEOMJ FOR GEAR
C      III   0 FOR EXTERNAL GEARS; 1 FOR INTERNAL GEARS
C
C      EXTERNAL SUBPROGRAM(S) REQUIRED:  FUNCTION AGMAE1, FUNCTION ARCCOS,
C      FUNCTION ARCSIN, FUNCTION FALFA, FUNCTION RTFNDR, FUNCTION
C      SHRLD, FUNCTION THICK
C
C      REAL INV, KF, MN
C      DIMENSION CNST(5)
C      COMMON /DES DAT/ PWRIN(2), RPMIN(2), RPMOUT, DHELIX(3), HELIX(3), PD(3),
1PND(3), DPHI(3), PHI(3), DPHIN(3), PHIN(3), NDIFP, IARR, IEPIC(3), IHARD(3),
2,2), IOPRO, NPWRIN, IPWRSR(2), NRED, NPATH, NPLNT(3), NHELX
C      EXTERNAL FALFA
C      INV(Z) = TAN(Z) - Z
C
C      INITIALIZATION
C
C      4MOD1420
C      4MOD1430
C      4MOD1440
C      4MOD1450
C      4MOD1460
C      4MOD1470
C      4MOD1480
C      4MOD1490
C      4MOD1500
C      4MOD1510
C      4MOD1520
C      4MOD1530
C      4MOD1540
C      4MOD1550
C      4MOD1560
C      4MOD1570
C      4MOD1580
C      4MOD1590
C      4MOD1600
C      4MOD1610
C      4MOD1620
C      4MOD1630
C      4MOD1640
C      4MOD1650
C      4MOD1660
C      4MOD1670
C      4MOD1680
C      4MOD1690
C      4MOD1700
C      4MOD1710
C      4MOD1720
C      4MOD1730
C      4MOD1740
C      4MOD1750
C      4MOD1760
C      4MOD1770

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4MOD1780
4MOD1790
4MOD1800
4MOD1810
4MOD1820
4MOD1830
4MOD1840
4MOD1850
4MOD1860
4MOD1870
4MOD1880
4MOD1890
4MOD1900
4MOD1910
4MOD1920
4MOD1930
4MOD1940
4MOD1950
4MOD1960
4MOD1970
4MOD1980
4MOD1990
4MOD2000
4MOD2010
4MOD2020
4MOD2030
4MOD2040
4MOD2050
4MOD2060
4MOD2070
4MOD2080
4MOD2090
4MOD2100
4MOD2110
4MOD2120
4MOD2130

IF (IPG.EQ.1) III=0
PI=4.*ATAN(1.)
D=DP
IF (IPG.EQ.2) D=DG
C
A=ADDENDUM; B=DEDENDUM; TT=ARC THICKNESS OF THE TOOTH
AT THE PITCH DIAMETER, D
C
A=1./PD(IRED)
B=1.25/PD(IRED)
TT=PI*COS(HELIX(IRED))/(2.*PD(IRED))
C
RF=ROOT FILLET RADIUS; RT=RADIUS OF TIP OF GENERATING TOOL
(ASSUME AVERAGE VALUE OF 0.28/PND FOR RT)
C
RT=.28/PND(IRED)
BMRT=B-RT
RF=RT+BMRT*BMRT/(D/(2.*COS(HELIX(IRED)))*COS(HELIX(IRED)))+BMRT)
C
D=PITCH DIAMETER; DR=ROOT DIAMETER; DO=OUTSIDE DIAMETER;
DB=BASE DIAMETER; DI=INSIDE DIAMETER (INTERNAL GEARS ONLY)
C
DR=D-2.*B
DO=D+2.*A
DB=D*COS(PHI(IRED))
DI=D-2.*A
C
TC=CHORDAL THICKNESS OF THE TOOTH AT THE PITCH DIAMETER;
EPS=ANGLE FROM THE CENTERLINE OF THE TOOTH TO A POINT WHERE
THE INVOLUTE CURVE CROSSES THE BASE CIRCLE; TTIP=THICKNESS
OF THE TOOTH AT ITS TIP
C
TC=TT-(TT*TT*TT*COS(HELIX(IRED)))*COS(HELIX(IRED))/(6.*D*D)
EPS=INV(PHI(IRED))+ARCSIN(TC,D)
IF (III.EQ.1) EPS=ARCSIN(TC,D)-INV(PHI(IRED))
DTIP=DO
C

```



```

4MOD2140
4MOD2150
4MOD2160
4MOD2170
4MOD2180
4MOD2190
4MOD2200
4MOD2210
4MOD2220
4MOD2230
4MOD2240
4MOD2250
4MOD2260
4MOD2270
4MOD2280
4MOD2290
4MOD2300
4MOD2310
4MOD2320
4MOD2330
4MOD2340
4MOD2350
4MOD2360
4MOD2370
4MOD2380
4MOD2390
4MOD2400
4MOD2410
4MOD2420
4MOD2430
4MOD2440
4MOD2450
4MOD2460
4MOD2470
4MOD2480
4MOD2490

IF (III.EQ.1) DTIP=DI
TTIP=THICK(D, DB, DTIP, TT, III)

C
C PHILN=NORMAL LOAD PRESSURE ANGLE AT THE TIP OF THE TOOTH;
C RV=RADIUS TO THE TIP OF THE STRESS PARABOLA ON THE TOOTH
C CENTERLINE; XC=X-COORD OF THE CENTER OF THE ROOT FILLET;
C YC=Y-COORD OF THE CENTER OF THE ROOT FILLET (BOTH WITH
C RESPECT TO THE TOOTH CENTERLINE)
C
PHILN=ARCCOS(DB, DTIP) - (TTIP/DTIP)
IF (III.EQ.1) PHILN=ARCCOS(DB, DTIP) + (TTIP/DTIP)
RV=DB/(2.*COS(PHILN))

C
C COMPUTE XC AND YC
C
HYP=(DR/2.)+RF
RB=DB/2.
IF (HYP-RB) 10, 20, 20
XC=RF+HYP*SIN(EPS)
YC=SQRT(HYP*HYP-XC*XC)
GO TO 30

20 PH11=ARCCOS(RB, HYP)
OPP1=HYP*SIN(PH11)
OPP2=OPP1-RF
HYP1=SQRT(OPP2*OPP2+RB*RB)
PH12=ARCCOS(RB, HYP1)
DELTA=EPS+PH11-PH12-INV(PH12)
IF (III.EQ.1) DELTA=EPS+PH11-PH12+INV(PH12)
XC=HYP*SIN(DELTA)
YC=HYP*COS(DELTA)

C
C SOLVE FOR THE ANGLE ALPHA USED TO COMPUTE VALUES OF T, H,
C AND X WHICH ARE 'COMPUTED MEASUREMENTS' FROM THE TOOTH
C LAYOUT (NOTE: T=2*CAPT)
C ARRAY CNST IS A PARAMETER LIST OF CONSTANTS REQUIRED FOR
C THE EVALUATION OF THE FUNCTION OF ALPHA

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RAT2=T/H
KF=AH+(RAT12AL)* (RAT22AM)
MN=LOAD SHARING RATIO
MN=SHRLD(IRED,DP,DG)
GEOMETRY FACTOR J
GEOMJ=YSMC2COS(HELIX(IRED))*COS(HELIX(IRED))/(KFMN)
RETURN
END
FUNCTION POWERB (RPM,FACE,D,IRED,IPWR,IGR,GEOMJ)
CODED BY: LT J.L. PAQUETTE, USN JAN 1982
NAVAL POSTGRADUATE SCHOOL MONTEREY, CA 93940
SUBPROGRAM TO COMPUTE THE ALLOWABLE SERVICE POWER FOR A GEAR PAIR
BASED ON AGMA'S STRENGTH RATING (AGMA 221.02 JUL 1965)
RPM PINION SPEED IN RPM
FACE FACEWIDTH OF THE GEAR PAIR IN INCHES
D PITCH DIAMETER OF THE PINION IN INCHES
GEOMJ STRENGTH GEOMETRY FACTOR, J
IRED REDUCTION STAGE UNDER CONSIDERATION
IPWR POWER SOURCE IDENTIFICATION
IGR GEAR IDENTIFICATION (1=PINION, 2=GEAR)
COMMON /AGMAB/ SFB(2,2), AKV,AKS,AKM,AKO(2), SAT(6), AKL(2), AKR(6), AK4MOD(3170
1T
COMMON /DESDAT/ PWRIN(2), RPMIN(2), RPMOUT,DHELIX(3), HELIX(3), PD(3), 4MOD(3180
1PND(3),DPHI(3),PHI(3),DPHN(3),PHIN(3),NDIFF,IARR,IEPIC(3), IHARD(34MOD(3200
2,2), IOPRO,NPWRIN,IPWRSR(2),NRED,NPATH,NPLNT(3), NHELX
4MOD(3210
4MOD(2860
4MOD(2870
4MOD(2880
4MOD(2890
4MOD(2900
4MOD(2910
4MOD(2920
4MOD(2930
4MOD(2940
4MOD(2950
4MOD(2960
4MOD(2970
4MOD(2980
4MOD(2990
4MOD(3000
4MOD(3010
4MOD(3020
4MOD(3030
4MOD(3040
4MOD(3050
4MOD(3060
4MOD(3070
4MOD(3080
4MOD(3090
4MOD(3100
4MOD(3110
4MOD(3120
4MOD(3130
4MOD(3140
4MOD(3150
4MOD(3160
4MOD(3170
4MOD(3180
4MOD(3190
4MOD(3200
4MOD(3210

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```

C          4 MOD 3220
C          4 MOD 3230
C          4 MOD 3240
C          4 MOD 3250
C          4 MOD 3260
C          4 MOD 3270
C          4 MOD 3280
C          4 MOD 3290
C          4 MOD 3300
C          4 MOD 3310
C          4 MOD 3320
C          4 MOD 3330
C          4 MOD 3340
C          4 MOD 3350
C          4 MOD 3360
C          4 MOD 3370
C          4 MOD 3380
C          4 MOD 3390
C          4 MOD 3400
C          4 MOD 3410
C          4 MOD 3420
C          4 MOD 3430
C          4 MOD 3440
C          4 MOD 3450
C          4 MOD 3460
C          4 MOD 3470
C          4 MOD 3480
C          4 MOD 3490
C          4 MOD 3500
C          4 MOD 3510
C          4 MOD 3520
C          4 MOD 3530
C          4 MOD 3540
C          4 MOD 3550
C          4 MOD 3560
C          4 MOD 3570

INITIALIZATION

IH=IHARD(IRED,IGR)
IP=IPRSR(IPWR)
NPTH=NPATH
IF ((NRED.EQ.3) .AND. (IRED.GE.2)) NPTH=2

COMPUTE ALLOWABLE SERVICE POWER

ANUM=RPMD*AKV*GEOMJ*SAT (IH)*AKL (IOPRO)
DEN=SFBIOPRO,IP)*AKO(NPTH)*AKM*AKS*AKR (IH)*AKT*PPD (IRED)
POWERB=FACE*ANUM/(126050.*DEN)
RETURN
END

FUNCTION POWERH (RPM,FACE,D,IRED,IPWR,GEOMI)
CODED BY:  LT J.L. PAQUETTE, USN          JAN 1982
           NAVAL POSTGRADUATE SCHOOL  MONTEREY, CA 93940

SUBPROGRAM TO COMPUTE THE ALLOWABLE SERVICE POWER OF A GEAR PAIR
BASED ON AGMA'S DURABILITY RATING (AGMA 211.02 SEPT 1966)

RPM      PINION SPEED IN RPM
FACE     FACEWIDTH OF THE GEAR PAIR IN INCHES
D        PITCH DIAMETER OF THE PINION IN INCHES
IRED     REDUCTION STAGE UNDER CONSIDERATION
IPWR     POWER SOURCE IDENTIFICATION
GEOMI    DURABILITY GEOMETRY FACTOR, I

COMMON /AGMAH/ SFH (2,2),CV (3),CS,CM (2),CF,CO (2),SAC (6),CP,CL (2),CH
1,CT,CR (6)
COMMON /DES DAT/ PWRIN (2),RPMIN (2),RPMOUT,DHELIX (3),HELIX (3),PD (3),

```



```

1PND(3),DPHI(3),PHI(3),DPHIN(3),PHIN(3),NDIFF,IARR,IEPIC(3),IHARD(34MOD3580
2,2),IOPRO,NPWRIN,IPWRSR(2),NRED,NPATH,NPLNT(3),NHELX
4MOD3590
4MOD3600
4MOD3610
4MOD3620
4MOD3630
4MOD3640
4MOD3650
4MOD3660
4MOD3670
4MOD3680
4MOD3690
4MOD3700
4MOD3710
4MOD3720
4MOD3730
4MOD3740
4MOD3750
4MOD3760
4MOD3770
4MOD3780
4MOD3790
4MOD3800
4MOD3810
4MOD3820
4MOD3830
4MOD3840
4MOD3850
4MOD3860
4MOD3870
4MOD3880
4MOD3890
4MOD3900
4MOD3910
4MOD3920
4MOD3930

```

C INITIALIZATION

C IH=IHARD(IRED,1)

C IP=IPWRSR(IPWR)

C COMPUTE ALLOWABLE SERVICE POWER

C BRAC=SAC(IH)*D*CL(IOPRO)*CH/(CP*CT*CR(IH))

C ANUM=RP*GEOMI*CV(IRED)*BRAC*BRAC

C DEN=SFH(IOPRO,IP)*CS*CM(IOPRO)*CF*CO(IOPRO)

C POWERH=FACE*ANUM/(126050.*DEN)

C RETURN

C END

C FUNCTION RNDGEN (RO)

C PI=3.14159286745534

C A=(PI+RO)*5.

C I=IFIX(A)

C RNDGEN=A-FLOAT(I)

C RETURN

C END

C FUNCTION RTFNDR (AX,BX,F,CNST,TOL)

C THIS FUNCTION SUBPROGRAM IS A SLIGHTLY MODIFIED

C VERSION OF THE FUNCTION SUBPROGRAM ZEROIN GIVEN IN

C FORSYTHE, MALCOLM, AND MOLER, COMPUTER METHODS FOR

C MATHEMATICAL COMPUTATIONS, PRENTICE-HALL, INC. (1977).

C LT J. L. PAQUETTE, USN JAN 1982


```

E=D
30 IF (ABS(FC).GE.ABS(FB)) GO TO 40
A=B
B=C
C=A
FA=FB
FB=FC
FC=FA
C
C CONVERGENCE TEST
C
40 TOL1=2.*EPS*ABS(B)+.5*TOL
XM=.5*(C-B)
IF (ABS(XM).LE.TOL1) GO TO 90
IF (FB.EQ.0.0) GO TO 90
C
C BISECTION REQUIREMENT TEST
C
C IF (ABS(E).LT.TOL1) GO TO 70
IF (ABS(FA).LE.ABS(FB)) GO TO 70
C
C QUADRATIC INTERPOLATION TEST
C
C IF (A.NE.C) GO TO 50
C
C LINEAR INTERPOLATION
C
C S=FB/FA
P=2.*XM*S
Q=1.-S
GO TO 60
C
C INVERSE QUADRATIC INTERPOLATION
C
C Q=FA/FC
R=FB/FC
50
4MOD4300
4MOD4310
4MOD4320
4MOD4330
4MOD4340
4MOD4350
4MOD4360
4MOD4370
4MOD4380
4MOD4390
4MOD4400
4MOD4410
4MOD4420
4MOD4430
4MOD4440
4MOD4450
4MOD4460
4MOD4470
4MOD4480
4MOD4490
4MOD4500
4MOD4510
4MOD4520
4MOD4530
4MOD4540
4MOD4550
4MOD4560
4MOD4570
4MOD4580
4MOD4590
4MOD4600
4MOD4610
4MOD4620
4MOD4630
4MOD4640
4MOD4650

```



```

S=FB/FA
P=S*(2.*XM*Q*(Q-R)-(E-A)*(R-1.))
Q=(Q-1.)*(R-1.)*(S-1.)
C
C ADJUST SIGNS
C
60 IF (P.GT.0.0) Q=-Q
P=ABS(P)
C
C INTERPOLATION ACCEPTABILITY TEST
C
T1=2.*P
T2=3.*XM*Q-ABS(TOL1*Q)
T3=ABS(.5*E*Q)
IF (T1.GE.T2) GO TO 70
IF (P.GE.T3) GO TO 70
E=D
D=P/Q
GO TO 80
C
C BISECTION
C
70 D=XM
E=D
C
C COMPLETE STEP
C
80 A=B
FA=FB
IF (ABS(D).GT.TOL1) B=B+D
IF (ABS(D).LE.TOL1) B=B+SIGN(TOL1, XM)
FB=F(B, CNST)
T1=FB*(FC/ABS(FC))
IF (T1.GT.0.0) GO TO 20
GO TO 30
C

```

```

4MOD4660
4MOD4670
4MOD4680
4MOD4690
4MOD4700
4MOD4710
4MOD4720
4MOD4730
4MOD4740
4MOD4750
4MOD4760
4MOD4770
4MOD4780
4MOD4790
4MOD4800
4MOD4810
4MOD4820
4MOD4830
4MOD4840
4MOD4850
4MOD4860
4MOD4870
4MOD4880
4MOD4890
4MOD4900
4MOD4910
4MOD4920
4MOD4930
4MOD4940
4MOD4950
4MOD4960
4MOD4970
4MOD4980
4MOD4990
4MOD5000
4MOD5010

```



```

C ROUTINE COMPLETED 4MOD5020
C 4MOD5030
90 RTFNDR=B 4MOD5040
RETURN 4MOD5050
END 4MOD5060
C 4MOD5070
C 4MOD5080
C 4MOD5090
FUNCTION SHRLD (IRED,DP,DG) 4MOD5100
C 4MOD5110
C CODED BY: LT J.L. PAQUETTE, USN JAN 1982 4MOD5120
C NAVAL POSTGRADUATE SCHOOL MONTEREY, CA 93940 4MOD5130
C 4MOD5140
C SUBPROGRAM TO COMPUTE THE LOAD SHARING RATIO, MN=PN/(.95*Z) 4MOD5150
C 4MOD5160
C IRED REDUCTION STAGE UNDER CONSIDERATION 4MOD5170
C DP DIAMETER OF THE PINION 4MOD5180
C DG DIAMETER OF THE GEAR 4MOD5190
C 4MOD5200
COMMON /DESDAT/ PWRIN(2),RPMIN(2),RPMOUT,DHELIX(3),HELIX(3),PD(3), 4MOD5210
1PND(3),DPHI(3),PHI(3),DPHIN(3),PHIN(3),NDIFF,IARR,IEPIC(3),IHARD(3) 4MOD5220
2,2),IOPRO,NPWRIN,IPWRSR(2),NKED,NPATH,NPLNT(3),NHELX 4MOD5230
PI=4.*ATAN(1.) 4MOD5240
PN=(PI/PND(IRED))*COS(PHIN(IRED)) 4MOD5250
DGB=DG*COS(PHI(IRED)) 4MOD5260
DPB=DP*COS(PHI(IRED)) 4MOD5270
DGO=DG+(2./PD(IRED)) 4MOD5280
DPO=DP+(2./PD(IRED)) 4MOD5290
Z=.5*(SQRT(DGO*DGO-DGB*DGB)+SQRT(DPO*DPO-DPB*DPB))-SQRT(DG*DG-DGB*DGB) 4MOD5300
1GB)-SQRT(DP*DP-DPB*DPB)) 4MOD5310
SHRLD=PN/(.95*Z) 4MOD5320
RETURN 4MOD5330
END 4MOD5340
C 4MOD5350
C 4MOD5360
C 4MOD5370

```



```

C      FUNCTION THICK (D,DB,DT,T,III)
C      CODED BY:  LT J.L. PAQUETTE, USN      JAN 1982
C      NAVAL POSTGRADUATE SCHOOL  MONTEREY, CA 93940
C
C      SUBPROGRAM TO COMPUTE THE THICKNESS OF A TOOTH BASED ON
C      A THICKNESS KNOWN AT A SPECIFIED DIAMETER
C
C      D  REFERENCE DIAMETER OF KNOWN THICKNESS (USUALLY PITCH DIAM.)
C      DB  BASE DIAMETER OF THE GEAR
C      DT  DIAMETER AT WHICH THE THICKNESS IS TO BE COMPUTED
C      T   KNOWN THICKNESS AT REFERENCE DIAMETER
C      III 0 FOR EXTERNAL GEARS; 1 FOR INTERNAL GEARS
C
C      EXTERNAL SUBPROGRAM(S) REQUIRED:  FUNCTION ARCCOS
C
C      REAL INV
C      INV(X)=TAN(X)-X
C      PHI=ARCCOS(DB,D)
C      PHIT=ARCCOS(DB,DT)
C      IF (III.EQ.1) GO TO 10
C      THICK=DT*( (T/D)+INV(PHI)-INV(PHIT) )
C      RETURN
C      THICK=DT*( (T/D)-INV(PHI)+INV(PHIT) )
C      RETURN
C      END
10

```

```

4MOD5380
4MOD5390
4MOD5400
4MOD5410
4MOD5420
4MOD5430
4MOD5440
4MOD5450
4MOD5460
4MOD5470
4MOD5480
4MOD5490
4MOD5500
4MOD5510
4MOD5520
4MOD5530
4MOD5540
4MOD5550
4MOD5560
4MOD5570
4MOD5580
4MOD5590
4MOD5600
4MOD5610
4MOD5620
4MOD5630

```


APPENDIX E

FIGURES

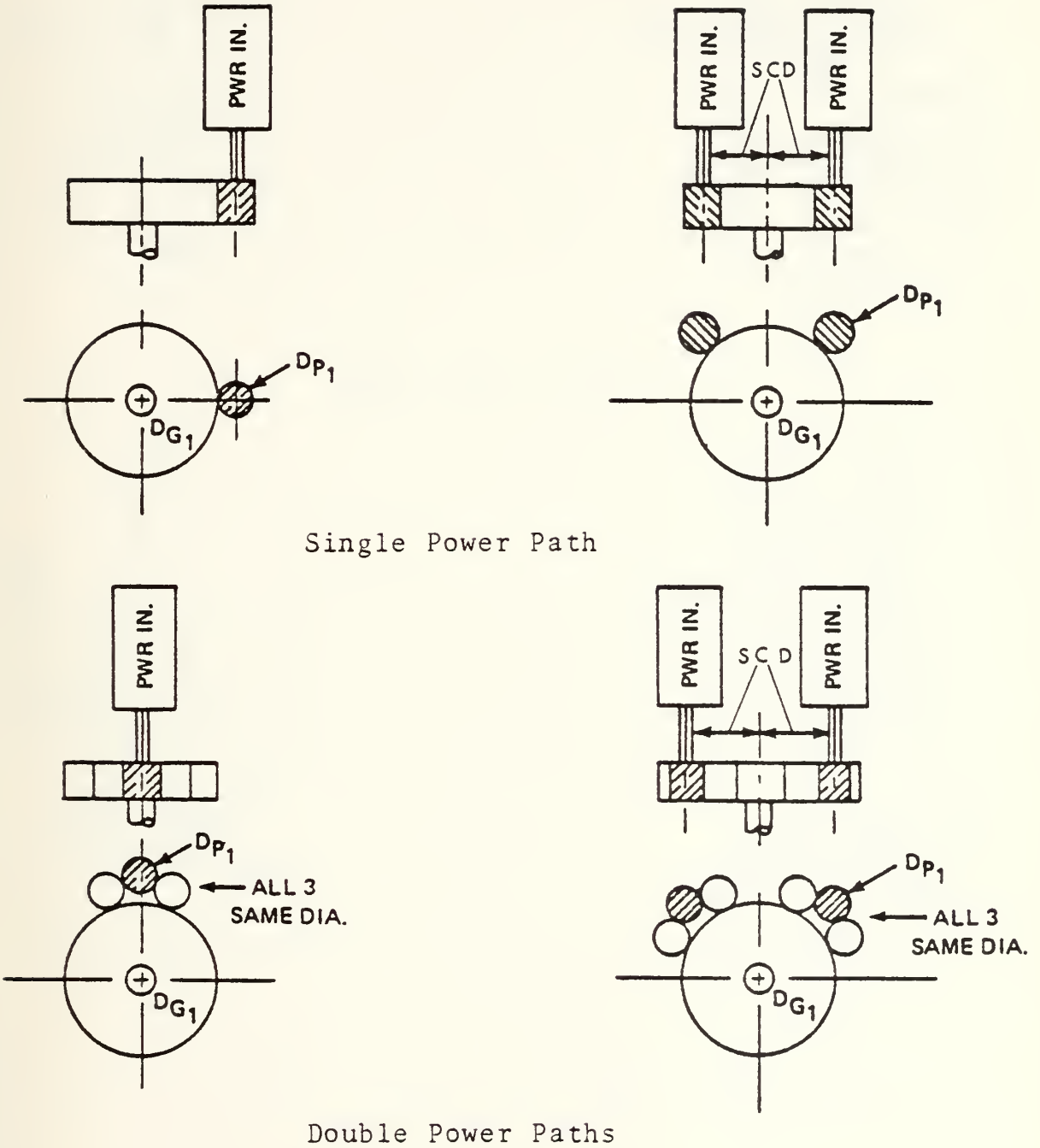
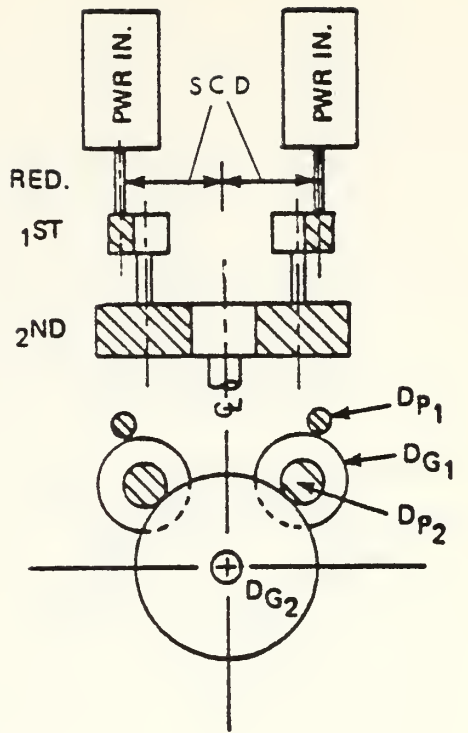
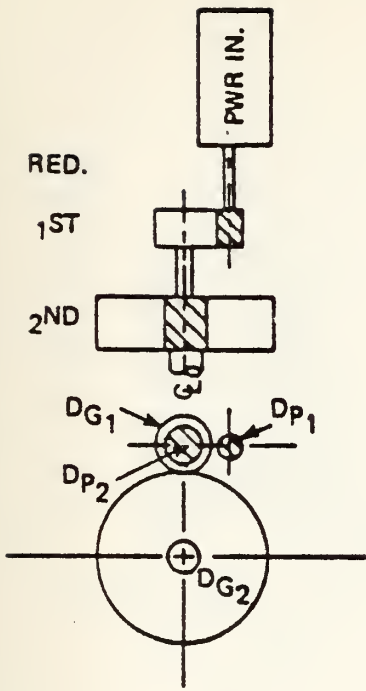
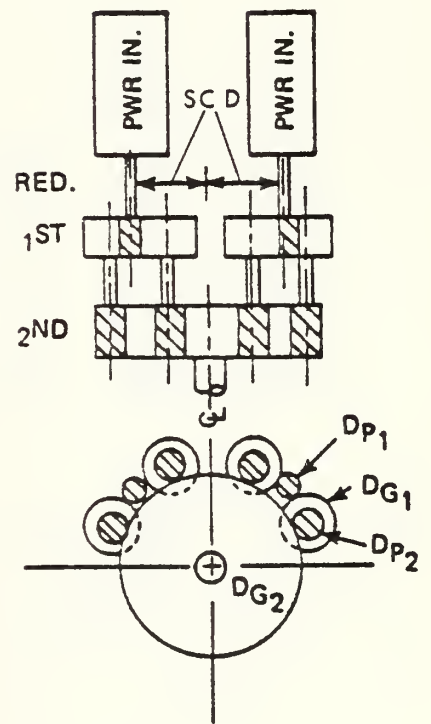
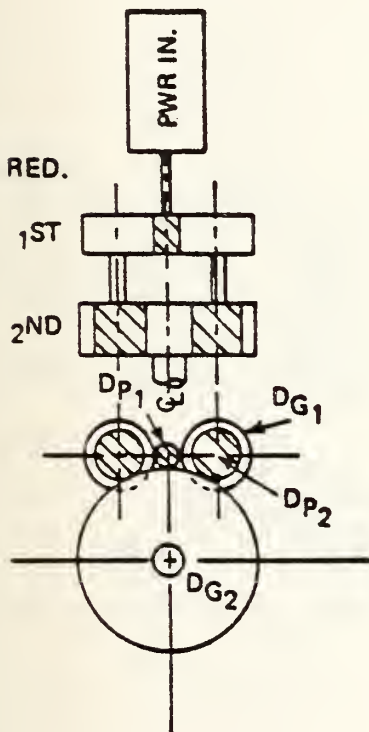


Figure 1: Single Reduction Parallel Axis Arrangements

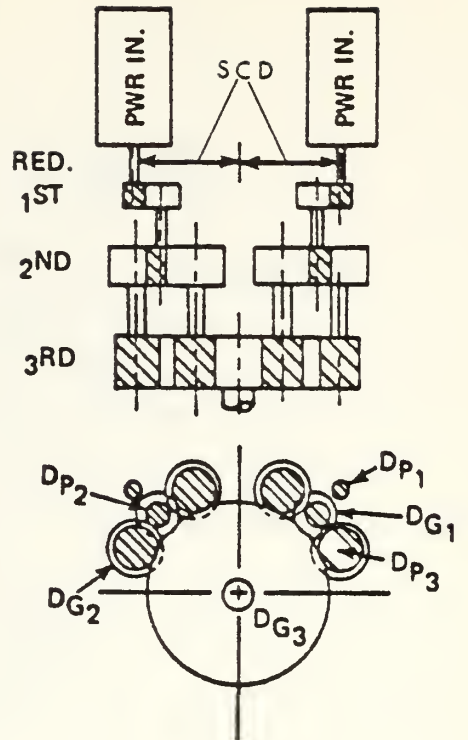
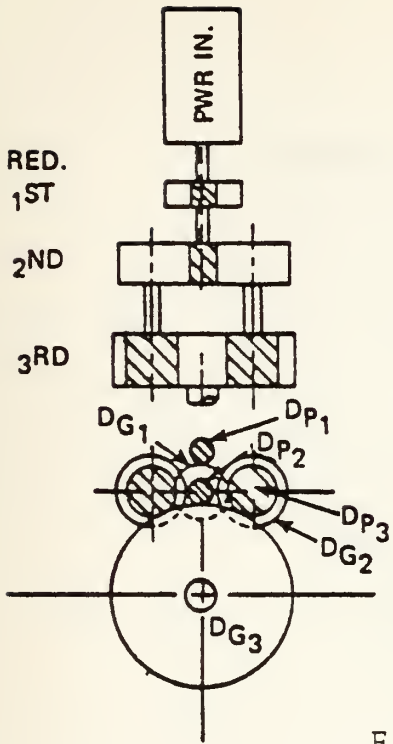


Single Power Path

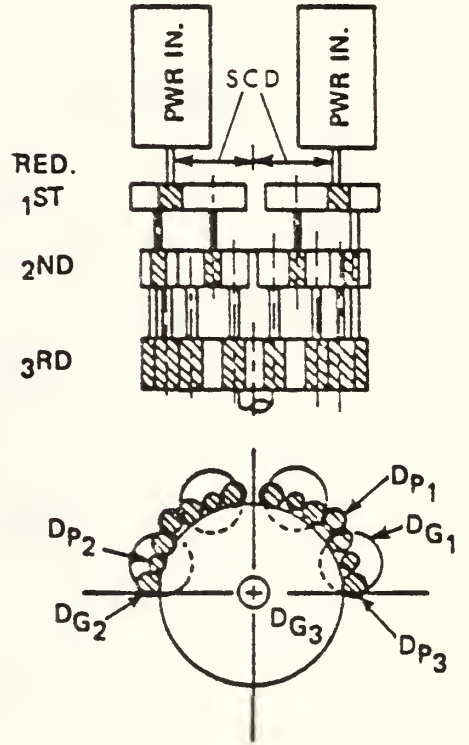
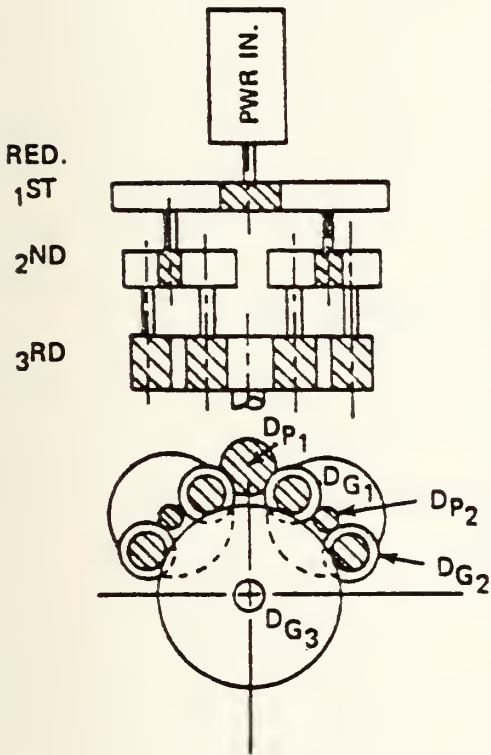


Double Power Path

Figure 2: Double Reduction Parallel Axis Arrangements

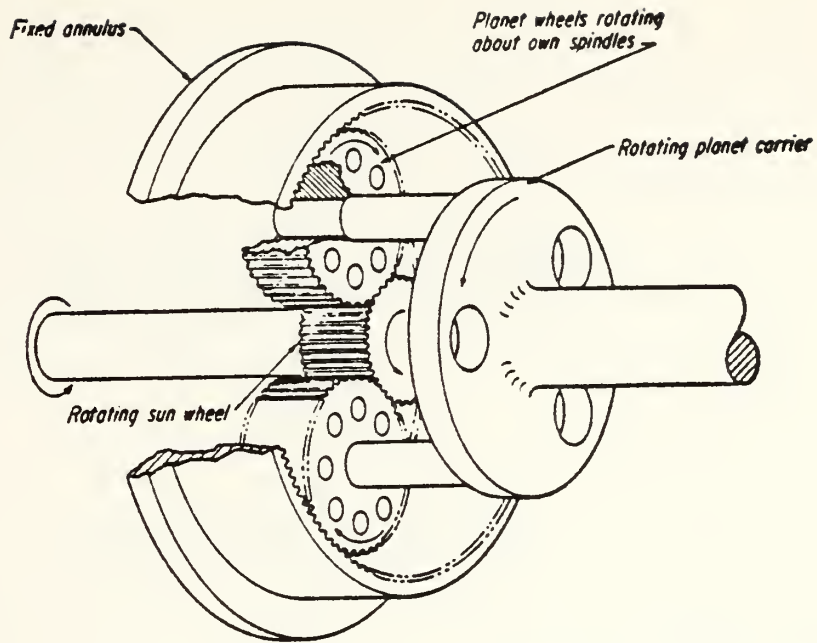


First Reduction
Single Power Path

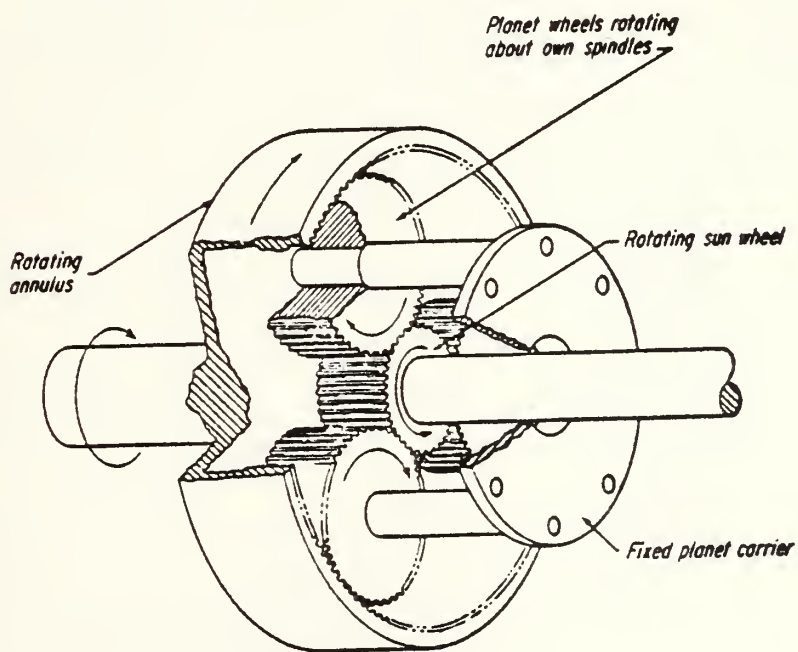


Double Power Paths

Figure 3: Triple Reduction Parallel Axis Arrangements



Planetary



Star

Figure 4: Single Reduction Epicyclic Arrangements (from Ref. 7)

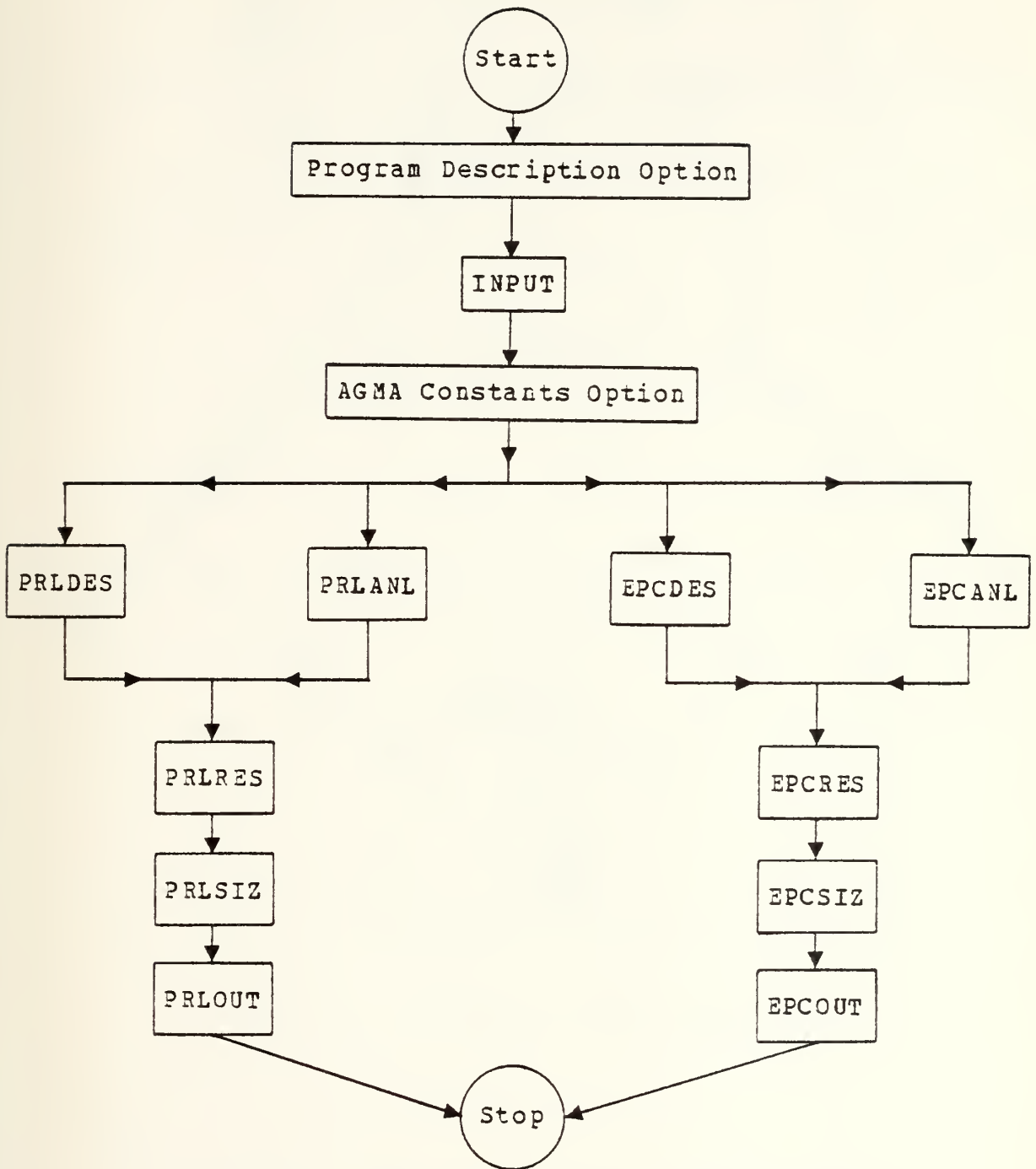


Figure 5: Flow Chart of the REGAD Package

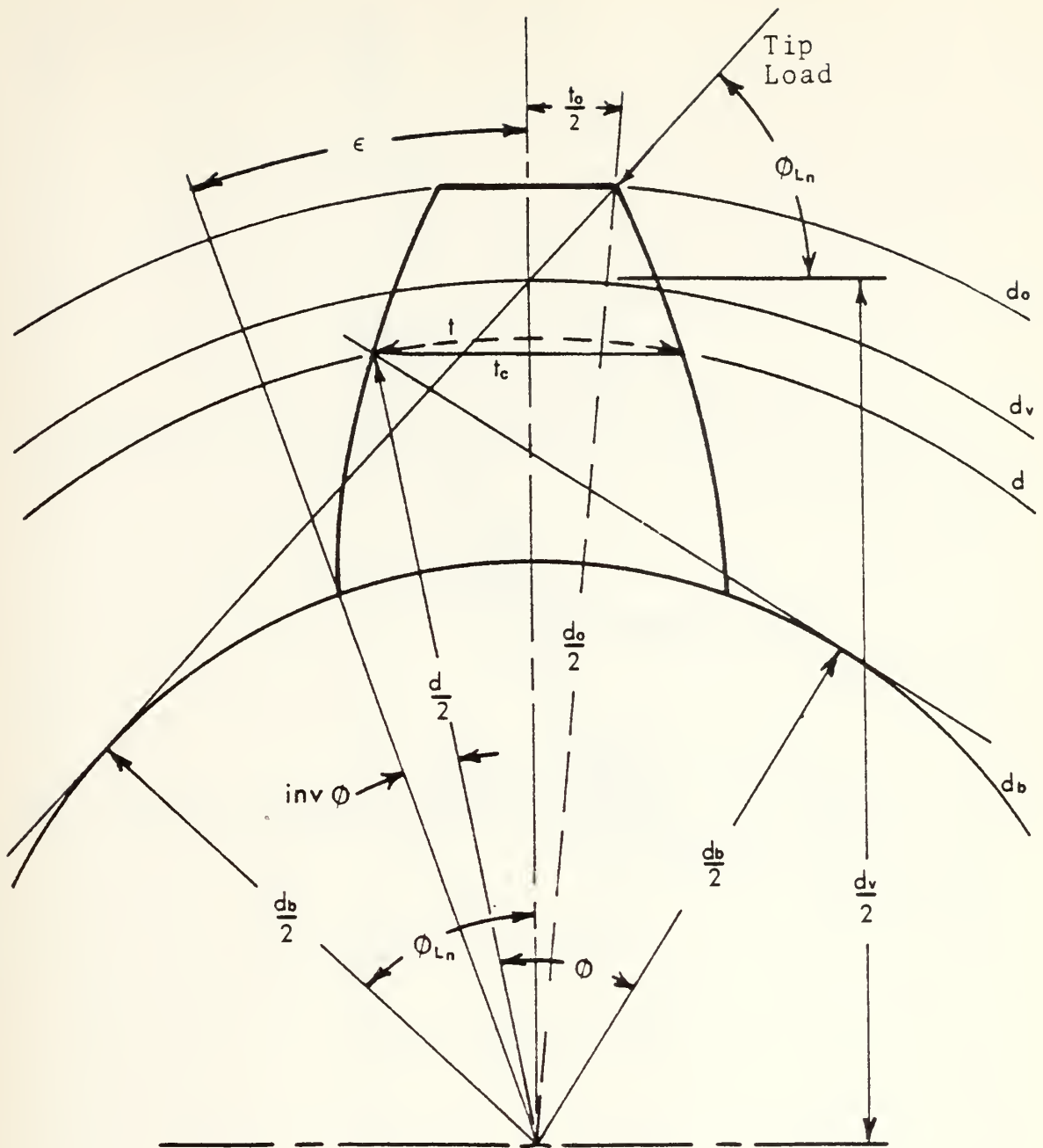


Figure 6: External Tooth Dimensions

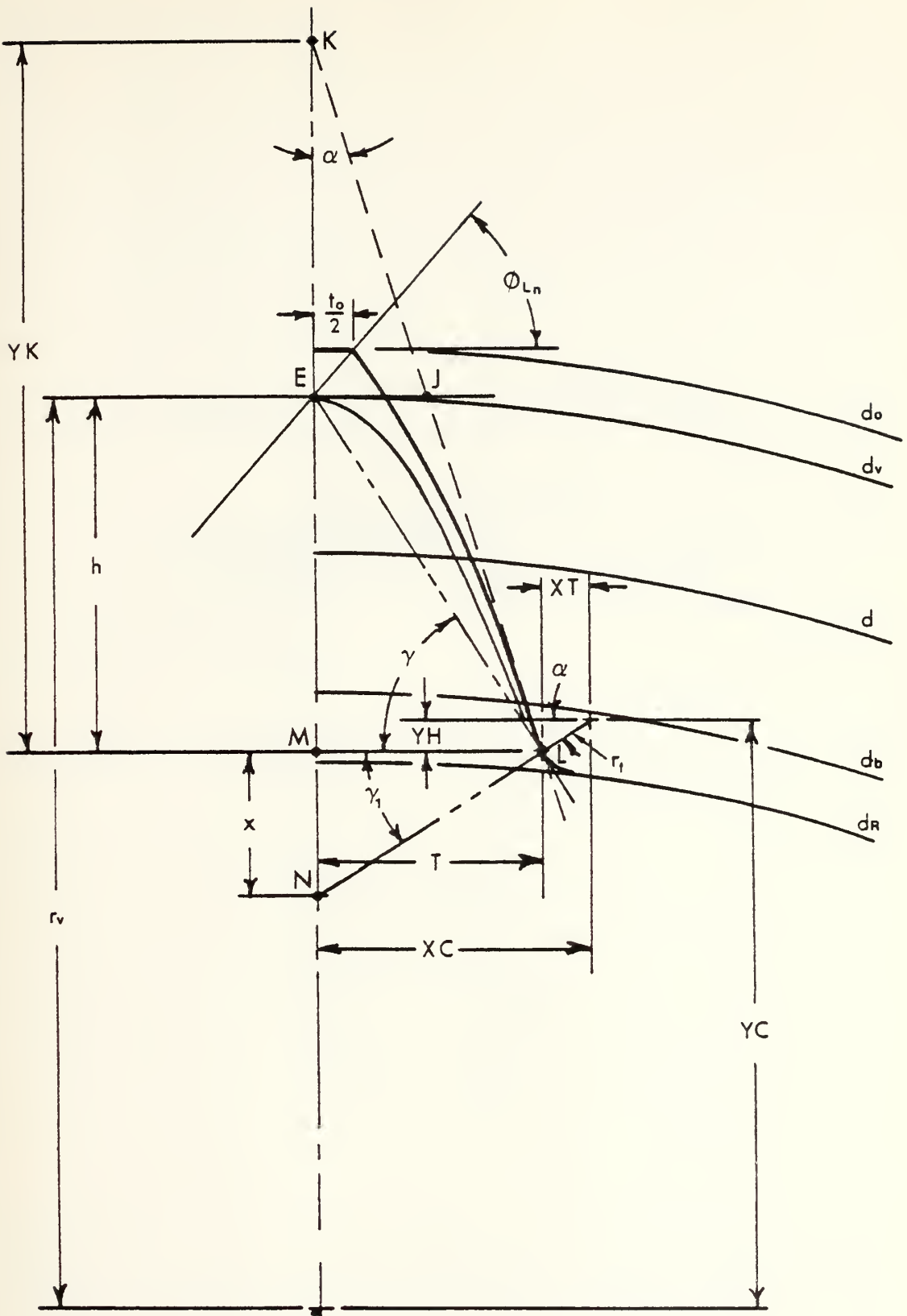


Figure 7: External Tooth Form Layout

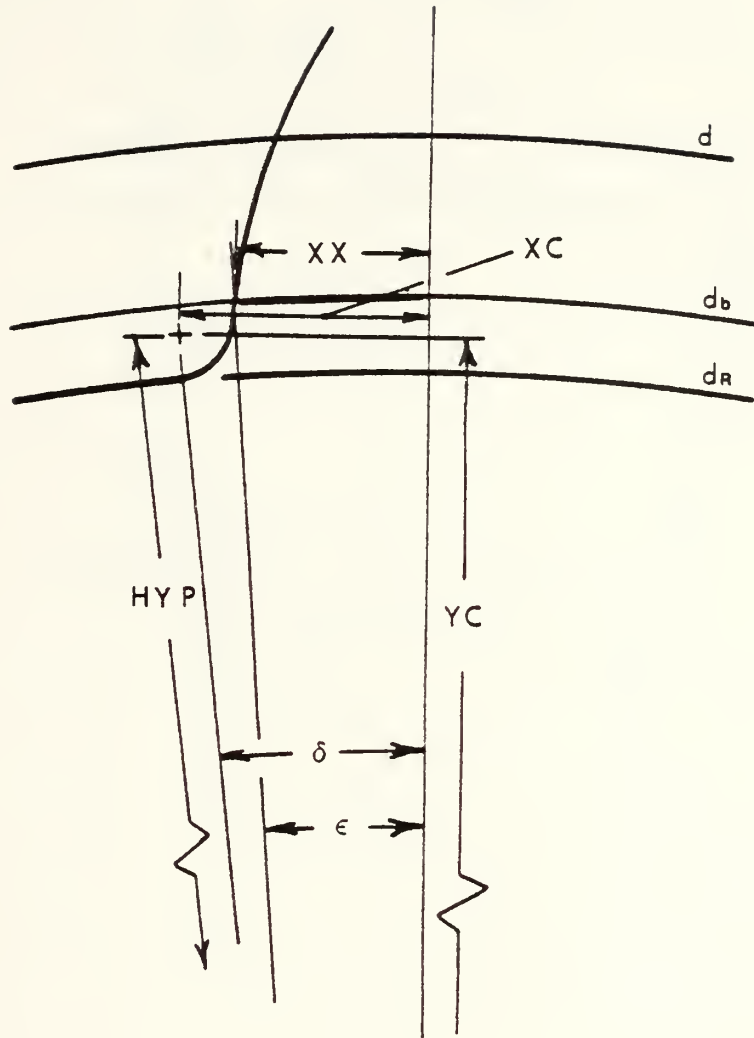


Figure 8: Fillet Center Location - Inside Base Circle

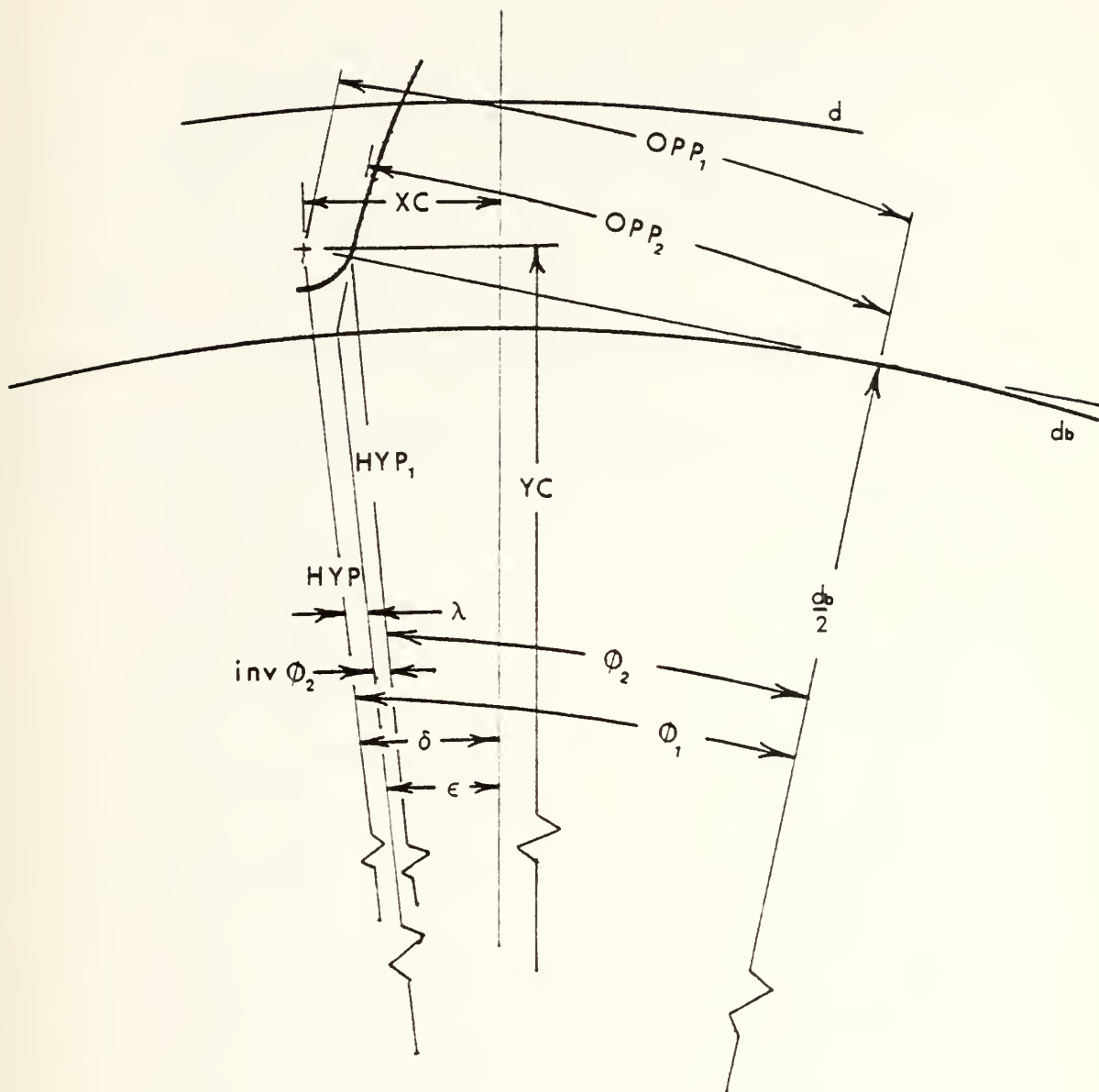


Figure 9: Fillet Center Location - Outside Base Circle

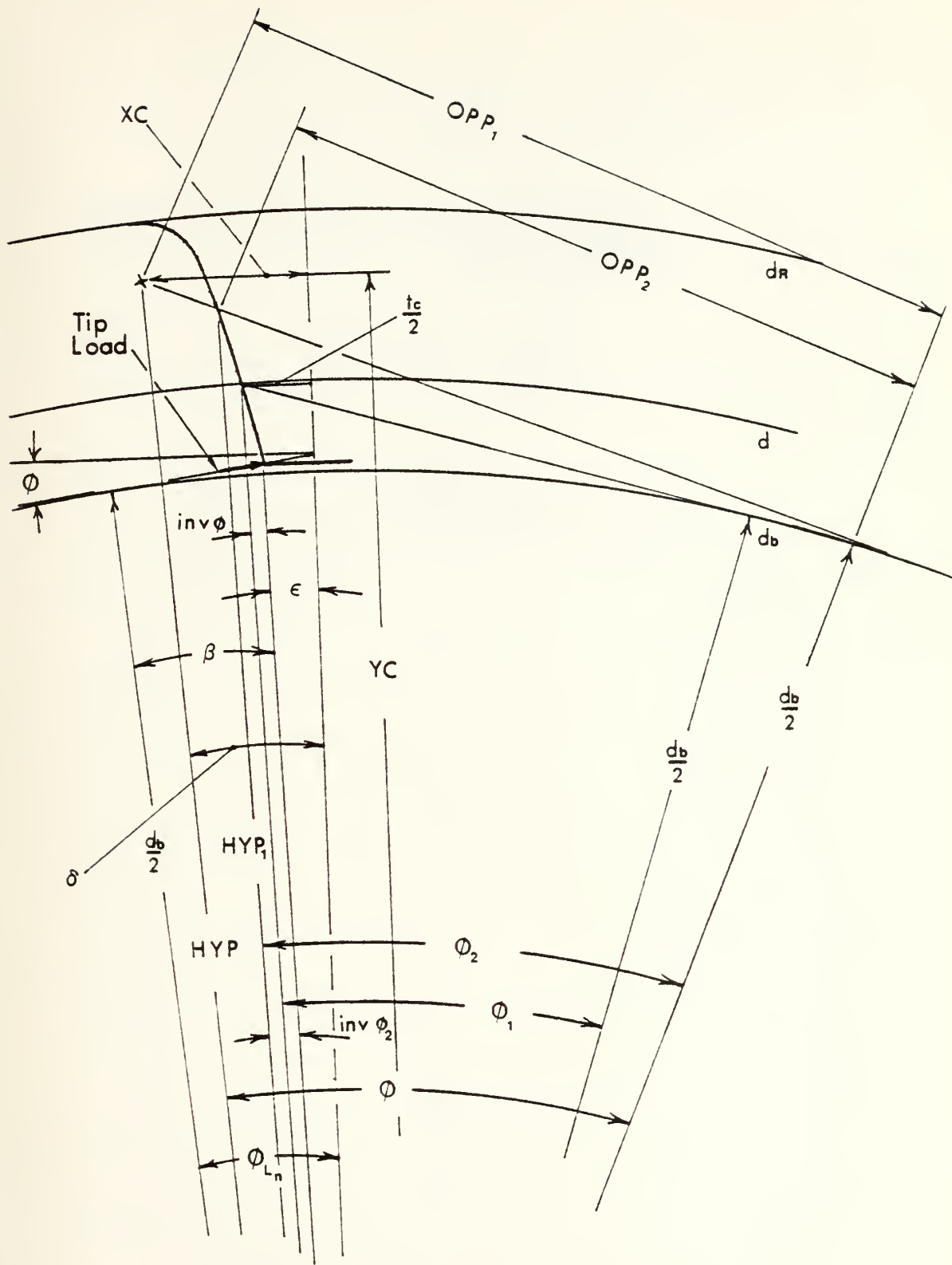


Figure 10: Internal Tooth Dimensions

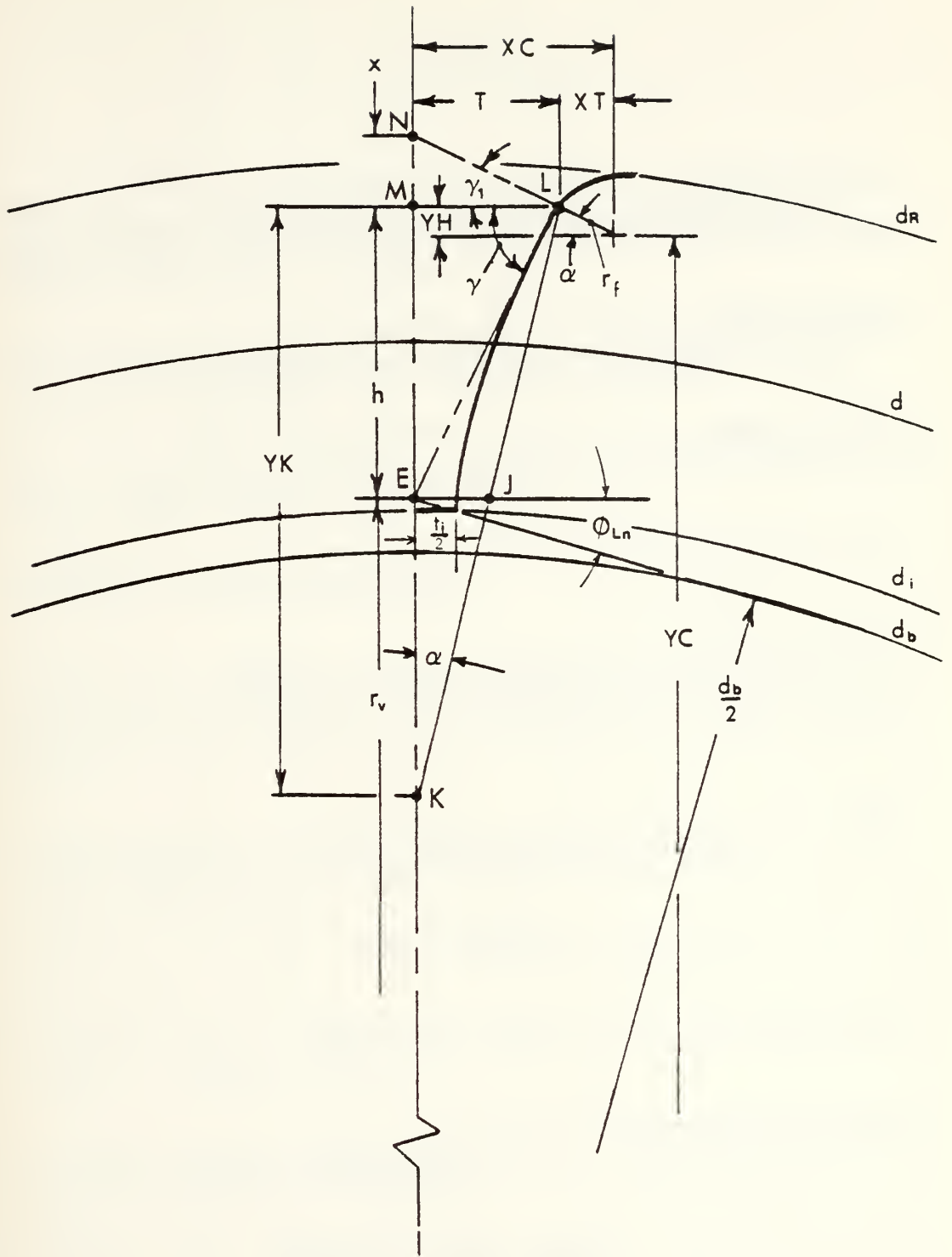


Figure 11: Internal Tooth Form Layout

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